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aßstract

suitable for high school physical and environmental science and mathematics classrooms. The activities are independent, each having its own objectives, introduction, and background information. A medial section of each activity is written for the instructor and contains limits, sample data, and suggestions for following activities. Nost of the activities are analytical or empirical and require students to have completed a second year of high school algebra. (Author/RE)

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ENERGY AND ENERGY CONSERVATION ACTIVITIES FOR HIGH SCHOOL STUDENTS

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Prepared by the Project Staff and Participants of the WORKSHOP ON ENERGY CONSERVATION, University of Colorado, School of Engineering, and supported by the National Science Foundation (Project #SPI 78-04527, 1978)

April 1979

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ERIC

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Abstract

This manual contains fifteen energy and energy conservation activities suitable for integration into high school physical and environmental science and math classrooms. The activities are independent, each being self-contained with its own objectives, introduction and background information. A special section of each activity (Teacher's Guide) is written for the instructor and contains helpful hints, sample data and suggestions for follow-up activities.

Most of the activities are analytical or empirical and require a second year high school algebra capability. Others may require some background in trigonometry and computer programming. Effort was made to insure a quantitative approach and, where possible, to relate activity results to the social, economic and environmental ramifications of energy conservation. Among the topics included are thermocouples, heat content of fuels, heat storage mechanisms, wind machines, solar geometry, illumination, exponential growth, insulating characteristics of various materials, electrical consumption of appliances, solar greenhouses and passive solar design. Both theoretical and practical engineering concepts are illustrated and demonstrated by the activities.

Student pages can be detached from the manual and reproduced in class sets. Additionally, the materials needed to implement these activities are inexpensive and can be found at home and in high school stockrooms or they may be purchased locally at hardware stores.

Preface

physical science and mathematics instructors convened at the Engineering Center, University of Colorado, to participate in a National Science Foundation supported "Workshop on Energy Conservation". One of the project goals was to produce a self-contained set of activity modules for infusion and integration into high school physical science and math classes.

The rationale for this project is straightforward: Although many fine energy educational materials have been produced by public and private organizations in the years following the OPEC embargo, much of what is available is suitable for the non-technical student. It was felt that little exists currently to challenge and to encourage the above average student to pursue technical careers related to new energy technologies and oriented to energy engineering. This volume is a contribution to that effort.

Chuan C. Feng Project Director

	TABLE OF CONTENTS	PACE I
	owledgements	PAGE 1
	ractract	ii
	· · · · · · · · · · · · · · · · · · ·	444
Pref	ace	1.
ACT 1	VITIES:	<u>_</u>
#1	Impacts of Exponential Growth	1
#2	Calibration of a Thermocouple	21
#3	A Time and Cost Comparison of Heat Sources	28
#4	Save That Ice!	36
#5	The Insulating Qualities of Various Fibers	44
#6	Home Heat Loss Study	50/.
#7	How Much Heat Do We Lose Through a Typical Doorway?	, 72
#8	Calculating Electrical Consumption In Your Home	81
#9	The Efficacy of Light Sources	
#10	Using Wind Power to Produce Mechanical Work	95
#11	Analytical Determination of the Solar Angle of Incidence	
	on a Flat Plate Collector	107
#12	Let the Sun Do It!	
#13	Constructing a Model Greenhouse	
	Solar Heat Storage and Heat of Phase Change	
#15	Solar Energy Stòrage in Gravel	152
APPI	ENDIX A:	
1	Selected Constants	158
2.	Conversion of Units	
3.	Conversion of Celsius to Fahrenheit	
	Energy Ratings of Fuels	
5.	Specific Heat and Heat Storage	464
6.	Heat of Fusion of Materials	165
	~	
APP	ENDIX B:	٨
1.	Useful Solar Flux Quantities	
2.	Normal Monthly Average Temperature of Selected Cities	168
3.	Normal Total Heating Degree-Days	169

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Table of Contents (continued)

			. '	•		* * * * * * * * * * * * * * * * * * * *	PAGE NO
<u>ÁPP</u>	ENDIX B:				ı	•	
4.	Mean Daily	Solar Radi	ation				173
5.	Solar Posit	ion and In	solation_V	alues	• • • • • • •	• • • • • • •	175
APP	ENDIX C:	•	•	•	•	•	•
1.	Insulation Heat Eoss f	,			· · · · · · · · · · · · · · · · · · ·	••••••	188 192
APP	ENDIX D:	•	•				•
	Estimated A	Annual Ener	gy Consump	otion of A	pp1iance مهر	es	216
APP	ENDIX E	•	• .	•	,	•	 :
1.	Glossary of	f Energy Te	rms		• • • • • • •	· · · · · · · · ·	219

1. IMPACTS OF EXPONENTIAL GROWTH

PURPOSE: The purpose of this activity is to focus your attention on the relationship between rates of population growth and consumption of finite resources. Our industrialized systems which support our contemporary lifestyles function on an energy base derived principally from the fossil fuels (coal, oil, natural gas). While the unused quantities of these material resources may seem very great to us, the fact remains that they are being consumed very rapidly. Some of these resources will be depleted in our lifetime, causing profound changes in our lifestyles.

All of us need to understand and practice ways we can use our energy and other finite resources more conservatively and intelligently. Some of you may wish to consider pursuing energy management careers in order that you may become closely involved in helping to solve our energy problems. It is, perhaps, one of the greatest challenges ever to be faced by our world societies.

INTRODUCTION:

1. Definitions:

- a. <u>Finite resources</u> resources such as coal, oil, gas and minerals, which are limited; they are not being replenished or renewed.
- b. Exponential growth the growth of a quantity (such as population) which increases by a fixed percentage of itself in equal time intervals.
- the energy (1) to generate our electricity, (2) to run our industrial systems, (3) to heat our homes, (4) to run our

automobilés.

- d. Doubling time the time required for a quantity growing at a constant rate to double in size.
- e. Extrapolation a type of prediction in which what has happened in the past is extended into the future.
- 2. Applications: The concept of applying exponential consumption rates can be applied to situations other than energy. For example, if we can determine the population growth rate for Denver, we can extrapolate the future requirements for water, food and clothing.
- 3. <u>History</u>: Perhaps modern man has more information available about his resources and their consumption than at any time in the past. What we do with this information may have profound implications for the future.

OVERVIEW OF ACTIVITIES:

- 1. First Day: A review of mathematics and problem solving.
- 2. <u>Second Day</u>: A simulation exercise to compare the space occupied by two populations having different exponential growth rates.
- 3. Third Day: A simulation exercise to compare consumption rates.
- 4. Fourth Day: Fossil fuels and problem solving.
- 5. Fifth Day: General review and test.

FIRST DAY: A Review of Mathematics and Problem Solving

<u>PURPOSE</u>: As a result of today's activity, you will understand and be able to work problems involving:

(1) exponents, (2) exponential growth rates, and (3) doubling time. ($T_2 \approx 70/P$).

MATERIALS: Paper and pencil

PROCEDURE: Problem solving

1. Exponent: the power to which a number is raised.



Examples: 5^2 means $5 \times 5 = 25$

 5^3 means 5 x 5 x 5 = 125

 5^4 means 5 x 5 x 5 x 5 = 625

Problems: 8² = _____

3 =

2. <u>Scientific Notation</u>: When using very large numbers, it is much easier to express the number in terms of powers of ten.

Examples:

 $10 = 1 \times 10^{1}$

 $100 = 1 \times 10^2$

 $1000 = 1 \times 10^3$

 $1000000 = 1 \times 10^{6}$

Problems: 1. Express in powers of ten:

a. $27 = 2.7 \times 10^{-4}$?

b. 876 = 💌

c.1390 =

d.27,100,000 =

2. In 1970 the United States produced 3.29 billion barrels of oil. Express this quantity in powers of ten.

Exponential Growth Rate: When a quantity grows by a fixed percent per year, the growth is exponential. This is represented by Equation (1): $N=N_0e^{kt}$ (1) where N_0 is the size of the growing quantity at time, t=0; N is the final size of the quantity; k is the fractional change in N per time unit and e is the base of natural logarithms. Exponential growth is characterized by doubling. When the

growth rate is constant, Equation (2) can be used to determine the time for the quantity to double. That equation is:

where,

Tale, the time required to double and.

P = the percent growth per year.

Problems:

Assume that in 1958 the population of Denver, Colorado was 500,000 (5 x 10⁵). If the growth rate for Denver has been 3.5% per year, how many years are required for Denver's population to double?

 $T_2 \cong 70/P \cong 70/3.5 \cong 20$ years. Therefore, Denver's population in 1978 =

2. If each person consumes a quart of milk per day in Denver, how many gallons of milk would be consumed per day:

in 1958?

in 1998?

3. If each dairy cow gives four gallons of milk/day, how many

cows would be required to provide milk to Denver's population?

in 1958 ?

in 1978 ?

in 1998 ?

5

4. Assume that there were 250,000 (2.5 x 10⁵) automobiles in Denver in 1958 and that the number of autos will increase in direct proportion to population increase.

How many autos would we expect:

In 1978?

5. Assume that each automobile uses three gallons of gasoline per day. Compute the number of gallons of gasoline
consumed per day.

In 1958?
In 1978.?

6. Assume that each person in Denver uses 100 gallons of water per day. Compute the number of gallons of water used per day.

In 1958?

In 1998?

JONCLUSIONS:

- 1. What effect does an exponential growth rate have on a population?
- 2. What effect does an exponential growth rate have on a population's consumption of resources?

SECOND DAY: A Simulation Exercise to Compare the Space Occupied by Two
Populations Having Different Growth Rates

PURPOSE: As a result of this exercise, the student will: (1) observe, calculate and compare the rates at which space is occupied by two populations growing at different rates; (2) gain an appreciation for the impact which population growth rates have on space availability.

MATERIALS: 1-150 ml bottle/student

300 ml beans/pair of students

PROCEDURE: Using beans as units of alpopulation, students place them in the two bottles at differents ates. The first population doubles every minute; the second population doubles every five minutes. The simulation will continue until one bottle becomes full.

DATA: In the space provided, record the number of population units placed in each bottle for every minute interval.

<u>Time</u> (minutes)	Number of Population Units $(T_2 = 1 \text{ min})$	Number of Population Units (T2 = 5 min)
1		
2	•	
_		•
. 3		
4		
5		
6		· ,
-		
8		

•	Time minute	Number of Populatio S-) Units (T-) = 1 mi	Number of Population Units $(T_2 = 5.min)$
•	9		
	10		
	11		
`	12		
•	13		
	14		
•	15	<u> </u>	
:a1:	. •		
SULTS	: 1.	Did either population fil Which one?	l its bottle?
	· ,	How many minutes were req	ion were placed in the first bottle
	2.	How many units of populat	1 .
	3.		ntinued for the full fifteen minutes would have occupied the first bot
	•	TIOM MATTER DODGIACION MILLS	
		now many population units	
,		The second?	
<u>NCLU</u>	SIONS:	The second?	**
NCLU:		The second?	ed by population growth rates?
		The second?	*

Я

		owing at the ra	•			- a				_	
we ex	(pec	t the world pop	ulation	to c	louble	to 8	51111 6	on peop	ile?		
years	5.	(T ₂ ≅ 70/P)	1	•		•	7			• •	•
What	aff	ect will this	nave on	the_i	number	of:				•	:
,	a)	hosp4taJs?		•				·			
	b)	power generat	ing plai	nts?_		;	<u> </u>			•	·
	c)	farms?	,		. :	•	ì		4'		
	d)	reservoirs of	water?	, 	·	· · · · · · · · · · · · · · · · · · ·	, •			<u>.\.</u>	-
Now	name	5 other needs	which	will	be aff	ectéd	by th	is dou	bli n g.		- 1
			•	:	••			· · · ·			
			• •	-	•	_				2	

THIRD DAY: A Simulation Exercise Comparing Consumption at Zero and Exponential Rates

PURPOSE: As a result of today's activity, you will be able: (1) to compare mathematically the consumption of a resource by two populations—one at zero growth rate and one at an exponential growth rate and; (2) to develop a greater appreciation for the significance of resource consumption by populations, growing at exponential rates.

MATERIALS: 12 liters of popcorn (popped)

Two 6-liter bowls

PROCEDURE: Two large bowls of popcorn will each represent a finite resource.

(Note: Each bowl could represent the world reserves of coal, crude oil, natural gas, food, water, etc.) One will be consumed by a non-growing population (i.e., ZPG); the second will be consumed by a population which doubles every minute. (Note: The minute interval could represent any interval of time, a day, month, year, century.).

9

One member of the class will represent a zero-growth population.

(Note: Each person can represent a thousand, a million, or any number of people.) This person will eat one piece of popcorn every minute from the first bowl. The remaining members of the class will compose the growing population. They, too, begin consuming with only one person eating one piece the first minute. They continue doubling their consumption rate every minute until all remaining students are eating.

In order to continue the exponential rate of consumption, the larger population will now simulate greater numbers by simply doubling the quantity it consumes at each minute interval. The simulation will continue until one bowl is empty.

<u>DATA</u>: In the space provided, record the quantity of the resource consumed by each population group for every minute interval.

1	•
2 /	,
3	
4	
5	
6	
7	,
8	•
9	
10	
.11	
12	

	Quantity Consumed Zero Growth Popula		Quantity (Growing Po	Consumed By pulation	
13		· ·	•		4
4.		•	· .		•
15			•	- /	3
tal:			• 7		
RESULTS:					
/ 1. D	id either population co	onsume ali	of its re	source?	
· · · · · · · · · · · · · · · · · · ·		• • • • • • • • • • • • • • • • • • • •	/		
) W	nich one?				-
2. H	ow many intervals of t	ime (minut	tes) were n	equired for	the
•	esource to be consumed		<i>,</i>		•
3. D	uring this period of t	ime, how r	nany pieces	of the res	ource
	ere consumed by the ze	•	•		
•	,			•	
• В	y the exponentially gr	owing popu	uvation?	1 4	
4. I	f the simulation could	have con	tinued for	twenty-five	minutes
	ow many pieces would h	_			_
	opulation?	بنو		•	:
- B	y the exponentially gr	owing pop	ulation?	· · · · · · · · · · · · · · · · · · ·	
			·•		
CONCLUSION:	How does population	growth ra	te affect t	the consumpt	ion of
	a resource?		•		
r •	14"				
•		• :		•	
	*	10	·		

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FOURTH DAY: How Long Will Our Fossil Fuels Last?

<u>PURPOSE</u>: As a result of this exercise, you will: (1) be able to calculate the lifetime for certain fossil fuels; (2) gain a better understanding of when we may expect these resources to be depleted; (3) understand the importance of conservation; and, (4) give some thought to pursuing careers in science.

MATERIALS: Paper, pencil and calculator

PROCEDURE: Problem solving

DATA:

- Problems: 1. It is estimated that the ultimately recoverable resources of oil from just the continental United States will be 190 BB (billion barrels). By 1972, 96.6 BB had been produced. How many BB of oil remain to be receovered?

 What percent of the total oil was produced by 1972?
 - 2. In 1970 the annual production of oil from reserves in the continental United States was 3.29 BB. Assume that the consumption rate remains constant. (Is that really likely?) How many years of oil production remains? (This number is the static reserve index for domestic petroleum.)

At what year may we expect U.S. oil production to be exhausted?

3. The Alaskan oil reserves are estimated to be approximately 10 BB. Assume that the 1970 production rate of 3.29 BB could be maintained. How many years of additional production will this give us?

- 4. It is estimated that there may be 103.4 BB of U.S. oil shale.

 Assuming the 1970 production rate of 3,29 BB, how many additional years might, this add to our oil production?
- first has been calculated that with a 3% growth rate all of these reserves will have been consumed in 35.3 years. At a 3% growth rate factor, calculate how many years would be required for production (and consumption) to double.

6.	Why is	the	price	of	gasoline	increasing	steadily?	•	•
		i				•	٠.		•
	·			<u> </u>					

Do you expect the price to increase, decrease, or remain the same in the future?

			. •
Why?		 <u> </u>	

- 7. The ultimate world crude oil production is estimated to, be 1952.

 BB. By 1972 we had consumed 261 BB, leaving a world reserve of 1691 BB. The world consumption rate in 1970 was 16.7 BB. Assuming a zero growth rate, how many years will be required to consume the world's remaining reserves of oil?
- 8. Can we realistically expect the world consumption of oil to remain at a zero growth rate? Why or why not?

The "developing third world nations" are demanding more oil to improve their productivity and standard of living. If the world
consumption growth rate is only 5% per year, in how many years
could we expect the world reserve of 1691 BB to be depleted?



Hint: Use the following equation:

$$T = \frac{1}{k} \ln \left[\frac{Rk}{r_0} + 1 \right]$$

where:

k = growth rate, 1970

'R = remaining reserves

 $r_0 = production rate, 7970$

CONCLUSIONS:

1.	How do	production	growth rate	es affect	the	lifetimes	of	finite	re-
٠,	sources	?		,			•	•	

2:	Some people say	that	"growth is	goodbigger is better.	What do
	j.		•	•	•
•	you think about	this	statement?		·

3.	Through	"planneg	obsolesc	ence"	many	manuf	acturers	maintai	n high	pro-
•	duction	rates.	Should we	out1	aw "p	lanned	obsoles	cence?"	Why o	r why
	not?				_ _			·	.*	

•	Except for the continuous input of sunlight, the human race must
	finish the trip with the supplies that were aboard when the "space-
	ship earth" was launched. What impact does this statement have on
	the concept of recycling?

5. Many raw chemicals (used to make medicines, fertilizers, and plastics) are made from petroleum products. Should we save our petroleum for these important chemicals or use the oil for fuels?

Why or why not

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•	increasing rates of consumption of resources in the hope that s and technology will rescue us from the consequences of doing so	? 👌
	and technology with too as a	, , .
•		
•		
	What is of critical importance in this statement: "At current	levels
•	What is of critical importance in this statement: "At current of output and recovery, coal reserves can be expected to last in	•
•	of output and recovery, coal reserves can be expected to last	•
•		•
•	of output and recovery, coal reserves can be expected to last methan 500 years."	nore
•	of output and recovery, coal reserves can be expected to last methan 500 years."	nore



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TEACHER'S GUIDE

OBJECTIVE: Students will understand the impact of the exponential growth rate of population on the consumption of finite resources such as the fossil fuels, water and food.

BACKGROUND:

1. On Finite Resources: Professor Albert A. Bartlett, Department of Physics and Astrophysics, University of Colorado has stated, "The question of how long our resources will last is perhaps the most important question that can be asked in a modern industrial society."

Modern industrial and technological developments coupled with the voracious public appetite for the use of consumable resources enables us "to see the light at the end of the resource tunnel." The light, in this case, means the end of those resources, for all practical purposes.

The fossil fuels (coal, oil, gas) serve as the primary energy base for most of our industrial productivity. By practicing conservation techniques, we can prolong the use of but not prevent the exhaustion of these resources.

- 2. Exponential Growth Rates of Human Populations: In very general terms, we may assume the premise that a direct quantitative correlation exists between the number of people and the consumption of finite resources. Since population tends to grow exponentially, we can assume, therefore, that finite resources also will be depleted at an exponential rate.
- 3. Exponential Math: The classic equation for exponential growth is: $N = N_0 e^{kt}$ (1)

where N =the size of the quantity at time t,

 N_0 = the initial size of the quantity at time t = 0. e = the base of natural logarithyms (2.71828), k = the fractional change in the quantity per given time interval (the growth rate) and

t = time.

If the quantity doubles in size (N=2N₀) during an amount of time. t₂, then

and t_2 (the doubling time) = $\frac{.69}{k} \approx \frac{.70}{100k} \approx \frac{.70}{P}$

where P is k expressed as a percentage.

If Equation (1) is applied to resource production, integrated and solved for t, an expression termed the exponential reserve index is obtained:

 $t = \frac{1}{k} \ln \left[\frac{Rk}{r_0} + 1 \right] \qquad (2)$ where t = the resource expiration time; i.e., the time it will take from t_0 to consume all of the resource,

R = an estimate of remaining reserves of the resource.

k = the fractional rate of growth,

 r_0 = the production rate at t=0 and

In = the natural logarithym.

or comprehensive treatment of the subject of energy or resource consumption. Rather, it is intended to focus the student's attention on the relationship between rates of population growth and resource consumption. This relationship is having and will continue to have profound implications on all aspects of our lives.

TIME REQUIRED: This activity can be accomplished in five teaching periods of 45 minutes each:

MATERIALS:

Regular graph paper
300 ml of rice per pair of students

- 3. One 250 ml beaker or bottle per student
- ₹ 4. 2 1bse of unpopped popcorn (for 175 students)
 - 5. Two 6 liter bowls
 - 6.) Notebook paper

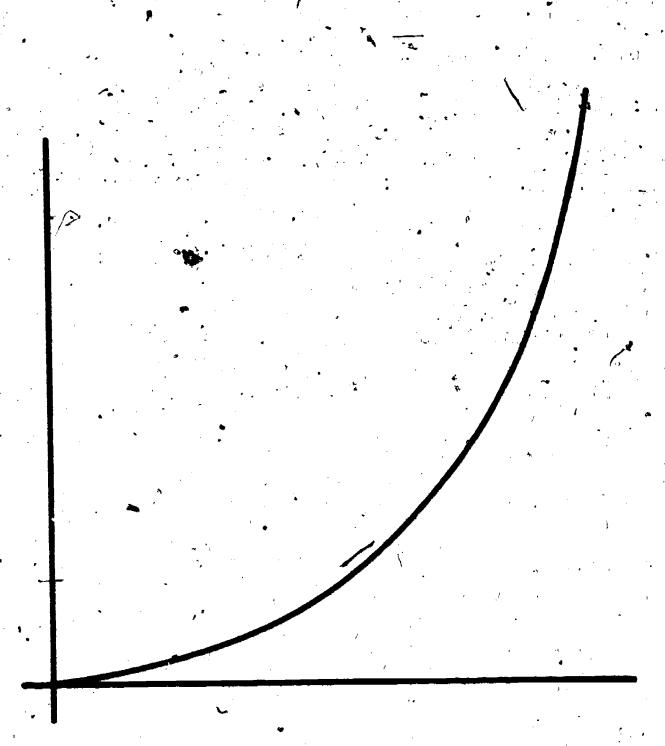
Suggestion for popcorniconservation: Save the remaining zero growth materials for other classes.

SUGGESTED FOLLOW-UP ACTIVITIES:

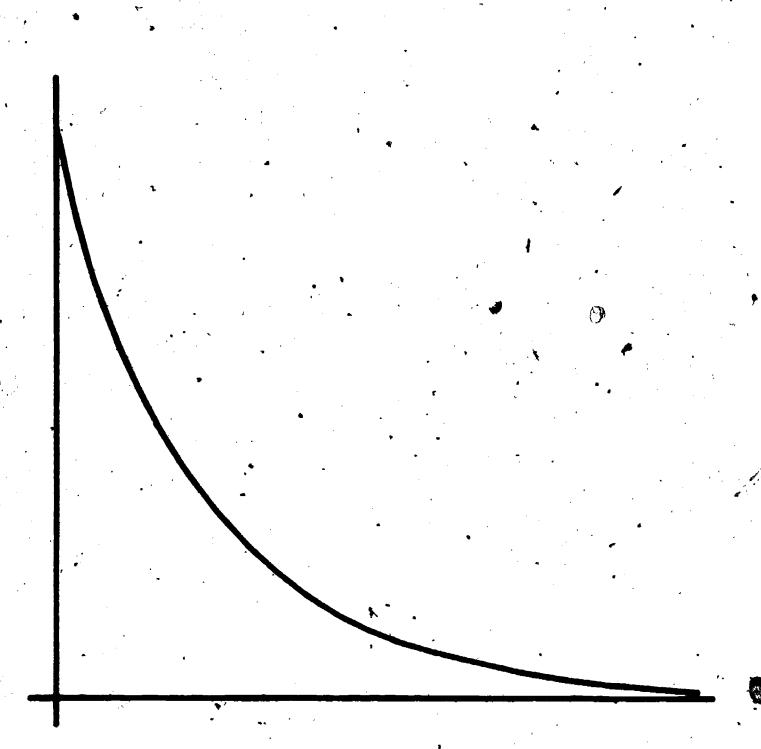
- 1. Construct exponential growth graphs for varying rates of growth
- 2. Repeat the above using semilog graph paper.
- 3. Show and discuss other types of exponential growth graphs using the graphs on the following pages.

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 A. Bartlett, Department of Physics and Astrophysics, University
 of Colorado, Boulder, Colorado.
- 4. "U.S. Energy Resources, A Review as of 1972; a National Fuels and Energy Policy Study, Serial 93-40 (92-75) Part I," by M. King Hubbert. U.S. Government Printing Office, Washington, D.C. 20402. \$2.35, 267 pages.



Exponential Curve, $y = ce^{kx}$ (k positive)



Exponential Curve, $y = ce^{kx}$ (k negative)

Exponential Curve, $y = 1 - ce^{kx}$ (k negative)

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#2. CALIBRATION OF A THERMOCOUPLE

PURPOSE: To calibrate a thermocouple for convenient temperature measurements.

INTRODUCTION: Society has always needed some form of measurement to be able to trade goods and build cities. It becomes increasingly important that we adopt standard units to insure that everyone is talking about the same quantity.

Scientists have accepted the freezing point and the boiling point of water (at one atmosphere of pressure) as the two reproducible points for temperature measurement. You will be calibrating a thermocouple against these same reference points by comparing the voltage output of the thermocouple against various water temperatures as measured with a thermometer.

Basically, a thermocouple is a device, composed of two dissimilar metal wires joined at two points, which converts heat into electrical energy by virtue of the potential difference or electromotive force (EMF) between the two metals.

This EMF is dependent upon temperature, allowing us to use thermocouples as temperature-sensing devices.

MATERIALS: Thermocouple, millivolt meter, thermometer, 250 ml beaker, heat source, ice, ring stand and graph paper.

PROCEDURE:

1. Prepare the ice-water bath by placing ice and cold water in a 250 ml beaker. Stir to bring the temperature down to 0° C. Assemble your apparatus as shown in Fig. 1.



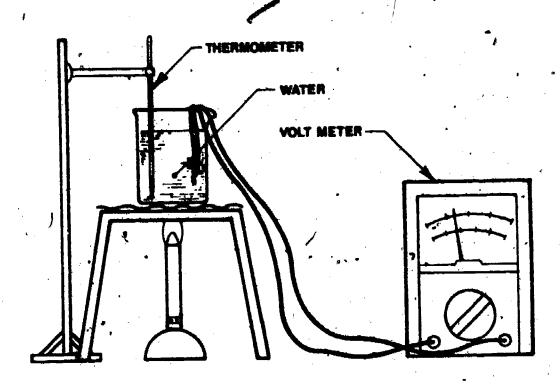


Fig. 1 Apparatus

- Wait until the bath temperature is at 0°C before taking any measurements. Record the voltage output at 0°C on your Data Sheet.
- 3. Begin heating the water bath and record both the water bath temperature and the voltage output every minute, until the water boils. Continue for two more minutes after the water boils. (Be sure to read both the voltage and water bath temperature simultaneously.)
- 4. Construct a graph of voltage as a function of temperature.
- 5. Determine the slope of the line and use a dotted line to extrapolate the curve to 300°C .

DATA SHEET

in	Time, minutes	Temperature (°C)	Voltage (millivolts)
0			
<u>, 1</u>			
2			•
3	0		
4		<i>2</i> 1	
5	· · · · · · · · · · · · · · · · · · ·		
6	,		
7			
8	· · · · · · · · · · · · · · · · · · ·		
9			
10			3
11	,		•
12		•	
13	,		
14			
15	•		1
16		1	
17			1
18_	```	1	-
19			

CALCULATIONS AND RESULTS:

- 1. Graph the voltage on the "y" axis and the temperature on the "x" axis.
- 3. Graphically, determine the voltage for the following temperatures:

 0°C = (what units?)

 · 20°C =



100-	C =
/150 ^o	C =
300°	C =
Be sure to	label and save your graph for further use with this thermocouple.
QUESTIONS:	
1. (-	Give three reasons for the need for accurate measuring tools.
2.	Why are at least two points needed to calibrate a scale?
3.	What voltage corresponds to 75°C?
	What is the corresponding temperature at 10 millivolts?
5.	Is there any difference in the slope between 10° C and 30° C and
	the slope between 80° C and 100° C?
	Using the slope, determine the voltage at 75° C?
7.	What information can you obtain from the slope?
8	What is one advantage of a thermocouple over a thermometer?
9.	What is one advantage of a thermometer over a thermocouple?
10.	Where could a thermocouple be used in your home?

TEACHER'S GUIDE

OBJECTIVE: The student will:

- 1. Understand how to calibrate a thermocouple using a water bath, a voltmeter and a thermometer.
- 2. Record and graph the data for future use.
- 3. Demonstrate an understanding of graph reading by determining the slope and extrapolating to 150° G (302° F).
- 4. Show an awareness of measurement by discussing the concept of scale calibration.

BACKGROUND: Thermocouples are a very convenient form of temperature measurement since their output can be monitored with a millivolt meter, thus keeping system disturbance at a minimum.

The operation of a thermocouple is based on the fact that, a small electric current will flow between two dissimilar metal wires if their junctions are maintained at different temperatures. One junction, called the reference, is normally kept at 0° C while the second junction serves as the sensor. However, acceptable results are available without maintaining the reference junction at 0° C, provided that the reference junction is always at the same temperature for each use.

This activity utilizes the meter itself as the reference junction while also completing the circuit. The best thermocouple for classroom use is the copper-constantan (type T) since its effective range is -60° C to 300° C. Its accuracy is $\frac{+}{2}$.8° C up to 100° C, increasing to $\frac{+}{2}$.25°C at 300° C. One short-coming, however, is the rapid oxidation of copper at high-temperatures over 400° C.

Thermocouple Temperature Measurement, P.A. Kinzie, 1973, John Wiley & Sons, Inc., p. 122.



THE REQUIRED:

- One period for actual measurements.
- 2. Two to three periods if the teacher wishes to include more about measurement and measuring skills.

MATERIALS (per group)

Thermocouple (see suggestions for sources)

Millivolt meter

Water bath (with ice)

Thermometer

Heat source

Ring stand and clamps

SUGGESTIONS: The simplest method of thermocouple construction is to buy type T thermocouple wire available in any city or from Instrument Service Co., at 4241 Jason, Denver, Colorado at about 8¢ per foot. The second choice is to buy equal guage copper and constantan wire or to try hardware stores for either wire or thermocouples.

The joint can be twisted, but is better if soldered or clamped.

This activity can be used as a replacement of the traditional blank thermometer calibration lab. It can be practically applied in some of the solar energactivities outlined in this manual. Additionally, it can serve as a source of discussion regarding measurement and graphing applications.

SUGGESTIONS FOR FOLLOW-UP: The thermocouple is almost a requirement for solar lab activities as thermometers need to be read by opening the system while thermocouples can be monitored without disturbing the system. Secondly, thermocouples can read higher temperatures than the standard classroom thermometers can safely attain.

The thermocouples produced in this activity can be used later in the following activities found in this manual:

Save That Ice, Activity #4

The Insulating Qualities of Various Fibers, Activity #5

The Efficacy of Light Sources, Activity #9

Constructing a Model Greenhouse, Activity #13

Solar Energy Storage in Gravel, Activity #15

REFERENCES:

- 1. Thermocouple Temperature Measurement, P.A. Kinzie, 1973, John Wiley & Sons, Inc.
- 2. The Theory and Properties of Thermocouple Elements, P.D. Pollock, 1971, American Society for Testing and Materials.
- 3. Source: Instrument Service Company, 4241 Jason, Denver, Colorado, (303) 458-7302.

#3. A TIME AND COST COMPARISON OF HEAT SOURCES

<u>PURPOSE</u>: In this experiment, you will: 1) use several different heat sources to raise the temperature of a known quantity of water, 2) calculate the quantity of heat required to heat the water, 3) calculate the relative costs for using each of these fuels, and 4) correlate the experimental data with home heating data.

INTRODUCTION: In our technological society, we obtain our energy from many different sources. In this country, homes are heated using a variety of. methods including oil, natural gas, wood, electricity, and solar energy.

This particular experiment will focus on the use of several fuels and a hot plate as heat sources. Some of the fuels that will be used are petroleum based; e.g., propane, butane and paraffin. Other fuels are distilled from wood products; e.g., alcohol and sterno. You will experimentally determine the cheapest method of providing heat. Upon completion of the experiment, other considerations such as efficiency, pollution, and fuel availability will be discussed. It is important for all Americans to understand the implications of using a particular fuel.

MATERIALS: (per team)

250 ml beaker

Various heat sources including candles, sterno, alcohol lamp, hot plate, propane torch

Water

Graduated cylinder, 100 ml

Clock or watch with second hand

Thermometer

PROCEDURE: For this experiment, you will be using several of the heat sources listed above. In order to compare the various heat sources, calculations of the amount of fuel must be done for each heat source. These calculations will vary depending on the heat source. Below, you will find the steps involved in calculating the amount of fuel used for each heat source.

- 1. Add exactly 200 grams of water to a 250 ml beaker.
- 2. Record the temperature of the water in the data table.
- 3. A. For sterno, alcohol lamp, propane burner: Determine the mass of the source before heating.
 - B. For hot plate: Record the watt rating given on the hot plate identification plate and turn the hot plate to the highest setting.
- 4. Place the beaker just above the hottest part of the source.
- 5. Record the starting time.
- 6. Continue heating until the temperature has been raised by exactly 50°C.
- 7. Record the time when the desired temperature has been reached.
- 8., Turn the heat source off.
- 9. For sterno, alcohol lamp, propane burner: Determine the mass of the source after heating.
- 10. Repeat 1-9 for each of the other heat sources.

DATA:

Fuel	Starting Mass, g	Final Mass, g	Starting Temp., C	Final Temp., OC	Starting Time	Ending Time
Candle		<u> </u>				
Sterno						
Alcohol Lamp	•		•	•		,
Propane Burner						
Hot Plate					,	
•	Wattage	Rating:				<u> </u>

FINAL RESULTS:

				·	
Fuel	B Units Used	C Cost/Unit	D Cost to Heat 200 ml H ₂ 0 to 50°C	E Cost/Calorie	F Time Needed
Candle	g :	¢/g	•		~
Sterno	9	- ¢/g		•	•
Alcohol Lamp	9	¢/g		,	
Propane Burner	g	, ¢/ g		,	
Hot Plate	KWH	¢/KWH			

CALCULATION:

A. Heat used = (specific heat) (mass) (change in temperature) = calories specific heat of water = 1 cal/gram O C mass of water used = 200 g change in temperature = 50^{O} C heat used = $(1 \frac{\text{cal}}{\text{g}^{O}\text{C}})$ (200 g) (50^{O} C) = ____ calories HEAT IN KILOCALORIES = $\frac{\text{calories}}{1000 \text{ cal}}$ | Kcal or Calories

		for I	not pla	ate: (of watts 000 w/kw)			<u>tes)</u> = _	KWH
,	Ç.	0bta:	in cos	t per u	nit fro	om your i	instruct	or.	•	
	D.	Cost	to he	at 200	ml of v	vater- 50 ⁰	c:	•		•
			cost p		•	umber of (B	units u	sed)		
	E.	Cost	per k	ilocalo	rie =			ted in D alories		(
•	F.	Time	neede	d = End	ling Ti	ne - Stai			;	
0			•			٠			•	
		•				W	• •	•		
,	• .			•	•		. *	•	•	
				15.		• .	•		-	
		•		•			•	· · · · · · · · · · · · · · · · · · ·		•
CONCLU	is to	N •							1	,
ONCL	J310	·				g		, ,		
		 .		v *	·	·				
UEST I	ONS	:					,			-
	1.	Whic	h heat	source	was ti	he most e	expensiv	e?	 	
٠	0	116.4	•. •.		, , ,		expensiv	e?	,,,	• ,
	2.	whici	n neat	source	was the	he fastes	st!			



· 9

3.	Other than	time and	cost,	what	other	factors	might	be	considered
	in choosing	a heat	source	? <u>;</u> _	,	. `			
			•	• .	,	-	e e		

4. Check with local heating contractors, electric companies and solar installation companies in order to complete the chart below:

HOME HEATING SOURCES -- UNITED STATES

Unit	No. of Calories per Unit	Cost per Unit	Cost per Calorie	Installation of Equipment Average Home	Pollution Level
gallon	36,000			.,	medium
1000/cubic ft	260,000				low
cord	5,300,000				high .
ton	6,200,000	•	•		high
Kwh	860		d.	•	variable
langley	1200 cal m ² /hr		-	•	none
	gallon 1000/cubic ft cord ton Kwh	Unit Calories per Unit gallon 36,000 1000/cubic ft 260,000 cord 5,300,000 ton 6,200,000 Kwh 860 langley 1200 cal	Unit Calories cost per Unit Sallon 36,000 Swh 860 Sallon Sallon Sallon Sallon Sallon Sallon Cord Sallon Sal	Unit Calories cost per Calorie gallon 36,000 1000/cubic ft 260,000 cord 5,300,000 ton 6,200,000 Kwh 860 langley 1200 cal	Unit Calories per Unit Cost per Calorie Average Home gallon 36,000 1000/cubic ft 260,000 cord 5,300,000 ton 6,200,000 kwh 860 langley 1200 cal

disadvantage	, , , , , , , , , , , , , , , , , , ,	, 4.1.5			,	
the table			<u> </u>			
			•	.•		•
		Make :	list o	f vour	lst. 2nd.	3r
Now, the cho	nce is your	s. Marc		*	,,	

TEACHER'S GUIDE

OBJECTIVES: Upon completion of this activity, the student will be able to:

- 1. Demonstrate proficiency in measurements of temperature and time in comparing heat sources.
- 2. Understand the source of each of the fuels encountered.
- 3. Make calculations of heat, amount of fuel consumed, cost of each heating source, and cost per calorie of heat delivered.
- 4. List several factors in choosing a fuel.
- 5. List five common methods of heating a building.
- 6. Research current prices for both installation and fuel, and make calculations necessary to compare the methods of heating.
- 7. Make an educated judgment as to the type of heating system most appropriate for his (her) needs.

BACKGROUND: Most students have heard about the different methods of heating a house or building. In this activity, they will be able to directly observe the efficiency of various laboratory heaf sources and relate the knowledge to actual heating systems in buildings. This particular activity could be used in a variety of scientific disciplines: the physics teacher could implement the activity when discussing energy and heat; the chemistry teacher could use it when specific heat is studied. It would be best utilized after students have had some background in fuel production and use.

TIME REQUIRED: 1 laboratory period for data collection

regular class periods for calculation and discussion

34

MATERIALS (per group): Heat sources can be rotated so that in a class of 30 students working in groups of 2, you would need 3 of each heat source.

- 3 candles
- 3 sterno cans
- 3 alcohol lamps
- 3 propane burner (stove)
- 3 hot plates
- 15 beakers
- 15 thermometers
 - 1 clock with second hand
- 15 graduated cylinders

SAMPLE DATA:

Fuel	•	Units Used	Cost Per Unit	Cost to Heat 200 m1 H ₂ 0 50 C	Cost/KCAL	Time Needed (min)
Sterno	,	10.2 g*	.31¢/g	3.17¢	.317¢	7.6
Alcohol Lamp	.	6.4 g	5.5¢/g	35.2¢	3.52¢	8.2
Propane		4.4 g	.37¢/g	1.65¢ §	_165¢	5.3
Butane	۸	5,3(g	`.78¢/g	4.12¢	412¢	3.5
Hot Plate (750 watt)	;	.083 Kwh	4.4¢/Kwh	.37¢	.037¢	6.7

SUGGESTIONS: ..

- 1. Caution students in the proper use of all heat sources. Safety goggles must be worn. Exercise care with hot beakers.
- · 2. Other heat sources, such as a Bunsen burner or butane camp stove could be substituted or added.
 - 3. In calculating the mass of fuel used for sterno, alcohol lamp, and propane stove, it is important to subtract the mass of the container when calculating the cost per gram.
 - 4. Have ready a quantity of water at room temperature.

SUGGESTED FOLLOW-UP ACTIVITIES:

- . 1. Discuss the use of fuels in producing electricity.
 - 2. Demonstrate the use of the sun's energy for heating water.
 - 3. Discuss heat loss in houses and methods for reducing fuel consumption.
 - 4. Demonstrate the use of fuels in transportation.

REFERENCES:

Energy - Environment Source Book, John Fowler, 1975; by National Science Teachers Association.

Energy - Environment Mini-Unit Guide, Stephen Smith, et al., 1975, by National Science Teachers Association.



#4. SAVE THAT ICE

PURPOSE: In this activity you will:

- 1. Evaluate the comparative value of the insulating ability of several different materials.
- 2. Calculate the heat loss (or gain) in Btu's.
- 3. Determine the R factor of your insulating material.
- 4. Propose a revision to the activity so that you can make the ice last 10% longer than your first attempt.

INTRODUCTION: Whether you are trying to keep heat out of a container or keep it in, the problem is still the same - how can you resist the transfer of heat from the hot region to the cooler one. The material that you use will accomplish this in varying degrees depending on its ability to resist that flow. This is known as the R-value of the insulating material, with the higher number indicating a greater insulating ability.

Your problem in this activity is to use any one material commonly available to you to wrap a known quantity of ice so that the least amount of ice will have melted in 24 hours. You will determine the percentage of ice lost to melting, and will compare the relative values of insulating properties of each of the substances used by the various teams in your class. In addition, you will calculate the amount of heat energy transferred into your ice "package".

MATERIALS (per group)

2 - styrofoam cups or 1/2 pint milk cartons Balance Insulating materials (your choice) Thermometer (-10 to 120° C)



PROCEDURE:

Day 1:

- Determine the mass of a styrofoam cup (or milk carton). Fill it
 with water and find the mass of water and container. Record in
 appropriate space on Data Sheet.
- 2. Determine and record the mass of the water (Data Sheet, #3).
- 3. Collect the insulating material that you will use.
- 4. Overnight, freeze the container of water.

Day 2:

- 5. In the next class period, remove ice from freezer and wrap it in your insulating material (not to exceed 2 feet in circumference along any of its three axes.
- 6. Label your team's package and place it in the same location as all the others as directed by the teacher.

Day 3:

- 7. During the next class period, unwrap the container of ice, being careful not to spill any melt water.
- 8. Determine the temperature of the melt water. (Data Sheet, #4)
- 9. Quickly determine the mass of the remaining ice and record. Subtract to find the mass of melt water. (Data Sheet, #5, 6 & 7)
- 10. Calculate the number of calories absorbed by the ice. This is the energy necessary to melt that mass of ice.
- 11. Determine the number of calories of heat that was used only in warming the water (#11).
- 12. Determine the total amount of heat energy absorbed by this system (#9 and #11).
- 13. Determine by calculation the number of Btu's of energy to melt the ice (#13). (1 Btu = 252 calories)



DATA

Day 1:			
T.	Mass of container	• .	g
2.	Mass of container and water		_ g
3.	Mass of water (#2 - #1)	· · · · · · · · · · · · · · · · · · ·	g
Day 2:	*	•	
4.	Temperature of melted water	·	_ oc
5.	Mass of water after 24 hours (pour off into a separate container)		_ g
6.	Mass of ice at end of 24 hours		_ g
7.	Mass of ice that melted (see #5)	·	_ g
8.	Percent of ice that remained after 24 hours (#6/#3 x 100)		_ %
*·9.	Number of calories of heat absorbed by the ice (each gram of ice that melts absorbs 80 calories of heat) $(80 \text{ cal}) \times \#7$	•	_ calories
10.	Temperature rise of melt water at end of 24 hours (Final temperature #4 - 0° C)	•	_ c ^o .
11.	Number of calories of heat absorbed by the melt water (#5 x #10)		calories
12.	Total amount of energy absorbed by the melting of the ice and warming of the melt water (#9 + #11)		_calories
13:	Number of Btu's absorbed (#12 x 1 Btu 252 calories)		_ Btu's
	·	46	

CONCLUSIONS AND QUESTIONS:

•
What experimental errors might have been encountered in this acti
How would you change this activity so that your remaining ice will be at least 10% larger?
Rank in order from the most expensive to the least expensive, the
relative cost of each of the materials.
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value?
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value? If you were to place any of these insulating materials in your
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value?
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value? If you were to place any of these insulating materials in your
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value? If you were to place any of these insulating materials in your house, what problems might you encounter?
Compared to the relative cost of each, which material would be the most economical to use considering its insulating value? If you were to place any of these insulating materials in your house, what problems might you encounter? Do any of the materials suggest difficulty in installation?



TEACHER'S GUIDE

OBJECTIVES: After completing this activity the students will be able to:

- 1. Compare the relative value of the various types of insulating materials.
- 2. Discuss insulating materials in terms of R-value.

BACKGROUND: Heat flow depends on area, time, thickness and temperature difference. The flow of heat occurs only when there is a temperature difference (the "driving force"). Regardless of how much insulation you use, there will be no heat lost or gained if the inside temperature is the same as the outside temperature. The greater the difference in temperature, the greater is the heat flow. The direction of heat flow or transfer is always from the warmer side to the cooler side.

The resistance to heat flow of a material, the R-factor, is determined by laboratory testing. The test results are expressed usually as "U-factors" rather than R-factors. A U-value is the transmission rate of a material expressed as the rate of flow (British Thermal Units, Btu) per square foot of area, per stated thickness, per hour of time subject to a temperature difference. This is often stated as Btu/hr/ft²/OF for specific thickness. Rather than writing Btu/hr/ft²/OF every time, the symbol U is used to express this value. The resistance to this flow rate is expressed as R, which is equal to 1/U.

One Btu 19 the amount of heat energy required to raise the temperature of one pound of water one degree F. All furnaces and air-conditioners are rated in terms of Btu's.



TIME REQUIRED:

- 4 class periods
 - 1 planning of approach and assigning of groups
 - 1 preparing insulating package
 - 1 measuring and recording data
 - 1 discussion and comparative evaluation

MATERIALS (per group):

- 2 water containers (1/2 pint milk cartons or large styrofoam cups)
- 1 freezer
- 1 thermometer
- 6 balances (several groups could share)

SUGGESTIONS:

- Avoid starting the activity such that the melting process will occur over a weekend or holiday. Twenty-four hours is about the limit to expect ice to last except under the mast ideal conditions.
- 2. Have each team or group summarize its data and type of insulating material used on the board so that the rest of the class can arrive at conclusions.
- 3. Suggested insulating materials:
 - a. spun fiberglass
 - b. styrofoam sheets
 - c. cllulose fiber (shredded newspapers)
 - d. urethane foam
 - e. rock wool
 - f. wood shavings
 - All of these may have varying thickness and density of packing.



- 4. Compare the R-value ratings of the various materials to their costs.

 A local lumber yard can give you current prices. Which material is the least expensive for a given R-value?
- 5. Experience has shown that using larger blocks of ice and having insulating material on hand makes for a more successful experience.

SUGGESTED FOLLOW-UP ACTIVITIES:

- 1. Have the entire class discuss the ideas elicited by Question #3. Have a group carry it out to quantify any improvements.
- 2. Discuss how retarding heat flow from our homes during the winter or into our homes during the summer is a real-world application of the principles demonstrated in this activity.

SAMPLE DATA:

	•		
1	Mass of container	14	_ g
2.	Mass of container and water	250	_ g
3.	Mass of water (#2 - #1)	236	_ g
4.	Temperature of melted water	8	_&c
5.	Mass of water after 24 hours (pour off into a separate container)	200	_ g
6.	Mass of ice at end of 24 hours	36	_ 9 ~
7.	Mass of ice that melted (see #5)	200	_ g /
8.	Percent of ice that remained after 24 hours (#5/#3 x 100)	15.25	%
.9.	Number of calories of heat absorbed by the ice (each gram of ice that melts absorbs 80 calories of heat)	16,000	_ calories
	(<u>80 cal</u>) x # 7	·- \	
•10. ·	Temperature rise of melt water at end of 24 hours $T = 0^{\circ}C \cdot T_{0} = 8^{\circ}C \cdot T_{0} - T_{1} = 8C^{\circ}$	8.0	co



11. Number of calories of heat absorbed by the water (#5 x #1Q)

1,600 calories

12. Total amount of energy absorbed by the melting of the ice and heating of the water (#9 + #11)

17,600 calories

 $(If T_2 - T_1 = 0^{\circ}C \text{ then } 16,000 + 0 = 16,000 \text{ calories})$

13. Number of Btu's absorbed

69.8 Btu's

REFERENCE:

Solar Experiments for High School and College Students, Norton, Hunter and Cheng. 1977 Rodale Press. p. 16.

NOTES:

Latent heat of fusion = 80 calories per gram.

, 80 calories of heat energy are required to melt 1 gram of ice to water at 0 $^{\circ}$ C.

1 Btu = 252 gram-calories

#5. THE INSULATING QUALITIES OF VARIOUS FIBERS

<u>PURPOSE</u>: The student will perform an experiment to collect data on the insulating capability of various cloth fibers used in clothing manufacture.

INTRODUCTION: One of the ways that man has protected himself from the forces of nature is through the use of proper clothing. He has learned to use the correct type of fiber or, in some cases, combinations of fibers to reduce the rapid loss of his body heat.

The purpose of the clothing we wear is to reduce heat loss to a reasonable level in order that we can remain comfortable under most weather conditions. Generally, the body's heat is lost by the three methods of heat transfer: convection, conduction, and/or radiation. Convection is heat transfer via moving fluids; conduction involves loss of heat through molecular motion in solids, and radiation is the transfer of heat via electromagnetic waves from a source through a separating medium.

We will determine in the following activity why certain types of clothing seem to keep us more comfortable in cold conditions than others.

MATERIALS: Several pieces of cloth (various weights of wool, cotton, nylon shelf, down). Two "heat" boxes/student group, thermocouple or thermometer for each "heat" box, source of wind (fan), wind guage (hand held). See Figure 1

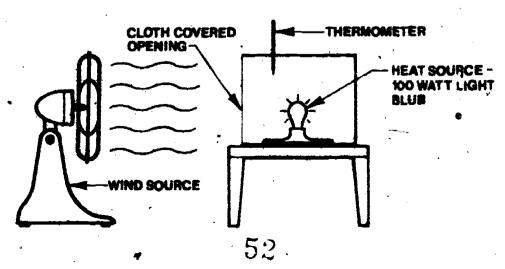


Figure 1. Apparatus



Two heat boxes are needed so that the one without the wind passing over it would be the "control". \

PROCEDURE:

- The heat boxes should be set up to begin generating heat at the start of the class period with the cloth removed to allow time for heat to accumulate and the temperature to stabilize in the boxes.
- 2. Place a piece of the selected material over the open sides of both heat boxes (masking tape should work well for this), completely covering the opening.
- 3. Direct a steady flow of air at the test cloth from a fan.
- 4. Begin 1 minute observations of temperature change, recording results in the data table.

DATA: In the following data table record the type of cloth being tested, the time and the heat box temperature. Continue the experiment, taking temperature readings each minute for approximately 20 minutes.

. •		Time, mi	nutac	Temperatu	re of	Heat F
Cloth Samp	le		·		•	
		•	DATA TAB	LE ·	n	•

Tim	e, minu	tes ,	Tempera	ature of	Heat Bo	ox, ^o c
0			•	•	2	·
1		,	#	•		
2		•				•
3			,			
4	_			•		€,
5.	•					
6			•	•	•	
7					``	

DATA TABLE (Continued)

Cloth Sample	`
--------------	---

Time, minutes 🛓	Temperature of Heat Box
8	
9	
10	
11 •	
12	
13	<u>.</u>
14	
15,	<i>/</i> :
16	
17	
18 _	
19	
20	

CALCULATIONS AND RESULTS:

- 1. From your data plot a graph of temperature as a function of time.
- 2. Determine the slope of the line on your graph; i.e., find the number of degrees change in temperature per minute of "exposure."
- 3. Compare your results with those of students who used different samples of cloth.

CONCLUSIONS AND QUESTIONS: Use class data to answer the following:

- 1. Which fiber has the highest insulating property in this expertment?
- 2. Can you suggest why some fibers are used in combination with other fibers?



- 3. Which of the natural fibers (wool or cotton) would you prefer for winter clothing?
- 4. Can you think of weather conditions that would alter the insula- r ting qualities of cloth fibers? Explain.

TEACHER'S GUIDE

OBJECTIVE: Students will learn how to determine what effect various clothing has on retarding the loss of body heat.

BACKGROUND: This activity is designed to illustrate the resistance of clothing material to heat transfer.

Building insulation (or R-factor) is discussed in another activity in this manual, but the purpose of this activity is to aquaint students with some of the principles of heat energy conservation even though it is dealing with heat conservation in the human body and not a building or home.

TIME REQUIRED: This activity should take no more than two class periods.

One period may be required to gather the data.

MATERIALS (per class or group): The teacher has the option of demonstrating this activity or assigning it to student groups. The materials required are: two "heat boxes/group; thermocouple or thermometer; card-board box; wind guage; wind source (fan). One or two fans will probably accommodate all of the students' test boxes. Suggested samples of cloth to use are wool, cotton, nylon, other natural and man-made fibers.

The use of "down" (a combination of fibers) can lead to a discussion of proper "layering" of clothes. Wool may also vary as to weight and tightness of weave.

SUGGESTIONS: This activity is appropriate for independent study in the following classes: energy, meterology, home economics, physics, earth science and biology. You may wish to check an ASHRAE manual for further



studies on values of clothing insulation.

FOLLOW-UP: A related activity involves the use of the following equation to determine wind chill:

$$H = (0.14 + 0.47 v^{1/2})$$
 (36.5 - T), where

H is the wind chill factor, v is the velocity of wind in m/sec and T is the ambient temperature in ${}^{O}C$.

You may also wish to define Clo units for your classes. These units are used by the clothing industry and engineers to determine comfort under varying environmental conditions—particularly wind speed and hymidity. One Clo is defined as the amount of insulation necessary to maintain comfort and mean skin temperature at 92° F in a room at 70° F with air movement not over 10 ft/min, humidity not over 50% (metabolism of 50 calories/ square meter/hour).

REFERENCES: ASHRAE Manual 1977, American Society of Heating, Refrigerating and Air Conditioning Engineers.

#6. HOME HEAT LOSS STUDY

<u>PURPOSE</u>: You will learn the concepts of heat flow and infiltration and will gain a practical knowledge about heat transfer in homes and their construction.

INTRODUCTION: Most exciting about this activity is its potential for studying heat flow in your own house. We will study an example of a 1000 ft², two-story house to observe what effects various construction materials and building techniques have on its thermal properties.

This activity is based on the derived expression for conductive heat loss:

$$q_k = kA \frac{\Delta T}{w} \qquad (1)$$

where q_k is the rate of heat transfer in $\frac{Btu}{hr}$ (Btuh),

k is the thermal conductivity of the surface material in

$$\frac{Btu - in}{hr \cdot ft^{20}F} \quad \text{or} \quad \frac{Btu - ft}{hr \cdot ft^{20}F},$$

A is the area of the surface in ft^2 ,

 Δ T is the temperature difference across the surface in 0 F and w is the surface thickness in feet or inches.

In the process of using Eq. (1), we will be introduced to thermal conductivity, R and U—values and the calculations of area for the house.

We will also study infiltration, using the density and specific heat of air, the number of air changes per hour, the volume of air in the house and the inside and outside design temperatures.

MATERIALS: .

A 4K computer, programmed in BASIC

Graph paper

58

Activity sheet



PROCEDURE:

- 1. Choose the house transmission area you wish to study: floors, ceilings, walls, wall on buffer space, windows and doors.
- 2. Brainstorm for various types of constructions for your transmission area. (See Appendix C)
- 3. List 5 or more different types of constructions for your transmission area.
- 4. Draw or describe each type of construction.
- 5. Input construction materials, and their R-values (or their k-values and their thicknesses). (See Appendix C.)
- 6. Get heat flow results in Btu/hr for each transmission area.
- 7. Get heat flow in Btu/hr for total house.
- 8. Find the gain or loss in Btu/hr for your change.

	Construc	tion #1	•
Layers of Materials	,		R-valůe
1		- ,	
2			
3			
4.9			
5		•	
6			
7	·		
8	,		
		Total R-valu	e '



Construction #2

s of Materials	· · · · · · · · · · · · · · · · · · ·		•	R-value
	·	•		,,
		1	•	٠,
	;			
		14		
•		•		
,		. 7		
				·
			•	<u></u>
		Total	R-value	
4		, oca i		
•	Construction	n #3	•	•
s of Materials	•	* *		•
, , , , , , , , , , , , , , , , , , ,	•	70' ,		
	. 3-4	٠	F	 ,
				\
	· · · · · · · · · · · · · · · · · · ·		•	
-		•	•	<u> </u>
		<u>.</u>	•	
4		•		
*	•	Total	R-value	
	Construcțio	n #4	8	
rs of Materials	· · · · · · · · · · · · · · · · · · ·			Dia.
• •	•			·
		_ ម		
		2		
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• • • • • • • • • • • • • • • • • • • •		- - - -		
		6	·	

ERIC Full Yeart Provided by ERIC

Construction #5

1.	ayers	of Materials			R-value
3.	1.		₽ -		
3.	2.	•			
4.	_	<u> </u>		_	
5. 6. 7. 8. Total R-value Construction #6 ayers of Materials 1. 2. 3. 4. 5. 6. 7. 8. Total R-value Construction #7 ayers of Materials 1. 2. 3. 4. 5. 6. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 8. 7. 7. 8. 7. 7. 8. 7. 7. 7. 8. 7. 7. 7. 8. 7. 7. 7. 7. 8. 7. 7. 7. 7. 7. 7. 7. 7. 7. 7. 7. 7. 7.	_		,	,	
7.	_				
Total R-value Construction #6 ayers of Materials 1.	6.	•			
Total R-value Construction #6 ayers of Materials 1.	7.				
Construction #6 ayers of Materials 1.	8.	•		•	
1		•	•	Total R-value,	*
1.		· (Construction	#6	
1.		of Materials			•
2. 3. 4. 5. 6. 7. 8. Total R-value Construction #7 ayers of Materials 1. 2. 3. 4. 5. 6. 7. 8.		o or materials		1	
3.	_			•	
4	2				
5	3	· · · · · · · · · · · · · · · · · · ·	, , , , , , , , , , , , , , , , , , , 		
6	4		<u> </u>		<u>.</u>
7	5				
Total R-value Construction #7 Ivers of Materials 1.	6			•	
Total R-value Construction #7 ayers of Materials 1	7		· · · · · · · · · · · · · · · · · · ·		
Construction #7 ayers of Materials 1	8			•	
1				Total R-value	
1			Construction	#7	,
1. . 2. . 3. . 4. . 5. . 6. . 7. . 8. .		•			
2.	ayer:	S OT Materials			<u>.</u>
3	1.	· ·		,	
3.	2.			•	
5. 6. 7. 8.					
6	4.			·	
7	5.	·		•	
8.	6.				
8	7.				
Total R-value					·
, , ittui n value				Total R-value	



		Construction	n #8	
Layers (of Materials '	•	•	R-value
1			•	
2.	•	-		
3.			•	
4	-1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1	<u> </u>	·	,
5.			•	
ъ	· · · · · · · · · · · · · · · · · · ·			
7	· 1		•	<u> </u>
-8				
		•	Total R-value	
	i	-	•	
•	• •	Table of Res	sults	1
Run #	Total R-value of Area	Area Heat Loss (Btu/hr)	Total Heat Loss (Btu/hr)	Difference in Gain or Loss
Control				
1	è	<u></u>		
2	,			
3				
4				
5		 		
6				
7		·		
8				
	• .		4,	
QUESTION	<u>'S</u> :		• • • • • • • • • • • • • • • • • • •	
			the lowest total	hoat Inse?
1.	Which transmis	sion area shows	THE TOWEST LOCAL	11eac 1033.
1.	41			nt of the total hea



If natural gas costs \$1.50 per million Btu's in Denver, how much

would one pay per day with the original system? \$ ____ The new

"construction": ____ %.

system? \$ _____ How much is saved? \$

Betwe	en heat	flow a	nd cons	strúcti	on mate	erials	?		£ .
	•				ى •				
What	factors	would	influe	nce you	r choi	ce of	constr	uction m	áterials
for a	new, hou	se?	•				•		•
	,			···				• ,	
	•		· •	** * *	· =				,
		<u> </u>				` .	•		
	could you	chang	e your	existi	ng hou	se to	reflec	t these	conclusi



TEAPHER'S GUIDE

OBJECTIVES: Students will:

- 1. Gain a practical knowledge of a house's thermal system.
- 2. Study the derived expressions for heat flow and infiltration.
- 3. Make a graph of heat loss as a function of R-value.
- 4, List ways to decrease heat loss.
- 5. Use the computer to gather data.

BACKGROUND: To analyze a home's heat loss, one must use several concepts such as conductive, convective and radiative heat flow. Only heat loss by conduction, the transfer of heat through solid materials, is considered in this activity.

If given a surface which separates two spaces having different ain temperatures, heat will flow (be conducted) through the surface from the warmer to the colder space. Conductive heat transfer is described by EQ (1):

$$q_k = \frac{kA}{w} \Delta T$$
 (1)

where q_k is the rate of heat transfer in $\frac{Btu}{hr}$ (Btuh),

k is the thermal conductivity of the surface material in

$$\frac{\text{Btu}-\text{in}}{\text{hr}\cdot\text{ft}^{20}\text{F}} \quad \text{or} \quad \frac{\text{Btu}-\text{ft}}{\text{hr}\cdot\text{ft}^{20}\text{F}}.$$

A is the area of the surface in ft²,

ΔT is the temperature difference across the surface in ^OF and w is the surface thickness in feet or inches.

EQ (1) indicates that q_k is directly proportional to the thermal conductivity, the surface area and the temperature difference; it is inversely proportional to the material's thickness.

In general, the rate at which most physical processes occur is directly proportional to a "driving" potential and inversely proportional to a resistance.

In conductive heat transfer, the "driving" potential is the temperature difference; therefore, by EQ (1), the thermal resistance, R_s , of the surface must be

$$\hat{R}_{S} = \frac{W}{kA} \qquad (2)$$

and EQ (1) can be rewritten as

$$q_k = \frac{\Delta T}{R_s}$$
 (3)

However, most tables listing thermal properties of construction materials and insulation do so in terms of one or more of the following:

- a) the thermal conductivity, k, in $\frac{Btu in}{hr ft^{20}F}$;
- b) the reciprocal of the thermal conductivity, $\frac{1}{k}$ in $\frac{hr}{Btu} = \frac{1}{10}$;
- t) the "R-value" for one ft^2 of material of a given thickness, w, $\frac{hr\ ft^{20}f}{Btu}$;
- d) the thermal conductance, $C = \frac{1}{R-value}$, in Btu , hr ft $\frac{2}{OF}$

Using the units for k and w, it can be shown that $\frac{w}{k}$ = R-value for one ft² and a given thickness of material:

$$\frac{\text{in}}{\frac{\text{Btu} - \text{in}}{\text{hr} \cdot \text{ft}^2 \cdot \text{o}_F}} = \frac{\text{hr ft}^{20}\text{F}}{\text{Btu}} = \text{R-value}$$

Substitution of R-value for $\frac{W}{K}$ into EQ (2) yields $R_{c} = \frac{R-value}{A}$

and substitution of R_S into EQ (3) gives

$$q_k$$
 (for a surface) = $\frac{1}{R-value}$ x ΔT x A (4)

Since $C = \frac{1}{R-value}$,

$$q_{\nu}$$
 (for a surface) = $\hat{C} \times \Delta T \times A$ (5)

(The reader should verify that the units do check.)

Depending on how thermal information is tabled, Equations (1), (4) and (5) will yield the same rate of heat flow (within rounding errors) through a surface composed of one material with a thickness, w, and area, A.

For a given heat barrier (wall, floor, ceiling, etc.) composed of two or more layers of building materials, it is necessary to define U, the overall coefficient of heat transmission (commonly called the U-value) for n-layers of materials:

$$U_{r} = \frac{1}{R_{T}} \tag{6}$$

where $R_T = R_1 + R_2 + \dots + R_n$ is the total thermal resistance for the multi-layered surface, and R_1 , R_2 R_n are the resistances of layer, layer,...layer,

The heat loss (or gain) through the multi-layered barrier is then expressed by

$$q_{\nu} = U \times \Delta \dot{T} \times A$$
 (7)

It can be seen from EQ (6) that the higher the U-value, the lower the thermal insulating value of the composite surface.

Appropriate units for
$$U$$
 are $\frac{Btu}{ft^2 hr^0 F}$.

EXAMPLE 1: Using Equations (1), (4) and (5), determine q_k for a wall (30 ft x 8 ft) of 4-inch face brick with inside and outside temperatures of $65^{\circ}F$ and $+10^{\circ}F$, respectively, (k for face brick is $\frac{9 \text{ Btu} - \text{in}}{\text{hr ft}^{20}F}$.)

Neglect air films and wind. Are your answers the same (within rounding error)?

SOLUTION:

a) By EQ (1):
$$q_k = \frac{k}{w} \times A \times \Delta T$$

$$q_k = \frac{9 \text{ Btu} - \text{in.}}{\text{hr ft}^2 \text{ o}_F} \times \frac{1}{4 \text{ in.}} \times 240 \text{ ft}^2 \times 55^0 \text{F} = \frac{29,700}{\text{hr}} \frac{\text{Btu}}{\text{hr}}$$

b) By EQ (2):
$$q_k = \frac{1}{R - value} \times A \times \Delta T$$
 $R - value = \frac{1}{k} \times w = \frac{1}{g \frac{Btu - in}{hr ft^2 o_F}} \times 4 in. = \frac{.11 hr ft^2 o_F}{Btu - in.} \times 4 in. = \frac{.444 \frac{hr ft^2 o_F}{Btu}}{hr ft^2 o_F^2}$
 $q_k = \frac{1}{.444} \frac{Btu}{hr ft^2 o_F^2} \times 240 \text{ ft}^2 \times 55^o_F = \frac{29.730 Btu}{hr}$

c) By EQ (5):
$$q_k = C \times A \times \Delta T$$

Since $C = \frac{1}{R-value} = \frac{1}{.444} = 2.252 \frac{Btu}{hr ft^2 o_F}$, then
$$q_k = 2.252 \frac{Btu}{hr ft^2 o_F} \times 240 \text{ ft}^2 \times 55^0 F = \frac{29,700}{hr} \frac{Btu}{hr}$$

EXAMPLE 2: Determine U and q_k for a composite "wall" (10' x 10') made of 3/4 inch plywood backed by 2 inches of blanket/batt insulation. Inside and outside temperatures are 65° F and -10° F, respectively.

SOLUTION: From Tables:

k for plywood is .802
$$\frac{Btu - in}{hr ft^2}$$
;

1 for blanket/batts is 3.5 $\frac{hr ft^2}{Btu - in}$.

R-value for plywood of this thickness is $R-value = \frac{1}{k} \times w = \frac{1}{.802} \frac{hr ft^{2} oF}{Btu - in} \times .75 in. = .935 \frac{hr ft^{2} oF}{Btu}$

R-value for blanket/batt insulation of given thickness is: R-value = $\frac{1}{k}$ x w = 3.5 $\frac{hr}{Btu} = \frac{1}{n}$ x 2 in. = $7 \frac{hr}{Btu} = \frac{1}{n}$ By EQ (6) $U = \frac{1}{n}$

$$U = \frac{126}{7.935} = \frac{.126}{hr} \frac{Btu}{hr}$$
By EQ (7)
$$q_{k} = U \times A \times \Delta T$$

$$q_{k} = .126 \frac{Btu}{hr} \times 100 \text{ ft}^{2} \times 75^{0}\text{F}$$

$$q_{k} = \frac{945.2}{hr} \frac{Btu}{hr}$$

Another expression used in the computer program relates to infiltration.

Infiltration is expressed by EQ (8):-

$$q_{\downarrow} = V \cdot a_{H} \cdot \rho \cdot C_{a} \cdot \Delta T \qquad (8)$$

where q_i = rate of heat loss due to infiltration

V = volume of air in the house, ft³

a_H = air changes in the house per hour, usually between .5 and 1.5;

(.7 used in the program)

 ρ = the density of air (1b/ft³)

 $C_a =$ the specific heat of air (Btu/1b^OF)

 Δ T = difference in temperature between inside and outside, O F Example:

For a house with 1000 sq. ft. and 8 ft. ceilings, determine the heat required to maintain an inside air temperature of $65^{\circ}F$ with an outside temperature of $15^{\circ}F$; ρ = .075 lbs/ft³

$$q_1 = V \cdot a_H \cdot C_a \cdot \rho \cdot \Delta T$$

$$= 8000 \text{ ft}^3 \cdot .7 \text{ changes/hr} \cdot .24 \text{ Btu/lb}^0 \text{F} \cdot .075 \text{ lbs/ft}^3 \cdot 50^0 \text{F}^0$$

= 5040 Btu/h or Btuh

TIME REQUIRED: One to two hours, depending on availability of computers

MATERIALS:

Activity sheet

Computer

"Graph paper

SUGGESTIONS: Students working in groups of two or three is suggested, allowing more construction combinations to be used. Encourage wall constructions of cement block, cement form, brick and wood siding; for ceilings: glass fiber, rock wool, cellulose fiber; for floors: wood with and without insulation underneath, carpet, tiles; for windows: with storm windows, double-paned, triple-paned; for doors: with and without storm doors.

It is suggested that you group your students to share their data with others so that they may see the relative importance of their results in respect to the total house.

In the computer program there may be two sources of difficulty with your BASIC system. On line 710 two LET statements are combined on one line with a colon. If you can't use the colons, use 710 LET R = R(9), 715 LET R(1) = R(9) as substitutes. In a few lines the LET statement in an equation has been omitted. For lines 202, 280, 315, 689, and 695 you may add them if necessary.

Also lines 750 to 830 may be a problem in some systems. The logic would require two additional statements for each IF THEN.

EXAMPLE:

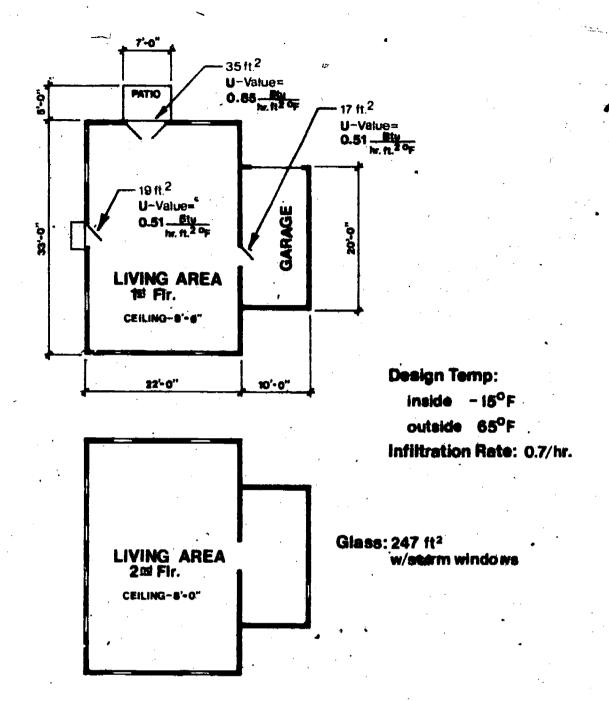
750 IF I = 1 THEN A = W1 + W2 - W3 = W9 - D1 - D2 - D3

You should substitute:

750 IF I = 1 THEN 831

831 LET A = W1 + W2 - W3 - W9 - D1 - D2 - D3

832 GO TO 840



FLOOR PLAN FOR STANDARD RUN HOUSE

Walls: brick veneer 4", wood sheathing 3/4"; stud wall with
fiberglass insulation 3-1/2" and wallboard 1/2". R = 11.72

2nd floor ceiling wallboard and fiberglass ball insulation (6")

1st floor floor hardwood flooring 1/2" on plywood subflooring 3/4"

(uninsulated)

SAMPLE DATA

TRANSMISSION AREA 1st Floor		R-values
Layers of mater ls 1. Hardwood flooring 1/2"		.48
2. Plywood subflooring 3/4"		.93
· · · · · · · · · · · · · · · · · · ·	TOTAL R-values	1.41
TRANSMISSION AREA <u>Ceiling</u>		
Layers of materials		
1. Wallboard 1"		.45
2. Fiberglass batt 5"	•	16.67
	TOTAL R-values	17.12
TRANSMISSION AREA <u>Walls</u>		
Layers of materials	•	,
1. Brick veneer 4"		5.99
2. Wood sheathing 3/4"	•	.93
3. Studwall with fiberglass ins	ulation 3 1/2" 📑	~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~ ~
4. Wallboard 1/2"		.23
	TOTAL R-values	18.98
TRANSMISSION AREA <u>Windows</u>	•	
Layers of materials		
1. Windows with storm windows		2.00
•	TOTAL R-values	2.00
TRANSMISSION AREA <u>Door</u>) ,	•
Layers of materials	,	
1. Single doors		1.96
	TOTAL R-values	1.96
TRANSMISSION AREA French Doors		
Layers of materials	•	
1. French doors	•	1.18
* · · · · · · · · · · · · · · · · · · ·	TOTAL R-values	1.18

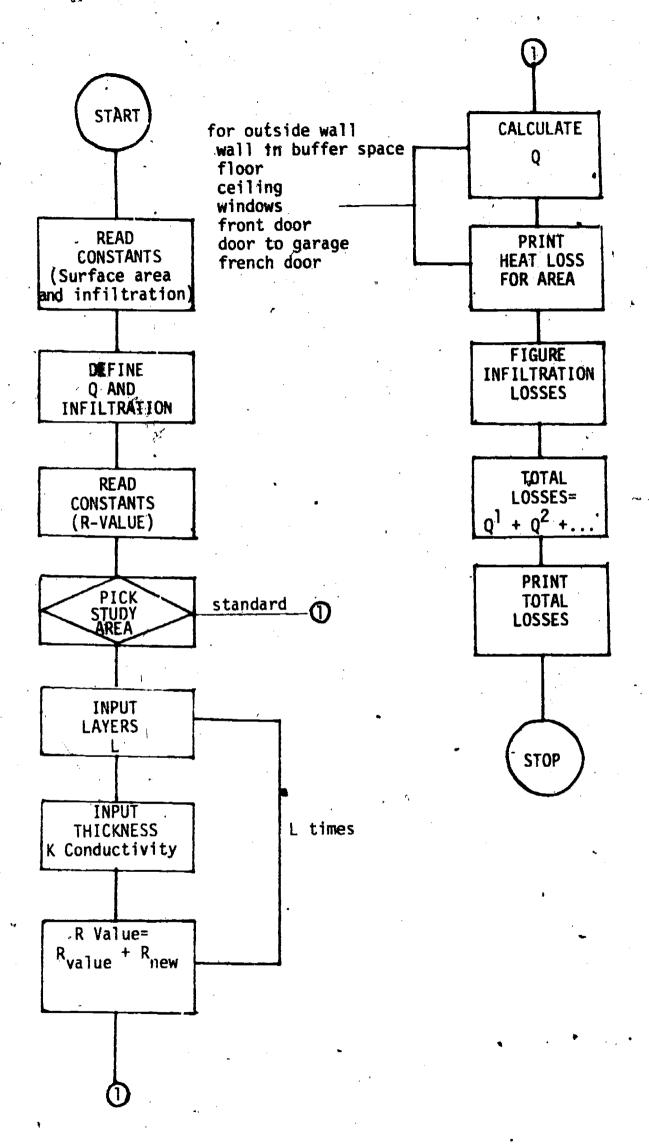


Fig. 2. Programming Flow Chart

PROGRAM

```
THIS PROGRAM IS DESIGNED TO STUDY HEAT FLOW THROUGH
20 REM
        DIFFERENT MATERIALS IN A HOUSE
        WRITTEN 8, 13, 78 BY DENNIS COLE REVISED 9.1.78
30 REM
        THE FIRST READ INPUTS THE CONSTANTS FOR THE
40 REN
        HOUSE'S SURFACE AREAS. THE SECOND READ ADDS
50 REM
        THE CONSTANTS FOR INFLITRATION, THE THIRD INPUTS
60 REM
70 REM R VALUES FOR THE SURFACE AREAS.
80 READW1, W2,W9,D1,D2,W3,D3,F1,C1
90 READV, C, T1, T2
92 PRINT "ENTER YOUR DESIGN TEMPERATURES FOR YOUR HOUSE"
93 PRINT"(INSIDE JEMPERATURE COMMA OUTSIDE TEMPERATURE)";
94 INPUTTI,T2
100 DATA935, 1040,247,19,17,153,35,926,926,13579,17,65,-15
130 FORI=1T08
135 READR(I)
140 NEXT.
150 DATA11.72,11.72,.74,17.12,2,1.96,1.96,1.18
160 PRINT"WHICH TRANSMISSION AREA DO YOU WANT TO STUDY":PRINT,
165 PRINT"1, WALLS", "2, WALL OF BUFFER SPACE", "3. FLOORS"
170 PRINT"4. CEILING", "5. WINDOWS", "6. DOORS", "7. CONTROL RUN"
180 INPUTX
190 IFX=7THENT360
200 PRINT"DO YOU KNOW THE TOTAL R VALUE OF YOUR MATERIALS!"
201 PRINT" (YES OR NO)":
202 R=0
210 INPUTB$
220 IFB$="YES"THEN232
221 PRINT"DO YOU KNOW THE R VALUE OF EACH OF YOUR MATERIALS";
222 INPUTC$
223 IFC$="NO"THEN260
224 PRINT"HOW MANY LAYERS ARE THERE INCLUDING AIR SPACES":
225 INPUTZ
226 FORJ=ITOZ
227 PRINT"WHAT IS THE R VALUE OF MATERIAL NO. ";J;
228 INPUTRR
229 LETR=R+RR
230 NEXTJ
231 GOTO360
232 PRINT"WHAT IS THE R VALUE";
240 INPUTR
250 GOT0360
260 PRINT"HOW MANY LAYERS ARE THERE INCLUDING AIR SPACES";
270 INPUTZ
280 R=0
290 FORJ=ITOZ
300 PRINT"HOW THICK IS LAYER"; J; "IN INCHES";
310 INPUTM
```

(,

PROGRAM (continued)

```
315 M=M/12
 320 PRINT"WHAT IS THE THERMAL CONDUCTIVITY CONSTANT K"
 321 PRINT" (IN BTU.-FT./HR.SQ.FT.DEGREES F.)";
 330 INPUTK
 340 LETR=R+(M/K)
 350 NEXTJ
 360 LETR(9)=R
 380 GOSUB700
 400 LETQ(9)=V*C*.075*.24* (T1-T2)
 405 DIMA$(20)
 410 FORI-IT09
 420 READA$(1)
 430 NEXTI
 440 DATA "OUTSIDE WALL", "WALL ON GARAGE AREA", "FLOOR", "CEILING"
 450 DATA"WINDOWS", "FRONT DOOR", "GARAGE DOOR", "FRENCH DOORS"
 460 DATA"INFLITRATION"
 610 PRINT: PRINTTAB(25); "HEAT LOSSES": PRINT: PRINT
 630 PRINT"TRANSMISSION AREA"; TAB(25); "R-VALUE"; TAB(40); "HEAT LOSS"
 635 PRINTTAB(41); "BTU/HR."
 640 FORI=ITO8
 650 PRINTA$(1); TAB(27);R(1);TAB(42);Q(1)
 660 NEXTI
 670 PRINTA$(9);TAB(42);Q(9)
 680 PRINT: PRINT" TOTAL LOSSES"; TAB(42);
 689 Q=0
 690 FOR 1-1709
 695 Q=Q+Q(1)
 696 NEXTI
 697 PRINTO
 699 END
 700 FORI=1T08
 701 IFI-7THEN703
 702 IFI<>8THEN705
 703 \text{ LETR}(1) = R(6)
 704 GOT0750
 705 IFX<>1THEN740 *
 710 R-R(9):R(I)=R(9)
 720 GOT0750
 740 R-R(I)
 750 IFI-1THENA-W1+W2-W3-W9-D1-D2-D3
 760 IFI-2THENA=W3/2
 770 IFI-3THENA-F1/2
 780 IFI-4THENA=C1/2
 790 IFI-5THENA=W9
800 IFI=6THENA=D1
805 IFI=7ANDX=6THENR=R(6)
~806 IFI=8ANDX=6THENR=R(6)
810 IFI=7THENA=D2/2
830 IFI=8THENA=D3
840 LETQ(I)=A*(T-T2)/R
850 NEXTI
860 RETURN
```

SAMPLE PRINTOUTS

CONTROL

RUN

SENTER YOUR DESIGN TEMPERATURES FOR YOUR HOUSE (OUTSIDE TEMPERATURE COMMA INSIDE TEMPERATURE)? 65,-2 WHICH TRANSMISSION AREA DO YOU WANT TO STUDY

1. WALLS

2. WALL OF BUFFER SPACE

3. FLOORS

4. CEILING

5. WINDOWS 6.

6. DOORS

7. CONTROL RUN

?7

HEAT LOSSES

TRANSMISSION AREA	R-VALUE		HEAT LOSS BTU/HR.
OUTSIDE WALL WALL ON GAME AREA FLOOR CEILING WINDOWS FRONT DOOR GARAGE DOOR FRENCH DOORS INFLITRATION	11.72 11.72 .74 17.12 2 1.96 1.96 1.18	•	8597.95 437.329 41920.3 1811.97 8274.5 649.49 290.561 1987.29 11463.4
TOTAL LOSSES	•		75432.8

0K

2. WALLS

OK RUN

ENTER YOUR DESIGN TEMPERATURES FOR YOUR HOUSE (OUTSIDE TEMPERATURE COMMA INSIDE TEMPERATUR)? 65,-10 WHICH TRANSMISSION AREA DO YOU WANT TO STUDY

T. WALLS

2. WALL OF BUFFER SPACE

3. FLOORS

4. CEILING

5. WINDOWS

6. DOORS

7. CONTROL RUN

?)
DO YOU KNOW THE R VALUE OF YOUR MATERIALS (YES OR NO)?
?NO

HOW MANY LAYERS ARE THERE INCLUDING AIR SPACES? 2
HOW THICK IS LAYER 1 IN INCHES? .75

WHAT IS THE THERMAL CONDUCTIVITY CONSTANT K (IN BUT.-FT./HR.SQ.PT.DEGREES F.)? .14

HOW THICK IS LAYER 2 IN INCHES? 2
WHAT IS THE THERMAL CONDUCTIVITY CONSTANT K

(IN BUT.-FT/HR.SQ.FT.DEGREES F.)? .025

SAMPLE PRINTOUTS (continued)

2. WALLS(continued)

HEAT LOSSES

TRANSMISSION AREA	R~VALUE	HEAT LOSS BTU/HR.
OUTSIDE WALL WALL ON GARAGE AREA FLOOR CEILING WINDOWS FRONT DOOR GARAGE DOOR FRENCH DOORS INFLITRATION	7.1131 11.72 .74 17.12 2 1.96 1.96 1.18	15858.1 489.548 46925.7 2028.33 9262.5 727.041 325.255 2224.58 12832.2
TOTAL LOSSES	•	90673.2

3. CEILING

OK

RUN

ENTER YOUR DESIGN TEMPERATURES FOR YOUR HOUSE (OUTSIDE TEMPERATURE COMMA INSIDE TEMPERATURE)? 65;-10 WHICH TRANSMISSION AREA DO YOU WANT TO STUDY

1. WALLS

2. WALL OF BUFFER SPACE

3. FLOORS

4. CEILING

5. WINDOWS

6. DOORS

7. CONTROL RUN

?4

DO YOU KNOW THE R VALUE OF YOUR MATERIALS (YES OR NO)* ?NO

HOW MANY LAYERS ARE THERE INCLUDING AIR SPACES? 2

HOW THICK IS LAYER 1 IN INCHES? .5

WHAT IS THE THERMAL CONDUCTIVITY CONSTANT K

(IN BTU.-FT./HR.SQ.FT.DEGREES F.)? .0925

HOW THICK IS LAYER 2 IN INCHES? 12

WHAT IS THE THERMAL CONDUCTIVITY CONSTANT K

(IN BTU.-FT./HR.SQ.FT.DEGREES F.)? .024

SAMPLE PRINTOUTS (continued)

3. LING (continued)

HEAT LOSSES

TRANSMISSION AREA	R-VALUE	HEAT LOSS S
OUTSIDE WALL WALL ON GARAGE AREA FLOOR CEILING WINDOWS FRONT DOOR GARAGE DOOR FRENCH DOORS INFLITRATION	11.72 11.72 .74 42.1171 2 1.96 1.96 1.18	9624.57 489.548 46925.7 824.487 9262.5 727.041 325.255 2224.58 12832.2
TOTAL LOSSES		83235.8

- OK

SUGGESTED FOLLOW-UP ACTIVITIES:

1. Have students do heat flow analyses for their houses. This may be done by "cranking it out" or by changing these components of the computer program. (All the changes would be made in Lines 80 through 150):

W1 = wall area of 1st floor

= perimeter · height of · ceiling

W2 = wall area of 2nd floor

= perimeter · height of ceiling

W3 = wall facing buffer space

= length · height

W9 = Area of windows

D1, D2, D3 = area of doors (you must be particular about door placement)

F1 = area of first floor

C1 = area of ceiling or roof surface area

For simplification, treat basements like attics. (Temp = $\frac{1}{2}\Delta T$.)

R1 through 8 are the R-values of the areas.

- The computer program was designed especially for use with all BASIC computers. A student could be assigned to change the program. Fig. 1 is the floor plan used to create the standard run.
- 3. Discuss the topic of insulation payback at current fuel cost. Using degree days, compute the heating bill with and without the new insulation. One might also consider a house-design activity.
- 4. Further studies of heat flow by convection and radiation, infiltration and insulation materials, might be pursued.
- 5. Other programming activities are thermostat operation and set-back efforts, passive and active solar design, monitoring home consumption,

73

computerized home environmental control, buildings and energy codes, efficiency, and heat content of fuels.

REFERENCES:

ASHRAE Fundamentals, Chapters 19, 21, 23, 24, 22, 25, 26

Solar Heating and Cooling, Chapters 1 and 2. Jan F. Kreider and Frank

'Kreith. McGraw-Hill, 1977.

Energy Conservation in the Home. U.S. Dept. of Energy. Oct., 1977.
"Simulation of Temperature Within a Building Space" by Wai-Bui Zee
for NSF Workshop, 1978.

Lectures 16 and 17 by Dr. John Dow for NSF Workshop

#7. HOW MUCH HEAT DO WE LOSE THROUGH A TYPICAL DOORWAY?

PURPOSE: In this activity, you will examine various options for doorway design in a model house. You will determine the minimum amount of heat necessary to keep the model house at a desired temperature.

INTRODUCTION: All of our lives, we have heard the words "Close that door. Do you want us to freeze to death?" or some variation of that statement. But what you may not know is just how much heat is lost through an open door. In addition, there are simple ways to reduce that heat loss considerably.

There are several factors which will affect the amount of heat lost through a doorway. Obviously, the outside temperature is important, and the speed and direction of the wind must be considered. The design of the doorway, however, is probably the most critical aspect of heat loss through it. If the door is set back from the rest of the house, this may reduce heat loss. Another idea to consider is the addition of a vestibule, which makes it necessary to enter the house by opening and closing an outside door followed by opening and closing a second door into the house from the vestibule. In this activity, we will use a model house to compare different doorway designs.

MATERIALS: Two styrofoam model houses - one with a flush door, the other with a vestibule.

Heat source for model house (40-watt light bulb)

Thermostat

Thermometer

Clock (with second hand)

PROCEDURE: In this experiment, you will be comparing the amount of time that the light bulb "heater" is actually on in order to calculate the amount of energy necessary to heat your model house.

1. Control "house":

- a. Set-up your house with the door closed.
- b.' Record the temperature.
 - c. Set the thermostat at the setting determined by your instructor.
 - d. Turn on the light. Record the starting time.
- e. For each minute, record whether the light was on or off for that minute. If the light was on for more than half of a minute, record "on", less than half a minute record "off".
 - f. Record the temperature at 1 minute intervals throughout the experiment.
- g. Continue the experiment for ten minutes after the house has reached the thermostat temperature. Continue to record temperatures and on-times of the bulb.

House with door opening every 2 minutes for 5 seconds:

- a. Set up your house with the door closed.
- b. Record the temperature.
- c. Keep the thermostat at the same setting.
- d. Turn on the light and take readings as before.
- e. When the temperature has reached your thermostat setting, open the door and leave it open for exactly 5 seconds. Continue to do this at 2 minute intervals.
- f. Record the time and temperature readings as before as well as the bulb on-time.



- 3. House with door opening every minute for 5 seconds:
 - a. Repeat the above steps, only this time open the door every minute for 5 seconds.
- 4. House with vestibule and door opening every 2 minutes for 5 seconds:
 - Repeat the above except that when opening the doors, follow these instructions:
 - . Open outside door for 5 seconds, then close.
 - Open inside door for 5 seconds, then close:
- 5. House with vestibule and door opening every minute for five seconds:
 - a. Repeat step 4; only this time open the vestibule and door every minute.

•		trol '	House openin 2 minu	g every	openin	w/Door g every nute		ule & 🗞 pening	House very	ule & pening
Time (min)	Temp (°C)	Light On/Off	Temp (°C)	Light On/Off	Time (°C,)	Light On/Off	(oc)	Light On/Off	Temp (°C)	Light On/Off
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CALCULATIONS:

- A. Make graphs of the data you have collected for each of the models. The graphs should be temperature as a function of time. Indicate on each graph whether the light was on or off by showing a solid line for "on" and a dotted line for "off".
- B. Calculate the amount of time that the light was on for each condition after the temperature had reached the thermostat setting.

 This can be done by simply checking your data and adding up the minutes that the light was on. Record your results below.
- C. Calculate the amount of energy used during the ten minute period of each condition by following the sample below.

NUMBER OF : WATT RATING NUMBER OF KILDWATT = OF LIGHT X 1 KILDWATT X MINUTES THE X 1 HOURS BULB 1000 WATTS LIGHT WAS ON 60 MIN

Record your results below

D. Calculate the percent increase in energy use due to the opening of the doors and/or vestibule. (Hint: See the sample calculation below.)

Amount of Energy Used Amount of Energy Used

(KWH) in Control House (KWH)

Amount of Energy Used
in Control House (KWH)

Summarize the results below.

Results

Amount of Time
Light Was On Number of KWH % Increase in
(min) Used Energy Use

Control House

House with Door Opening Every 2 Min

House with Door Opening Every 1 Min

House with Vestibule Opening Every 2 Min

House with Vestibule Opening Every 1 Min

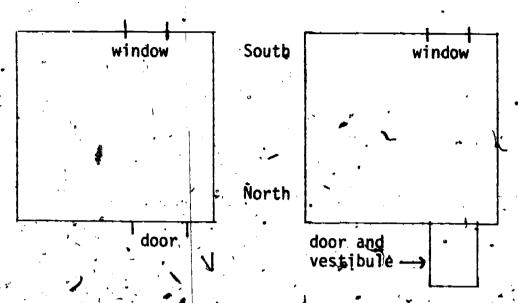
QUESTIONS:

- 1. Which house used the most energy to maintain the designated temperature?
- 2. Which house used the least energy?
- 3. Consider houses which are set up as follows:

Door and

Vestibule

Door Only



If the houses above were to be operated under the same conditions as the houses in which you opened the door every? minutes for 5 seconds, predict whether there would be more; less, or the same amount of energy used under the following conditions:

١.		<u> </u>)
a.	Window	slightly	open:(n	o wind).

- b. Window closed (north wind).
- c. Window closed (south wind).
- d. Window slightly open (east wind).
- e. Window closed (east wind).

4.	A vestibule is only one way of reducing heat losses through door	's in
, .	both residential and commercial buildings. List several other m	ethods
•	which are used to reduce heat loss through doorways.	•

,	•		
.*		•	

TEACHER'S GUIDE

OBJECTIVES: Upon completion of this activity, the student will be able to:

- 1. Take accurate data of temperature and time in a model house heated by a light bulb.
- 2. Calculate the amount of energy in kilowatt-hours required to heat the model house.
- 3. Explain the effects of opening and closing doors on building heat loss.
- 4. Calculate the percent increase in energy consumption as a result of opening doors.
- 5. List several methods of reducing heat losses through doorways:

BACKGROUND: In the energy-conscious society in which we live, much has been written and said about conservation of energy. In this activity, students will focus on one single aspect of conserving energy; i.e., reducing heat loss through doorways. Comparisons will be made among a closed house, a house with a conventional doorway opened at regular intervals, and a house with a vestibule doorway opened at regular intervals. Students will be able to calculate the percent increase in energy use due to door openings.

, TIME REQUIRED: Two 45 minute periods for data collection

One-two 45 minute periods for calculations and questions

MATERIALS (per group): For a class of 30 working in pairs, you will need 15 model houses, 6 with vestibules and 9 with conventional doors. These model houses can be constructed using small styrofoam coolers which have

been fitted with a small window on top (Saran wrap or plastic will work) and appropriate doors. It is suggested that the door be designed so that a string will pull the door up, allowing it to easily fall back into place when the string is released. Be sure to have a way to close the door securely.

The styrofoam cooler would be placed upside-down on a piece of wood onto which has been mounted a light bulb, socket and thermostat. Putty may be used to make a tight seal between the wood and the styrofoam. In addition, there must be a small hole for a thermometer. The thermostat should be located near the thermometer.

Thermostats are available at a nominal cost from heating companies and hardware stores. Many people have installed new clock thermostats, which leaves many old conventional types available.

These model houses could also be used for other empirical applications such as determining effects of insulation and stratification of temperature in houses.

SUGGESTIONS:

- 1. Be sure to set up each house with identical conditions, including door size, type of thermostat, type of bulb, accurate thermometers and the same initial temperature.
- 2. If necessary, calibrate each thermostat to be standard with the control houses.
- 3. If you wish to reduce the amount of time spent in lab, split the data collection between the lab teams and pool data at the end of one period.



- 4. Make sure that the students understand that they are not really measuring the amount of heat, since a light bulb is not an efficient source of heat.
- 5. Make sure that the thermostat setting that you choose is high enough to cause a reasonable heat demand. If the activity is done in a classroom at $65^{\circ}F$ ($18^{\circ}C$), a suggested thermostat setting would be $95^{\circ}F$ ($35^{\circ}C$).

SUGGESTED FOLLOW-UP ACTIVITIES:

- 1. This activity could be lengthened by testing the effect of wind (electric fan) on different housing orientations.
- 2. A calculation of heat loss in a normal house is an exercise which can point out the other heat loss causes.
- 3. A similar activity with these model houses could illustrate the effectiveness of various types of insulation.

REFERENCES:

University Physics, Sears and Zemansky, Addison Wesley.

Physics for Students of Science and Engineering, R. Resnick and D. Halliday, John Wiley and Sons.

ASHRAE Fundamentals, 1977

Solar Heating and Cooling, J.F. Kreider and F. Kreith, McGraw-Hill Book Co.

#8. CALCULATING ELECTRICAL CONSUMPTION IN YOUR HOME

<u>PURPOSE</u>: The purposes of this activity are: 1) to do a home survey of electrical appliances; 2) to calculate the number of kilowatt hours of electricity consumed annually by these appliances and; 3) to calculate the approximate annual cost of operating these appliances.

INTRODUCTION: Most people are not aware of the number of electrical appliances found in their homes. We would probably tend to guess we have fewer appliances than we actually possess. Few of us have any concept of the quantities of electricity consumed by our appliances. Each appliance will consume a certain quantity of electricity during the course of a year and few of us complete the collective effect or cost of operating our home appliances.

There are two methods by which one can attach a price tag to the usage of electricity in your home. One method is to read the electric meter each day. This will (by subtraction), tell you the number of kilowatt hours used per day. The problem with using this method is that it does not allow one to determine or analyze which appliances are consuming the greatest amounts of electrical energy.

The second method is to calculate the electricity consumed by each appliance and add the individual calculations to attain a total daily usage. This method entails a great number of calculations and a consideration of the time each appliance was operating.

by multiplying electrical pressure in vol. by current flow in amperes.

If a one-watt appliance is run continuously for one hour, it will consume

one watt-hour of electricity. A 100-watt light bulb will use 100 watt-hours of electricity every hour it is on. Your electric meter measures kilowatt-hours of electricity. One kilowatt-hour is equal to 1000 watt-hours, so ten 100-watt light bulbs turned on for one hour will consume one kilowatt-hour of electricity. Kilowatt-hours are units of energy.

The cost of electricity is calculated by multiplying the number of kilowatt-hours used by the rate per kilowatt-hour. Assuming the electric rate is 5 cents per kilowatt-hour, it would cost 0.6¢ to play records on a 120-watt solid-state stereo for each hour.

MATERIALS: "Estimated Annual Energy Consumption of Appliances" (Appendix D)

Home Data Sheets

PROCEDURE: (1) Survey your home for electrical appliances. Write the number of each appliance found in your home on the data sheet.

- (2) Multiply the number of each appliance by the average number of kilowatt-hours used by each appliance annually.
 (3) Multiply the number of kilowatt-hours used by your electric rate.
- (4) Add the costs to find the total annual electric bill.

QUESTIÓNS:

1. Does your home have more or fewer electrical appliances than you might have guessed before conducting the survey?

2. In your opinion, is your home above average, average, or below average in regard to the number of electrical appliances?

-Data Sheet .

Appliance	Number	Average And in	nual Consumption KWH	Total KWH Used Annually	Annual Cost @ 5¢ KWH
ajor:		,			
Air Conditioner (room)	· .	8	60		
Air Conditioner (central)		12	240	•	
Clothes dryer		9	93		
			\		•
•			7	<u> </u>	
			•		
• **	•	•			 ,
		- /	•	•	
				•	
				•	
TOTALS		/			
TOTALS	••	0	•		
, Addition	Number	Watts	Estimated Hours Used per Year	Total KWH Used Annually	Annual Cost 0 5¢ per KW
Additional Appliances:	•	,			
	,	. •		1	
			•		
***************************************	•			. ,	

3.	on electric hair-curling iron is rated at .75 amps, and is used
	an average of 15 minutes per day 6 days per week, calculate the
•	annual cost using your local utility rate.
	difficult cost using your rocal definity race.
4.	Of the electrical appliances found in your home, which five con-
	sume the greatest amounts of electricity annually?
.5.	If a federal law were enacted that would allow each home to have - only five electrical appliances, which five would you keep? Brief-
	ly explain why you would keep those five.
•	
6.	If your home were to be allotted a limit of 6000 Kwh per year with
	substantial penalties for exceeding the limit, which appliances would you keep?
w.Q<.	
7.	What effects would the doubling of electric rates have in your
	home? Why? What effects would occur to people on fixed incomes?
. 4	
8.	Calculate the percentage of your family's annual income that is spent
•	Ton electrical power.
•	
9.	Calculate the difference in annual cost of operating a tube-type
,	radio (125 watts) and a solid-state radio (25 watts) if both are
. 	used six hours per day.

TEACHER'S GUIDE

OBJECTIVES: Students will be able to: (1) survey their own homes for electrical appliances, (2) calculate the energy (in kilowatt-hours) consumed by their electrical appliances individually and collectively, (3) calculate the cost of operating their electrical appliances, (4) develop an awareness of electricity consumption and costs.

BACKGROUND: Electrical consumption due to home appliances has increased dramatically in the past few years. National studies indicate that consumption is increasing at 7% annually. This represents a doubling of consumption every 10 years.

Most are not aware of the number of electrical appliances found in their homes, the amounts of energy consumed by these appliances, or the cost of operating these appliances individually or collectively.

Some basic equations used in this activity are:

Power, Watts = Electrical Pressure, Volts x Current Flow. Amps

One kilowatt = 1000 watts

Energy, kilowatt-hours = watts x time (in hours)/1000 Cost = Energy kilowatt-hours x electric rate, cents/kilowatt-hour

TIME REQUIRED. Two class periods (55 minutes each); homework time

MATERIALS: Data sheets.

- SUGGESTIONS: 1) Electric rates vary according to region. Call your electric company and get the actual rate.
 - 2) Preview the data sheet with students, show how to calculate annual consumption and cost and allow students

several days to complete their surveys.

- 3) Give students the opportunity to add appliances to the data sheets. They will have to calculate kilowatt-hours used annually by these appliances.
- 4) Note: "furnace fan" is listed on the data sheet.
- you might anticipate the five appliances selected to keep will be: range, clothes washer, furnace fan, TV, and refrigerator. If six are selected, the clock will probably be sixth.

ANSWER 10 QUESTION #3: (Based on 5¢/kwh)

watts = volts x amps

120 v x 1.75 a = 90 watts

kiTowatts = 90 watts/1000 = .09 kw

time = 15 min/day x 6 days/week x 1 hr/60 min x 52 weeks/yr = 78 hrs/year

 $k \dot{w} h/y ear = 78 hrs/y ear x .09 kw = 7.02 kwh/y ear$ $cost = 7.02 k \dot{w} h/y ear x .05 = $.351 per y ear$

ANSWER TO QUESTION #9: (Based on 5¢/kwh)

Tube type:

125 watts x 6 hrs/day = 750 watt-hours/day
750 watt-hours/1000 = _75 kilowatt-hour/day,
.75 kwh/day x 365 days = 273.75 kwh/yr =

273.75 x .05 = \$13.69

Solid State:

25 watts. x 6. hrs/day = 150 watt-hours/day 150 watt-hours/day/1000 = .15 kilowatt-hours/day .15 $kwh/day \times 365 days = 54.75 kwh$

 $54.75 \times .05 = 2.74

Difference = \$13.69 - \$2.74 = \$10.95

Note: The following may be of interest:

U.S. Households with Major Appliances, 1973 (percent)*

· Clothes washer - 78

Refrigerator - 99

Nonautomatic - 10

.Manual defrost - 48

Automatic - 70

Frostless - 51

Dishwasher - 25

Stove - 97

TV - 97

Gas - 52

Black & white - 64

Electric - 46

Color - 53

Freezer - 34

Clothes Dryer - 53

Gas - 16

Electric - 38

*The American Energy Consumer, Dorothy K. Newman & Dawn Day, Ballinger Publishing Company, Cambridge, Massachusetts, 1975.

REFERENCES:

- 1. The American Energy Consumer, D. Newman and D. Day, Ballinger Publishing, 1975,
- 2. Tips for Energy Savers, Federal Energy Administration, August, 1977.
- 3. The Home Energy Guide, J. Rothchild and F. Tenney, Ballantine Books, 1978.
- 4. Energy Conservation in the Home, U.S. Department of Energy, October, 1977.

#9. THE FFICACY OF LIGHT SOURCES

PURPOSE: To determine the efficacy of various light sources.

INTRODUCTION: The lighting efficiency of lamps and other light sources is termed EFFICACY and is determined by dividing the light output in lumens by the power used by the light source in watts. Therefore, efficacy units are lumens/watt. In this activity different lamps will be tested to determine their efficacies as well as each lamp's desirable and undesirable characteristics.

MATERIALS: (each group)

G raph' paper

- 1 photometer and associated equipment
- 1 incandescent lamp fixture
- 1 15 watt fluorescent fixture
- 2 cardboard boxes, insides painted black and large enough to cover the fixtures
- 3 low wattage lamps, incandescent. Example: 15 watt, 60 watt, none greater than 100 watts
- 2 or more different 15 watt fluorescent lamps

PROCEDURE:

- Place a known standard light source at one end of a photometer.

 Place a box with one open side over the bulb. The interior should be black and the open side facing the photometer.
- 2) On the other side of the photometer set up the unknown light source in the same way.

- 3) Slide the photometer until the light is the same intensity on both sides of the meter stick. Measure the distances of light sources to photometer.
- 4) Using $E = \frac{1}{d^2}$, where

E is the light intensity falling on the photometer in lamens/meter. I is the light output in lumens (I_{known} found from bulb cartons) and is the distance from the photometer to the light source in meters.

If
$$\frac{E}{known} = \frac{E}{unknown}$$
,
then $\frac{I_{known}}{d_{known}^2} = \frac{I_{unknown}}{d_{unknown}^2}$

$$\frac{\int_{k \text{nown}} x \, d^2_{unknown}}{\int_{k \text{nown}}^2 x \, d^2_{unknown}} = \frac{1}{unknown}$$

Find I unknown in lumens and record on the data page.

5) Find the EFFICACY of light source $EFFICACY = \frac{1}{power}$

where I is light output in lumens and the power input (found on bulb) is in watts.

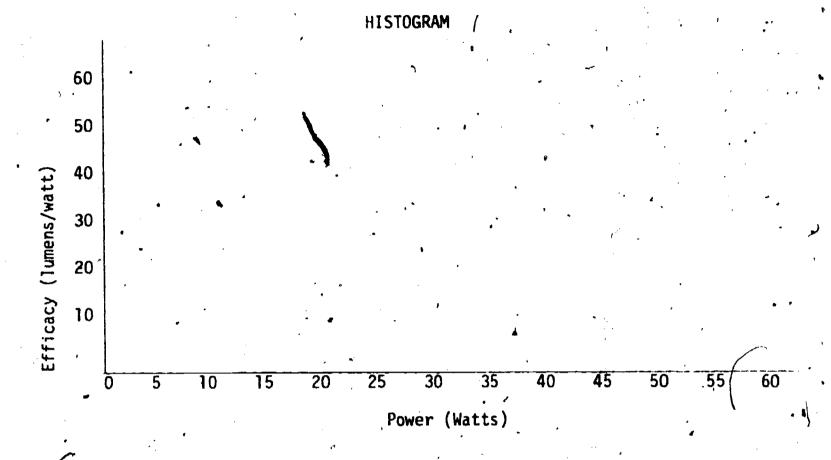
- 6) Repeat steps 3 through 5 for two more incandescent lamps and record results on the data page.
- 7) Replace the unknown incandescent lamp fixture with the fluorescent fixture and put in a fluorescent lamp.
- 8), Repeat steps 3 through 5 and record on data page.
- 9) If your teacher has any other special bulbs to test, then repeat steps 3 through 5 and record on data page.

10) Make a dual histogram of power in watts as a function of efficacy in lumens/watt. Distinguish the two lamp types with different colors.

DATA

Lamp	Illumination (lumens)	Distance (meters)	Power (watts)	Efficacy (lumens/watt)
Known Incandescent		,		
1)	,	, .		
2)				,
3)	•			
4) Known Fluorescent			, ·	
5)	• .	÷	•	
6)	•		,	
7)			-	89
8)	,			

91



CONCLUSIONS:

- 1) What conclusions can be drawn from the histogram?
- 2) How does the fluorescent lamp compare with the incandescent lamp?
- 3) Give some of the advantages and disadvantages of:
 - a) fluorescent lamps:
 - b) incandescent lamps.
- 4) When installing light fixtures in the home, where would you locate the different types of lamps? Give your reasons.

TEACHER'S GUIDE

OBJECTIVE: Students will be able to apply a standard photometric method in the determination of efficacies for several hight sources.

BACKGROUND: This is a consumer education activity that will increase the understanding of the positive and negative energy consumption aspects of typical lamps found in the home and school.

TIME REQUIRED: One 50 minute period is recommended for this activity; however, several follow-up activities listed below could be added to help, the student enhance his understanding of efficacy.

MATERIALS: The basic materials are listed on the student activity pages. It is recommended that the teacher have 3 different incandescent lamps; e.g., 25-, 60-, 75- or 100-watts. More lamps of different wattages could be used if you desire. When purchasing fluorescent lamps, the suggested wattage to use is 15 watts.

There are several special lamps that can be found in schools, including projection lamps, for example. Check with your custodian or audiovisual staff.

SUGGESTIONS: This activity should work well as a follow-up to the standard photometer experiment found in most physical science or physics sources. Students seem to enjoy determining the efficacy of unknown light sources.

FOLLOW-UP ACTIVITIES:

- 1) Try asking the student to determine the efficacies of some special light sources such as high intensity, halogen, mercury and projection bulbs.
- 2) Determine the effect of reflectors with the use of lamps. (This may be done by placing aluminum foil inside the box.)
- 3) Have students check the wattage, lumens and efficacy of the lifetime (5 year) light bulbs. (They will generally be low.)
- 4) Over the open side of the box place a cover and insert a thermometer into the top of the cardboard box. Turn on the lamp and take temperature readings over a period of time (5 or 10 minutes); then graph temperature as a function of time.

SAMPLE DATA:

Incandescent (all frosted):

Power (watts)	Illumination (lumens)	Efficacy (lumens/watt)
15	125	8.3
25	235	9.4
40	455	11.4
60	870	11.4 , '*
	1190	15.9
100	1750	. 7.5

Fluorescent lamps: ^

Power (watts	Name	Illumination (lumens)	Efficacy ((lumens/watt)
15"	cool white .	870	58
·* n	deluxe cool white	610	(40
	warm white	600 `	40
b 11	y warm	3 870	58

^{*} For GE bulbs only, others will show lower efficacy.

REFERENCES:

IES Lighting Handbook
Illuminating Engineering Society
Standard Lighting Guide
, 3rd Ed. 1962

G.E. Lamp Catalog, 1977 Ed. Form 9200

#10. USING WIND POWER TO PRODUCE MECHANICAL WORK

PURPOSE: To demonstrate the conversion of wind energy into work and the effect of wind velocity and aeroturbine diameter on the conversion.

MATERIALS: Fan with variable speed; wind tunnel, anemometer or other windspeed measuring device; ring stand and clamp for holding wind machine; 20- and 50-gram mass standards; classroom set of wind machine heads; cardboafd, acetate, or metal for fins; string.

INTRODUCTION: Wind power, derived from solar radiation, is an obvious force acting in our physical world. Winds can range from light cooling breezes of 1-7 mph to destructive hurricanes with speeds above 50 mph. Harnessing wind power has been a challenge for centuries, but the name of the man who discovered that wind could be used to turn a wheel has long been forgotten.

Wind machines have been called windmills (used to grind grain), wind pumps (used to pump water) and wind generators (used to generate electricity). Any device used to capture wind power by the rotation of a wheel is called an aeroturcine. It can be made in a variety of shapes, but the most familiar type consists of a circular head with two or more fins attached.

The power of the wind can be determined from the kinetic energy of a moving mass of air:

 $KE = \frac{1}{2}MV^2$

where M = mass of air in kg, V = velocity of air in meters/sec

By finding the mass of air passing through the aeroturbine per unit time,
we can arrive at power (energy per unit time).

The mass of air passing through an aeroturbine during a given time period is a product of the density of air, the area of the aeroturbine facing the wind and the velocity of the wind:

$$M = \rho AV$$

where $\rho = density of air in kg/m^3$

A = the area of the aeroturbine in m^2

V_i = the velocity of the wind in m/sec
Substituting:

Power input $(P_i) = KE/unit time = \frac{1}{2} (\rho AV) V^2$ Therefore,

$$P_{i} = \frac{1}{2} \rho A V^{3}$$
 (1)

From Eq(1) it can be seen that the power of the wind is directly proportional to the square of the diameter of the aeroturbine (A = $\pi \frac{d^2}{4}$) and to the cube of the wind velocity.

The power of the wind obviously cannot be completely coverted to usable power by a wind machine. A. Betz of Germany determined that the theoretical efficiency can be as high as 59.3%, but the actual efficiency of a well-built machine is about 40%.

In this experiment you will measure the power output of a model wind machine by having it lift a weight. The kinetic energy resulting from a vertical linear motion is the mass times the acceleration due to gravity times the distance the mass moved. Since power is kinetic energy per unit time:

Power output $(P_0) = KE/unit$, time = $P_0 = M_2gh/t$ (2) where $M_2 = mass in kg of the mass standard,$

g = acceleration due to gravity, 9.8 m/sec²,

h = height in meters, and

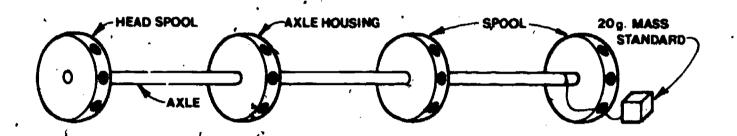
t = time in seconds.

Since efficiency equals power output divided by power input, you will be able to calculate the efficiency from the Equations (1) and (2).

$$\frac{\text{PROCEDURE}}{\text{Procedure}} \approx \text{efficiency} = \frac{P_0}{P_1} \times 100 = \frac{M_2 gh/t}{\frac{1}{2} \rho \text{ AV}^3} \times 100$$

1. Your wind machine head consists of the following parts:

Figure 1. Wind Machine Head



Place four spokes with fins attached evenly around the head spool.

- 2. Test the position best swited to produce constant rotary motion by rotating the "spokes" and fins to different positions. Try placing the fins parallel, perpendicular, and at various angles to the plane of the head spool and check which position produces the fastest spin when placed in front of the fan. Attach the axle housing to a clamp on a ring stand before using the fan to test spin.
- 3. Attach 30 cm of string to the axle as shown in Fig. 1 after you have found the best fin angle. Attach a 20 g mass standard, shortening the string so that it will rest on the table top or floor when the wind machine head is attached to the ring stand clamp.
- 4. Check the wind speed using the wind meter.
- 5. Place the wind machine in position and measure the time it takes for the mass standard to be lifted to the axle.



- 6. Repeat Step 5.
- 7. Replace the 20 g mass standard with a 50 g mass standard and repeat Steps 5 and 6.
- 8. Make two trials with one of the mass standards using a different wind speed.
- 9. Make two trials with one of the mass standards using larger fins.
- 10. Record all data in the table below.

DATA TABLE 1

Temperatu	re (°C)					•	٠ ٥
Trial No.	V (mph)	V(m/sec)	Mass (g)	Mass (kg)	h (m)	t (sec)
	•	_					
	1		1				•
			- 8			•	
		•					•
					•		
	n						
. 1		,					
		,			,		•

CALCULATIONS: Density varies with the temperature and pressure according to the Ideal Gas Law:

$$\dot{v}_2 = v_1 \frac{T_2 P_1}{T_1 P_2}$$

where: v_1 , T_1 and P_1 are the volume, absolute temperature and pressure (respectively

STP (Standard conditions of Temperature and air Pressure) and v_2 , v_2 and v_2 reflect your local conditions.

Since the density has an inverse relationship with the volume:

$$\rho_{2} = \rho_{1} \frac{T_{1}P_{2}}{T_{2}P_{1}}$$

(3)

where ρ_1 is STP density and ρ_2 is classroom air density.

(Density of air at 0° t and 760 mm pressure is 1.30 kg/m³.).

Convert to the density at your location and room temperature.

(Reminder: Temperatures must be expressed in Kelvin degrees to use the

Gas Law!)

Efficiency =
$$\frac{\text{power output}}{\text{power input}} \times 100$$
 (4)

Conversition units needed:

 $1 \text{ mi} = 3.61 \times 10^3 \text{ meters}$

 $1 hr = 3.6 \times 10^3 sec$

1 g = ,001 kg

gas(acceleration due to gravity) 9.8 meters/sec² on the surface of the earth

CALCULATIONS

		•		<i>I</i>			
	Trial No.	V3.	Diameter (m)	Area (m ²)	Power input $(P_i = \frac{1}{2} PAV^3)$	Power output Po= Magh/t	Efficiency (output input x 100)
-	•		, ,				ų s.
•	, o		12.				
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CONCLUSION: Include in your written conclusion: 1) the efficiency values you obtained in terms of the theoretical value, 2) the relationship between the power output and the velocity of the wind, and 3) the relationship between the power output and the area of the aeroturbine.

QUESTIONS:

- 1. How could friction in a wind machine be reduced?
- 2. What is the single most important factor which reduces the theoretical efficiency to 59.3%?
- 3. Why is the actual efficiency less than the theoretical efficiency?
- 4. What major factor would reduce the amount of power obtained from a full-sized wind machine?
- 5. List areas of major cost in erecting applied machine?

EXTENDED ACTIVITIES:

- 1. Build a model wind machine designed to operate more efficiently.
- 2. Write a report on wind generators.
- 3., Write a report on the structure of fins.
- 4. Write a report on placement of wind machines in terms of local and regional topography.
- 5. Wind machines operation when the winds range from 10 to 20 mph.
 Study your locality to check whether a wind machine would be practical.
- 6. Write a report about how the winds are driven by the sun.

TEACHER'S GUIDE

OBJECTIVE: The student will demonstrate the power of the wind, determine the amount of useable power that can be extracted from the wind, and determine the effect of wind velocity and aeroturbine area on power output.

BACKGROUND: The wealth of information about wind power is almost overwhelming. There are records of quantitative studies done as early as 1759.

Understanding how wind is produced makes a fascinating study. Tropical air rising to the upper level of the troposhere causes cold polar air to move in to replace the rising warmer air. Coriolis forces, resulting from the rotation of the earth, also play a large part. Localized winds, of course, depend on the topography of an area.

Historically, wind power included using sails which were attached to boats and bicycles. The first windmill was used to grind grain, and those wind machines used to pump water, their most common usage in the U.S., are actually wind pumps. Wind machines used for generating electricity are called wind generators or wind chargers.

Students may be particularly interested in wind power to generate electricity. Supposedly, home wind generators are capable of generating electricity into a power grid when both systems are hooked together.

Home Energy - How To has a very practical description of the advantages and disadvantages of home wind generators, as well as approximate costs of some recent attempts to harness wind for the generation of electricity.

Wind energy as an alternative source of power has several major draw-backs: 1) Wind does not blow steadily. 2) Storage of energy produced from



wind has many more problems than storage of energy from the sun. 3) The super-structure required to support a wind generator adds appreciably to the cost. 4) Wind energy at a particular location is hard to quantify. Although many studies have been done on the types of rotors and, in the case of propeller machines, the types of blades which work best, most of these studies are essentially qualitative in nature. A most useful quantitative study is described below.

A. Betz in Germany calculated the theoretical efficiency of wind machines by applying the momentum theory used for ship's propellers. Three velocities exist when wind passes through a wind machine:

V₁ = wind speed before the rotor

. . V = wind speed through the rotor

V2 = wind speed beyond the rotor

If M is the mass of air passing through the rotor per unit time, the rate of change of momentum is $M(V_1 - V_2)$ and this is the resulting thrust. Power can be found by multiplying the thrust by V: $M(V_1 - V_2)V = Power$.

The rate of change of kinetic energy is $1/2 \, M(V_1^2 - V_2^2)$ (1) which, when simplified, indicates that the wind speed through the rotor equals the average V_1 and V_2 , or $V_1 + V_2$

 $V = \frac{V_1 + V_2}{2}$

The mass of air passing through the rotor per unit time is equal to the air density times the area of the rotor times the wind velocity:

$$M = \rho AV$$

The power, P, is

$$P = pAWA(V_1 - V_2) V$$

Substituting $V = (V_1 + V_2)/2$,

$$P = \rho A \left[\frac{V_1 + V_2}{2} \right]^2 V_1 - V_2$$

Substituting $e = V_2/V_1$, power becomes

$$P = \frac{\rho AV_{1}^{3}}{4} \cdot (1 + \alpha) (1 + \alpha^{2})$$

Multiplying to simplify,

$$P = \frac{\rho A V_1^3}{4} \quad (1 + \alpha - \alpha^2 - \alpha^3)$$

differentiating,

$$\frac{dP}{\alpha d} = \frac{\rho AV_1^3}{4} (1 - 2\alpha - 3\alpha^2)$$

The maximum power occurs when the slope $\frac{dP}{d\alpha}$ =-0.

Solving for α , the power is maximum when $\alpha = 1 \neq 3$.

Thus, the power is maximum when the final velocity, V_2 , is one-third of the upwind velocity, V_1 .

Maximum power is obtained when

$$P = AV_1^3 \times 8/27$$
 as compared with $P = -\frac{\rho AV_1^3}{2}$ in the wind,

originally, or 16/27 (59.3%) becomes the maximum obtainable power. (Refer to E.W. Golding, The Generation of Electricity by Wind Power.) A similar coefficient has been developed for propeller wind machines, using aerodynamic principles of lift and drag. In that case, the coefficient is slightly higher, about 68.7%.

Both derivations ignore such factors as friction and other losses, resulting in the actual maximum efficiency of 40%. 11%



If students are familiar with rotational calculations, it is possible to approach the problem in terms of torque. Since the wind is acting perpendicular to the face of the wind machine, only that force which is acting parallel to the face of the wind machine on the tipped blades will produce a torque. This torque will, in turn, set up an angular velocity which has kinetic energy or power in terms of energy per unit time. This must be translated into linear energy to lift the mass attached to the shaft.

Listed below are some references. Practical-minded students would also enjoy articles from Popular Science or Mother Earth News which tend to feature unusual uses of alternative energy.

MATERIALS: One set of Tinker Toys, Set 136 (about \$4.95) can supply eight sets of wind machine heads, if you use only one axle housing; four, if you use two. If your class is small, a more stable wind machine head can be prepared by using two axle housings attached to two clamps and two ring stands. These axle housings are called 'W' in the set. The orange rods form the spokes, and the order for the rest consists of spool, blue rod, axle housing, spool, second axle housing (if used), yellow rod, string final spool or a wheel. Each of the 'W' pieces (axle housings) must be secured by the orange washers supplied or lock washers. You may also have to cut extra pieces the same size as the orange rods for spokes. A handheld wind meter is available for under ten dollars. If you have an anemometer available, it would be more accurate.

A propeller-type, portable fan with variable speed control which is larger than 12 inches in diameter should supply speeds from 10-12 mph.

A rheostat (try borrowing one from the drama dept.) would make it possible

to use more precise velocities; e.g., 6 mph or 12 mph, 5 mph and 10 mph, which would make the increase in power as a cubic function of the velocity easier to observe. A streamlining grid consisting of inch-wide boards placed in a network of squares (about 2 inches square) will reduce turbulence. A wind tunnel, perhaps made from a large cardboard box, might increase wind speed and reduce turbulence.

TIME REQUIRED: One period for actual operation, with 6 to 8 groups. More time may be needed if you choose to give an introduction to wind energy and its technical potential or if you feel that calculations need to be done with your class.

SUGGESTIONS:

Answers to questions: -

- J. Use an oiled metal axle holder.
- 2. The fins--not all the wind power can be captured.
- 3. Theoretical does not take into account friction losses, etc.
- 4. Wind is not steady.
- 5. Support structure.

Students will do well to get 5% efficiency.

Cardboard fins may be made out of heavy cardboard (such as a puzzle box).

ADDITIONAL ACTIVITIES:

- 1. Determine the maximum lifting mass for the wind machine.
- 2. Measure the effect of increasing or decreasing the number of fins.
- 3. Change the shape of the fins. Measure the effect.

4. An advanced class might wish to build a school wind generator.

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Inc., Harper and Row, New York, 1977. Very practical. Also realistic
on costs.

Book, John Wiley and Sons, Inc., New York, 1955. A very technical

Crowther, Richard L., <u>Sun Earth</u>, A.B. Hirschfield Press, Inc., Denver, Colorado, 1976. Good general information.

Hamilton, Roger and Emory Kristof, "Can We Harness the Wind?", National Geographic, Vol. 148, No. 6, December 1975, pp. 812-829. Good pictures. Hackleman, Michael A&, Wind and Windspinners, Peace Press, Culver City, California, 1974. Interesting.

ANALYTICAL DETERMINATION OF THE SOLAR ANGLE OF INCIDENCE ON A FLAT PLATE COLLECTOR

INTRODUCTION: Consider the rays of the sun as sources of heat and energy. Many calculations for solar energy are applications of basic trigonometry.

PROCEDURES, CALCULATIONS AND RESULTS:

A. Consider a flat plate solar collector as an example (Fig. 1):

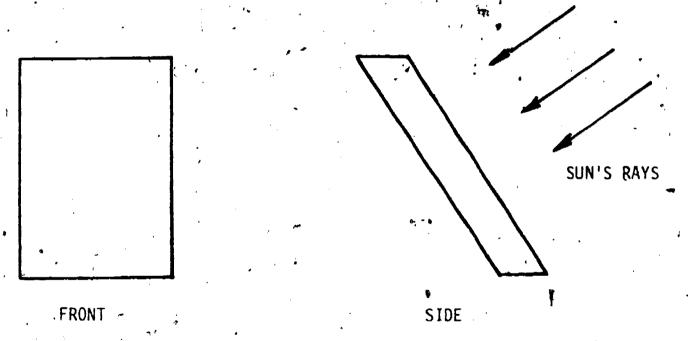


Figure 1. Flat Plate Solar Collector

Note that the sun's rays are considered to be parallel. The placement of the flat plate of the collector relative to the direction of the sun becomes critical. The direct normal component of solar insolation, $I_{\rm h}$, is given by the formula

$$I_b = I_{\text{maximum}} \cdot \cos \blacktriangleleft i$$
 (1)

where d i is the angle of incidence as shown in Fig. 2 and I maximum is the amount of solar insolation on a flat surface.

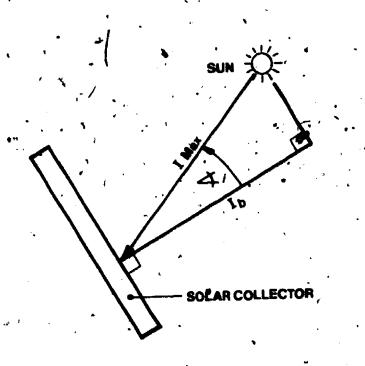


Figure 2. Side View of Collector

Does it seem logical that the <u>maximum</u> insolation will be received when the sun's rays are perpendicular to the collector face? Verify this from the formula $I_b = I_{max} \cos 4$ i when $4 i = 0^{\circ}$. (Hint: Recall the graph of the cosine shown in Fig. 3.)

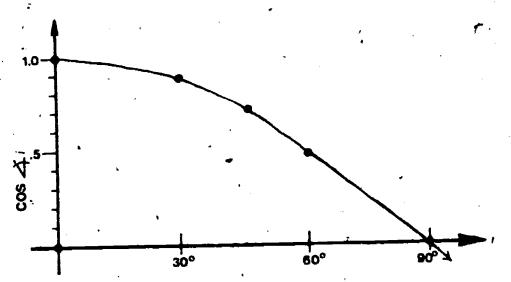


Figure 3. Cosine Curve



(Note that the graph of the cosine is at a maximum when \blacktriangleleft i \neq 0.)

Once the collector plate is in place, the sun will move relative to the collector. Note the apparent motion of the sun.

Exercise: What percentage of the maximum insolation will the solar collector have available as the angle of incidence varies? (Hint: Use equation (1), trig tables or the trig functions on a calculator to work the following excercise.

EXCERCISE:

Angle of Incidence, 4 i	%,of Maximum Solar Insolation		
a) $i = 10^0$			
b) i = 20 ⁰			
c) i = 30 ⁰	*		
(a) $i = 45^{\circ}$			
e) i = 60 ⁰			
f) i = 90 ⁰			

After 60^{0} you might as well zip yourself into a down sleeping bag and forget about splar energy.

B. Determining the Angle of Incidence. Now that you can see the effect of the angle of incidence of the sun's rays on the collector plate, of what is that angle i'a function? How does one determine < i for a south-facing collector?

Part 1. DETERMINE δ .

First, let's consider

 δ = the solar declination angle.

This angle varies with the day of the year as the earth rotates around the sun and changes because the earth's axis is "tilted" 23.5° from the vertical. See Figure 4.

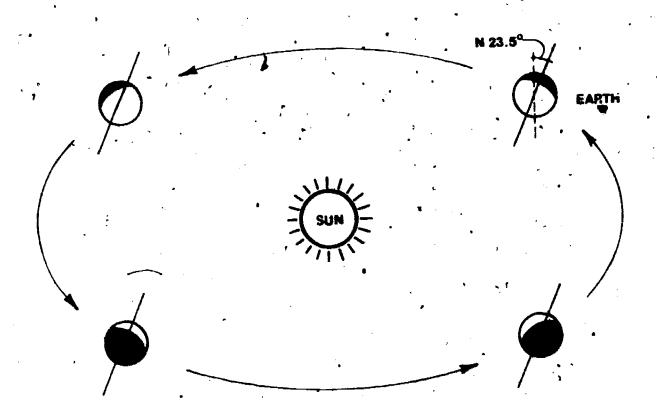


Figure 4. Earth's Revolution Around the Sun To find the solar declination $\checkmark \delta$ for any day of the year,

Dec 31 = day 365

Then
$$4\delta = 23.5 \sin \left[360 \left(\frac{284 + \text{day}}{365} \right) \right]$$
 (2)

113

Examples:

a) Jan 1,
$$< \delta = 23.5 \sin \left[360 \left(\frac{284+1}{365} \right) \right]$$

= 23.5 sin 281.10 = -23.06

b) Mar 21,
$$4\delta = 23.5 \sin \left[360 \left(\frac{284+81}{365} \right) \right]$$

= 23.5 sin 360°
= 23.5 \ 0

Note that March 21 is the vernal equinox when the day and night are of equal length. How convenient that on that day, $\sin = 0$ and that the angle of declination, $\delta = 0$. (Of course, this is not a coincidence.)

Exercises:

Find \lt δ for the following dates using Eq. 2.

a) May 21 (day 141)

46

b) September 21 (day 261)

(Hint: September 21 is the autumnal equinox) $\angle \delta$

c) Dec. 31 (day 365)

4δ

Part 2. DETERMINE L

Now that you have found δ , the angle of solar declination, consider the latitude L of the site of the flat plate collector. This

is easy enough to find on a map, a globe, or an atlas. (Let $L = 40^{\circ}N$ unless otherwise specified). See Fig. 5.

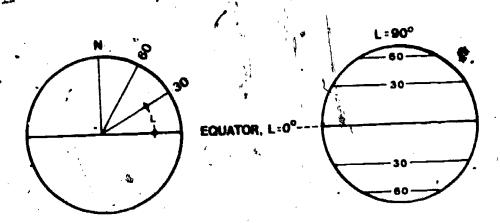


Figure 5. Latitude

3. Determine H

Next, consider H, the hour angle, for you know that the sun's angle varies throughout the day. The hour angle, $\angle H = 15^{\circ}$ x the number of hours away from solar noon.

A quick approximation may be made by using local standard time.

$$\angle H = (12:00 - Standard Time) \cdot 15^{0}/hr$$
 (3)

Examples:

at 11 a.m. (1 hour before noon), $\angle H = +15^{\circ}$

at 9 a.m. (3 hours before noon), $\angle H = +45^{\circ}$

at 7:30 a.m. (4.5 hours before noon), $\angle H = +67.5^{\circ}$

at 2 p.m. (2 hours after noon), $\angle H = 30^{\circ}$

Part 4. DETERMINE α

Now we have the information we need to find α , the angle between the sun and the horizon. The solar altitude, α , as seen by an observer on the earth is depicted in Fig. 6.

· 113

NT12

HORIZON

Figure 6. Solar Altitude a

 α is determined by Eq. 4:

 $\sin \alpha = (\sin L \sin \delta) + (\cos L \cos \delta \cos H)$

(4)

where you will recall.

L = latitude

 δ = angle of declination.

H = hour angle

Example; Find α at L = 45° north, at 10 a.m. standard time on

March 21, where $\delta = Q^0$.

 $H = (12:00 - 10:00) \times 15^0 = 20^0$ (from Eq. 3)

Then from Eq. 4:

 $\sin \alpha = (\sin 45^{\circ} \sin 0^{\circ}) + (\cos 45^{\circ} \cos 0^{\circ} \cos 30^{\circ})$

 $\sin \alpha = (.707 \cdot 0) + (.707 \cdot 1 \cdot .866)$

 $\sin \alpha = .612$

 $\alpha = 37.76^{0}$

Exercise: Use some of the data you have found from previous exercises.

Find α for $\frac{1}{4}$ = 40° north, H = TO a.m. standard time on May 21

(Day 14):

 $\delta = \frac{1}{2}$ (Hint: Use Eq. 2.)

$$AH = \frac{1}{100}$$
(Hint: Use Eq. 3)
(Hint: Use Eq. 4)

DETERMINE Y Part 5.

There is one more necessary preliminary angle to find.

= the solar azimuth, the angle in the horizontal plane between due south and the projection of the sun's rays.

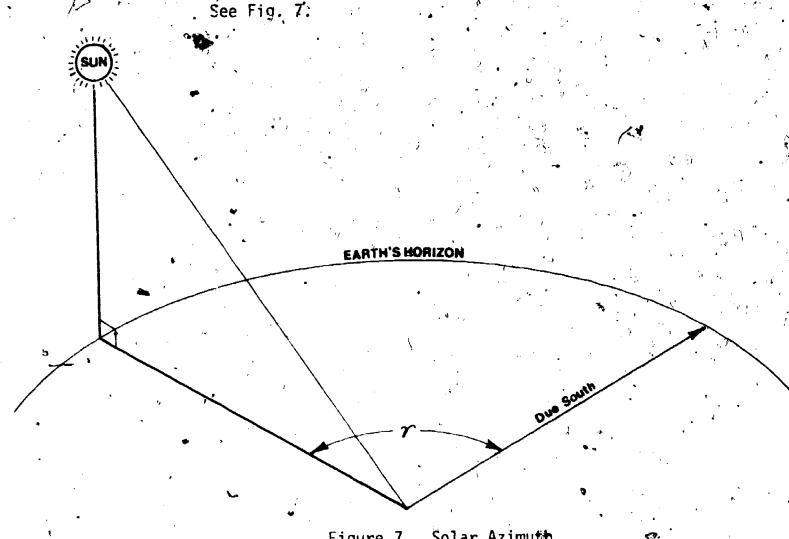


Figure 7. Solar Azimuth

The solar azimuth angle, γ , is given by $\sin \gamma = \frac{\cos \delta \sin H}{\cos \alpha}$ (5)

Example: .Find γ for the example in the previous section where

$$L = 45^{0}N$$

and H = 10 a.m. standard time (H =
$$30^{\circ}$$
) on March 21
where $\delta = 0^{\circ}$ and $\alpha = 37.76^{\circ}$
Then, $\sin \gamma = \frac{\cos 0^{\circ} \sin (+30^{\circ})}{\cos 37.76^{\circ}} = \frac{(1.0) (.5)}{.7905} = .632$
 $\gamma = 39.23^{\circ}$

. Exercise: Find γ for the exercise in Part 4

Part 6. DETERMINE 1

Finally, we have enough information to find \angle i, given by \cos \angle i = $(\cos \alpha \cos \gamma \sin \theta) + (\sin \alpha \cos \theta)$ where, θ = the "tilt" angle of the solar collector (i.e., the angle between the collector and the horizontal). See Fig. 8.

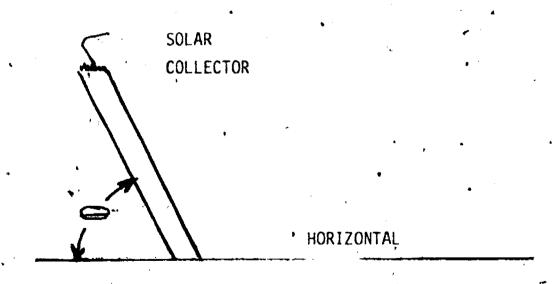


Figure 8. Tilt Angle, θ

Example: Determine the angle of incidence between the sun's rays and a flat plate collector surface tilted at an angle $\theta = 50^{\circ}$ on July 21st

at 11:00 a.m. The collector is located at 40^{0} north latitude.

a.
$$\delta = 23.5 \sin \left[360 \left(\frac{284 + 202}{365} \right) \right] = 20.55^{\circ}$$

b.
$$\sin \alpha = (\sin L \sin \delta) + (\cos L \cos \delta \cos H)$$

= $\sin 40^{\circ} \sin 20.44^{\circ}) + (\cos 40^{\circ} \cos 20.44^{\circ} \cos 15^{\circ})$

$$\alpha = 66.3^{\circ}$$

c.
$$\sin \gamma = \frac{\cos \delta \sin H}{\cos \alpha} = \frac{\cos 20.55^{\circ} \sin 15^{\circ}}{\cos 66.63^{\circ}}$$

$$\sin \gamma = .6108$$
.

$$\gamma = 37.65^{\circ}$$

d.
$$\cos i = (\cos \alpha \cos \gamma \sin \theta) + (\sin \alpha \cos \theta)$$
 (6)
= $(\cos 66.3^{\circ} \cos 37.5^{\circ} \sin 50^{\circ}) + (\sin 66.63^{\circ} \cos 50^{\circ})$
= $.8307$
 $i = 33.83^{\circ}$

Finally, we have found that $i = 33.83^{\circ}$ and $\cos i = .8307$. Therefore, by Eq. 1, one could collect 83.07% of the maximum insolation.

Part 7. SUMMARY. In order to do the calculations required to find the percent of the available energy a flat plate solar collector could absorb, one must give his trigonometry a real workout by finding:

- 1) δ the solar declination for a given day of the year.
- 2) L the latitude of the site.
- 3) H the hour angle for the time of day.
- 4) α the solar altitude
- $\dot{5}$) γ the solar azimuth
- 6) θ the angle of "tilt" of the collector.
- 7) i the angle of incidence.

CONCLUSION.

Return to (Eq. 1),

$$I_b = I_{max} cos i$$

where I_{max} is the maximum amount of solar insolation that falls on a flat surface. Assume idealized conditions, i.e., no reflection off the collector surface, no diffuse beam, clean plates, no loss to smog, fog, shade, etc. Then, $I_{max} = 1000 \text{ watts/meter}^2$ (this is a good round number estimate). Then, as an example, using the sample problem from the last section, where $i = 33.83^{\circ}$.

$$I_b = 1000 \cdot \cos 33.83^{\circ}$$

= 1000 \cdot .8307 = 830.7 watts/meter²

FURTHER INVESTIGATIONS.

Instead of using a fixed plate, can you see the advantage of devising a method for your collector to "track" the sun, or follow it across the sky? Or, instead of using a <u>flat</u> plate collector, think about the characteristics of a parabola, one of the conic sections.

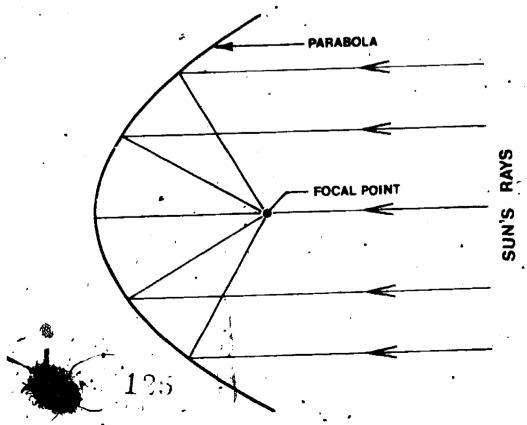


Figure 9. Parabolic Collection



The parallel rays from the sun will focus at a given point, the "focal point" of the parabola. Tracking devices and focusing collectors are now available. New worlds are open to you.

In conclusion, by using basic trigonometry, you can work your way through a typical engineering problem in solar heating. Now you can make up your own data and use the format below to solve your own problems.

Format for Solving Problems

₩.	1			
Given:	A site where latitude L =			
	Time of day (standard time):			
	Day of the year:	•		
•	the tilt angle θ =	. '\		
Find:	1) è =	(Eq. 2)		
•.	2) L =	(given)		
•	3) H =	(Eq. 3)		
	4) (2 =	(Eq. 4)		
•	5) γ =	(Eq. 5)		
	6). i =	(Eq. 6)		
	7) 1 =	(Fa. 1)		

GLOSSARY OF TERMS AND SYMBOLS

<u>Insolation</u> - the rate of solar radiation received by a unit surface in a unit time (units of watts/meter² used here).

Angles - all in degrees:

- Solar Declination (δ) the angle between the equatorial plane of the earth and the sun's direct rays.
- Solar Altitude (α) the angle between the horizon and the sun's direct rays. Solar noon occurs when α is a maximum.
- Solar Azimuth (γ) the angle in the horizontal plane between due south and the vertical projection of the sun's rays.
- Latitude (L) angle between the equatorial plane and the observer.
- Angle of Incidence (i) the angle between the sun's direct rays and the perpendicular to a surface.
- Tilt Angle (θ) the angle between the collector surface and the horizontal Hour Angle (H) $\frac{360^{\circ}}{24 \text{ hr}}$ = 15° /hr from solar noon

1000 watts = 1 kilowatt = 34
$$^{\circ}$$
3 Btu/hr
1 langley = $\frac{1 \text{ calorie}}{\text{cm}^2}$ = 369 Btu/ $^{\circ}$ † 2



Conversion Units:

TERHER'S GUIDE

OBJECTIVE: Enrichment in trigonometry, grades 11-12. At the end of the course in Basic Trigonometry, some students ask, "What is it good for?"

As enrichment for such a student, this unit could be used to illustrate a timely application.

BACKGROUND REQUIRED: An understanding of trigonometry.

TIME REQUIRED: 4-6 hours of individual student's time.

SUGGESTED FOLLOW-UP:

- a) Determine the solar insolation constant.
- b) Design and build a flat plate.
- c) Design and/or build a parabolic collector.
- d) Reading There is a glut of material on solar heating. Many journals, books, texts, etc. are flooding the market; write to SERI in Golden, Colorado for up-to-date bibliographies.

Answers to Exercises:

Section A. a) 98%

Section B. Part 1) a) 20.25⁰

b) 94%

b) 0⁰

c) 87%

c) -22.94⁰

d) 71%

Part 4) $\delta = 20.14^{\circ}$, H

e) 50%

 $\alpha = 57.58^{\circ}$

f) 0%

Part 5) $\gamma = 61.12^{\circ}$

#12. LET THE'SUN DO IT

<u>PURPOSE</u>: To determine the most efficient way to utilize the sun's energy for heating by experimenting with home design, insulation, and color.

INTRODUCTION: One purpose of a shelter is to moderate extremes of temperature. The combination of a fossil fuel shortage and the high cost of living enhances the need for solar heating. Proper planning and construction of a building to make use of the sun can result in energy savings. A technique which employs the conventional parts of a building to collect, store and distribute solar heat without energy-using mechanisms is called "passive" solar heating.

Collecting the sun's energy involves three basic requirements:

- 1. The house itself must be a solar collector, which means the sunlight should be let in when needed.
- 2. The building should be a storehouse for heating. Heavy materials such as stone or concrete accomplish this.
- 3. The house must be a heat trap. Insulation, weatherstripping, shutters, and storm windows help fulfill this requirement.

The optimum shape is an elongated building with an east-west axis. This design should permit the greatest number of windows facing the south and the least facing the north.

MATERIALS: Cardboard, Saran wrap, masking tape, insulation of some type, material for carpet, meterstick, thermometer.

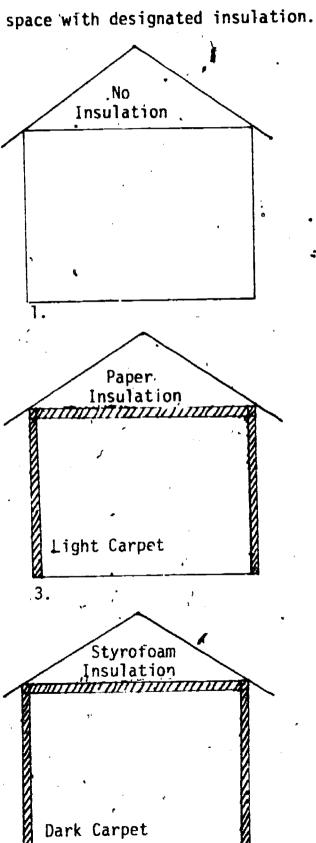
PROCEDURE:

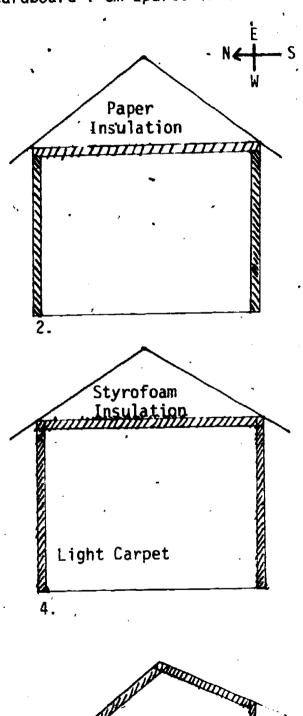
1. Choose one model design from the model page below and construct your model home of cardboard with a.45 m² $\stackrel{<}{-}$ base $\stackrel{<}{-}$.50 m². Windows should cover

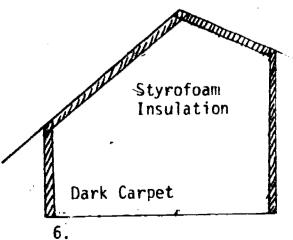


MODELS

These are side views of the model homes. Each "home" should be oriented outside with an east-west axis. To insert insulation, walls and ceiling should be made of two pieces of cardboard 1 cm apart. Fill in the

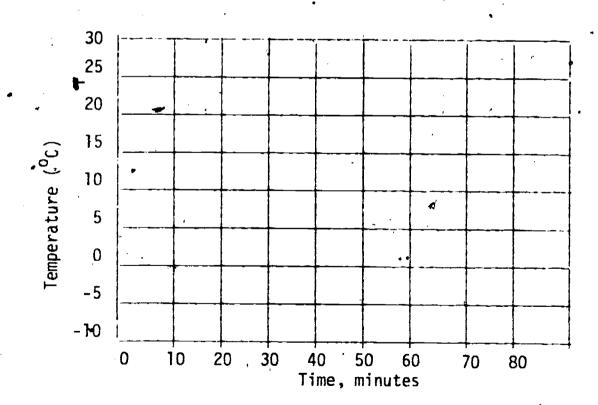






600 cm²: 450 cm² on the south, 150 cm² on the east and west sides. Use Saran wrap for windows. To construct walls use two pieces of cardboard spaced 1 cm apart. Fill in with specified insulation. All inside walls will be a light color. Carpet colors will be specified.

- 2. Compute the volume of the home in m³. (The volume of any regular prism is Ah where A is the area of the base and h is the height.)
- 3. On a cold, clear day place your model home outside for approximately one hour, oriented with an east-west axis.
- 4. With a thermometer, check the temperature outside and record. Then check the temperature inside at 10 minute intervals until no further temperature increase is observed.
- 5. Plot a graph of temperature as a function of time. Complete the data section below.



DATA:	Volume of building	m ³
9	Outside temperature	oc
	Final inside temperature	o
	Specific heat of air = .248 kilocalories/kg OC	
	Mass of $1m^3$ of air = 1.29 kg.	

QUESTIONS, PROBLEMS, AND CONCLUSIONS:

- 1. Define: calorie, kilotalorie, Btu.
- 2. Using the formula, Q = mc (T_{Inside}. T_{Outside}), where Q is heat in kilocalories, m is the mass of air contained in the model in kilograms, c is the specific heat of air in kcals per kilogram per degree and T is temperature in degrees Celsius, find the amount of heat-stored in your passive solar home in kilocalories.
- 3. One Btu = .252 kcal. Express your answer to #2 in Btu's.
- 4. Compare the amount of heat stored in your model solar home with the models made by your classmates.
- 5. What conclusions can you make regarding the passive heating capabilities of the various models tested?

TEACHER'S GUIDE

OBJECTIVES: At the completion of this experiment the students should be able to:

- 1. Calculate the amount of heat stored in a given volume of air.
- 2. Understand the meaning of passive solar energy.
- 3. Utilize the reflective and absorptive characteristics of interior colors.

BACKGROUND:

"Now in houses with a south aspect, the sun's rays penetrate into the porticoes in winter, but in summer the path of the sun is right over our heads and above the roof, so that there is shade. If then, this is the best arrangement, we should build the south side loftier to get the winter sun and the north side lower to keep out the cold winds." Socrates (469-399 B.C.) as quoted by Xenophon in Memorabilia.

Passive solar heating is defined as the technique which uses the conventional parts of a building to collect, store, and distribute solar heat without energy-using pumps or fans. Generally, it relies on the south wall to collect and store heat. Other innovators have used 55-gallon steel drums filled with water to construct the south wall. The water heats by day and radiates inside the home at night. A home with proper design and construct tion can result in energy savings.

As can be learned from the quotation, solar heating is not a twentieth century invention. According to Roman regulations the axis of a military camp had to be within 30° of true south. Teotihuacan, a Meso-American city, was laid out on a grid 15° west of south. Pueblo Bonito was built by the Indians in Chaco Canyon, New Mexico, between 919 to 1180 A.D. The semicircular structure with a 520 ft. diameter and four stories high housed 1200 inhabitants. The plan of the structure was based on the position of



the sun in the summer and winter solstices. Surfaces were exposed to more radiation in the winter than in the summer. Wall and roof construction was varied in thickness and composition to store the sun's heat and to permit the proper time lag of the day's heating effect to radiate into the interior at night. The Acoma Pueblo near Albuquerque was built on a similar plan. Early New England houses had masonry-filled walls and compact rayouts to minimize heat losses during frigid winter months. Many colonial American homes had high walls and windows on the south while a low sloping roof faced the north.

Since an abundant supply of cheap fossil fuels became available during the present century, the sun was ignored in designing buildings. Now high costs and the shortage of fossil fuels have reversed the situation.

TIME REQUIRED: ,5-10 days

MATERIALS: Each group will need a sufficient amount of cardboard (packing boxes will suffice for a source), masking tape, Saran wrap, material for walls and floors, thermometer, meterstick.

SUGGESTIONS: This experiment can be a group project with groups of three or more as the instructor deems advisable. One model can be built with a standard inverted v-shaped roof and no insulation to serve as a basis for comparison. The other homes can be varied according to types of insulation in wall and ceiling, color of carpets and chosen design. Model designs have been developed so that subtraction of results from two models will yield a difference attributable to one variable.

SAMPLE PROBLEM: Assume a model has a volume of .2m3. Find Q (amount of heat).

c = Specific heat of air = .248 kcal/kg \cdot °C

T_{Inside} = final inside temperature = 20 °C

Toutside = ambient air temperature = 0 °C

Mass of 1 m^3 air = 1.20 kg

Mass of air in kg contained by model = $.2m^3 \times \frac{1.29 \text{ kg}}{m^3} = .258 \text{ kg}$

Then Q = mc ($T_{Inside} - T_{Outside}$) = .258 kg x $\frac{.248 \text{ kcal}}{\text{kg} \cdot \text{ }^{\circ}\text{C}}$ x 20°C = 1.28 kcal

SUGGESTED FOLLOW-UP ACTIVITIES:

- 1. Let any of the models heat up to its maximum capacity. Open and close a door at minute intervals for twenty minutes. Graph the temperature as a function of time. Conclusions?
- 2. Add a "fireplace" (do not ignite) to one of the homes. Observe the temperature inside the model as the "flue" is opened and closed. Conclusions?
- 3. Vary orientation of the same houses.
- 4. Calculate mass of air at local conditions using gas laws.

REFERENCES:

The Solar HomeBook, by Bruce Anderson and Michael Riordan, Cheshire Books, Harrisville, New Hampshire, 1976.

Homeowner's Guide to Solar Heating and Cooling, by William M. Foster, Tab Books, Blue-Ridge Summit, Pennsylvania, 17214, 1977.

Designing and Building a Solar House, by Donald Watson, Garden Way Publishing, Charlotte, Vermont, 15445, 1977.

Physics: Energy in the Environment, by Alvin Saperstein, Little, Brown, and Company, Boston, 1975.

Direct Use of the Sun's Energy, by Farrington Daniels, New Haven, Yale University Press, 1974.

#13. CONSTRUCTING A MODEL GREENHOUSE

PURPOSE: The purpose of this activity is to provide students with and experience of constructing a model solar greenhouse from the materials provided and to determine quantitatively the total heat gain, in Btu's, for the solar greenhouse.

INTRODUCTION: This activity is designed especially for the student who likes to eat. Realizing that our present energy resources are being depleted rapidly, we must seek other useful forms of energy for long term use and storage. This is applicable to food production.

What is a solar greenhouse? The term solar greenhouse generally refers to greenhouses whose heating and light requirements are largely provided by the sun. All greenhouses receive most of their light from the sun and recently some have been designed to use the sun for heating as well. Solar greenhouses collect and store solar energy for heating and are well insulated so that the heat may be used at night and during cloudy periods.

MATERIALS (per group):

Cardboard box 33" x 20" x 20". Box size may vary

Styrofoam - enough to line all interior surfaces of greenhouse

Aluminum cans, 16 - standard pop cans

High temperature, non-toxic, flat black paint - 1 large spray can

Saran wrap

One-hole rubber stopper

Thermometer - 300°F maximum (oven thermometer)

G1 ue

Masking tape

Knife or razor blade

Ruler

Pencil

Data sheet

Graph paper

PROCEDURE:

Day 1. Activity 1

- Construct your solar greenhouse using the cardboard box. (Refer to Fig. 1 for plan.)
- 2. Insert thermometer into one hole rubber stopper and then insert stopper centered into side of greenhouse.
 - 3. Cover the opening with one sheet of Saran wrap.

Day 2. Activity 2

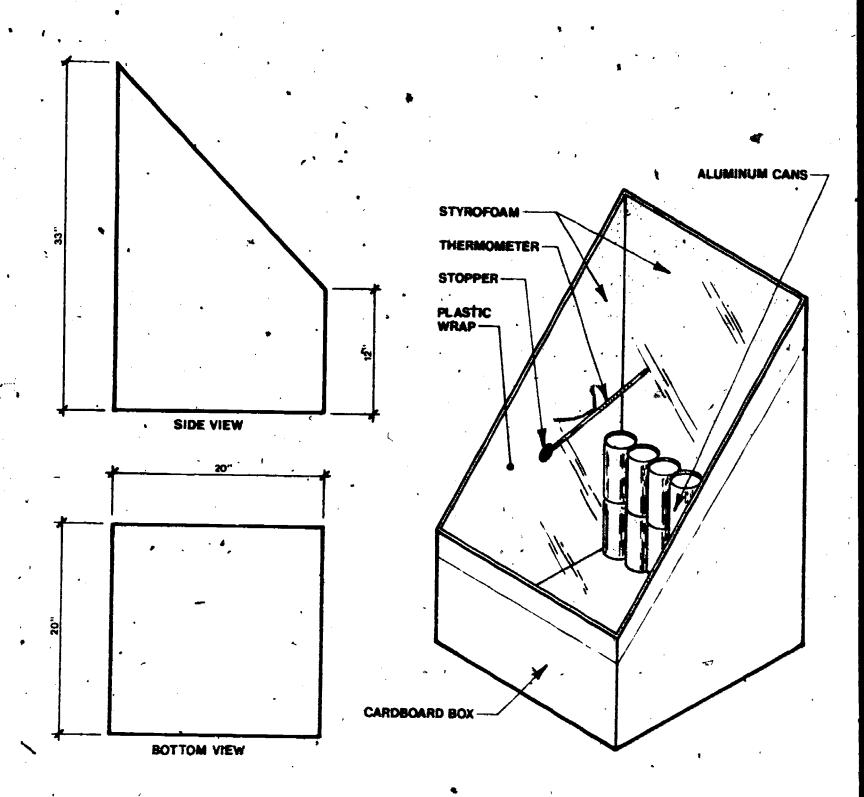
Place your greenhouse on the south side of the school building with the Saran wrap surface facing south.

- 1. Record the ambient air temperature on a data sheet and record the "greenhouse" temperature every two minutes on your data sheet.
- 2. Note changes of cloud cover on data sheet.
- 3. Graph the temperature as a function of time for outside and inside.

Review Questions for Activity 2.

- 1. How fast did the greenhouse heat up to its maximum temperature?
- 2. What maximum temperature was reached?

Fig. 1. Diagram of a Model Solar Greenhouse *



3. What effects do weather conditions have on a solar greenhouse?

Day 3. Activity 3

- 1. Remove Saran wrap and thermometer.
- Using ruler, knife and glue, insulate the interior walls of the greenhouse with styrofoam.
- 3. Paint the interior of the greenhouse with flat black, non-toxic paint.
- 4. Paint all sixteen aluminum cans with flat black.

 Allow everything to dry until next day.

Day 4. Activity 4

- Fill aluminum cans with water and cover holes with masking tape, Record weight of water. Fill each can with same amount.
- 2. With masking tape connect aluminum cans side-by-side in four groups of four.
- 3. Tape each set of cans on top of one another, and attach with tape or staples to the back of the greenhouse.
- 4. Touch-up inside of greenhouse with flat black paint.
- 5. Insert thermometer into side wall by pushing through styrofoam.
- Stretch one layer of Sarap wrap over opening.
- 7. Place greenhouse on south side of school building exposed to the sun. Record ambient air temperature on data sheet, and record inside temperatures at two minute intervals for fifteen minutes.
- 8., Graph temperature as a function of time for the next class meeting.



Day 5. Activity 5

- M. Place greenhouse outside and record temperature for litteen minutes. Remember to record ambient air temperature first.
- 2. Take greenhouse inside and record the temperature for fifteen minutes. Continue to observe temperature change if change occurs slowly.
- 3. Construct a graph of temperature as a function of time for your model from data sheets. Compare graph with uninsulated greenhouse.

Day 6. Activity 6

- 1. Using data provided by your teacher, calculate the heat gain for Activities 2, 4 and 5 in btu's/hr on the "collector" surface.
- 2. What effect did the insulation, black paint and cans of water have on temperature change?
- 3. What changes can be made in construction of your model to allow it to collect more heat?
- 4. How might a simple solar greenhouse be constructed to provide heat for a home.

TEACHER'S GUIDE

OBJECTIVE: The student will construct a model solar greenhouse, determine quantitatively the number of Btu's "collected" by the solar collector surface and the total heat gain, in Btu's. Optional activities are included.

BACKGROUND: The Btu, or British Thermal Unit, is a commonly used unit of heat. It is defined as the amount of heat required to raise the temperature of one pound of water one degree Fahrenheit; specifically, from 59°F to 60°F. Heat losses and furnace capacities are described in Btu's per hour. For example, one gallon of oil burned at 70% efficiency will deliver about 100,000 Btu's (one kilowatt-hour is the equivalent of 3,412 Btu's). In comparison, the maximum intensity of sunlight at 40°N latitude is 306 Btu's per square foot per hour.

The student will determine the principles upon which a solar greenhouse functions. Model construction, accurate calculations and measurements and recording of data, as well as visual observations, are stressed.

The term solar greenhouse generally refers to greenhouses whose heating and light requirements are provided by the sun. Most solar greenhouses collect and store solar energy for heating and are insulated so that the heat collected can be used at night and during cloudy days. Solar greenhouses can be used to collect and store solar energy in many ways. The design of solar collectors and storage systems depends upon such factors as climate, economics and local weather conditions.

The two main types of solar systems are active and passive. An active system collects heat energy at one point, transports it, stores it at another, and delivers it for use somewhere else. Conventional energy sources are needed to move the collected solar energy.



In a passive system water containers, water pools, rock walls and other materials with high thermal mass collect, store and deliver the heat without circulating fans and pumps. If at any point in the system, additional energy must be used for storage or delivery, then that portion of the system has become active.

Solar greenhouses can be built independently or attached to an existing structure. A well-designed solar greenhouse can deliver more heat to a house during the day than the house loses at night.

Another advantage of a solar greenhouse is the extended, year-round growing season at a much lower operating cost than the conventional units. They are relatively inexpensive and easy to build.

On the average, there is enough solar energy to heat a greenhouse in the winter in most of the United States and Canada. During the coldest time of the year, late January on a clear day, the intensity of direct solar radiation is about 290 Btu's per square foot per hour at noon at $40^{\circ}N$ latitude. This energy could heat the air from 40° to $70^{\circ}F$ in one hour in a properly oriented 12' x 16' greenhouse.

Solar energy enters the greenhouse as shortwave radiation. When it's absorbed by inside surfaces, infra-red waves (heat) are radiated. These longer wavelengths cannot escape back Anto the atmosphere (the "Greenhouse Effect").

The light requirements must be balanced with heat requirements in a solar greenhouse. If the greenhouse is to function properly, it will require some type of heat and storage capability. The activities are designed to illustrate these basic solar greenhouse principles.



TIME REQUIRED: Six class hours.

<u>SUGGESTIONS</u>: Some students may wish to gather data on days when there is partial cloud cover. This would provide comparative data.

Day 1. Activity 1:

- 1. Students will construct their model solar greenhouses.
- 2. Time remaining in class period may be used to discuss various types of greenhouse construction and cost.

Day 2. Activity 2:

- 1. Complete Activity 1. Remind students to save their data sheets.
- 2. Students complete their graphs as homework, using their data.
- 3. Students should answer review questions at end of Activity 2.

Day 3. Activity 3:

- 1. Students will make basic changes in greenhouse construction:
 insulating and painting interior and aluminum cans with non-toxic,
 flat black paint.
- 2. If time permits, discuss and compare student graphs from previous day (Activity 2).

Day 4. Activity 4:

- 1. Complete modifications of model.
- 2. Set-up model and record temperature on data sheet for fifteen minutes.

Day 5. Activity 5:

- 1. Discuss graphs from Activity 4.
- 2. Students may collect more data for comparison.
- 3. Data should be graphed for following class meeting.

Day 6. Activity 6:

- 1. Explain to students the method used to calculate total insolation on a surface in Btu/hr. Refer to Sample Problem (A) below.
- 2. Discuss implications of solar energy based on experiences with the model solar greenhouses.

OPTIONAL ACTIVITIES:

- 1. Students who have mastered trigonometry may wish to calculate total clear-sky insolation with reference to angle of collector surface. Refer to Sample Problem (B) below. (Reflective radiation need not be considered.)
- 2. Determine total number of Btu's collected in the solar greenhouse. Refer to Sample Problem (C) below.
- 3. Thermocouple may be substituted for thermometer. Insert it in one aluminum can and attach to voltmeter. (Refer to "Calibrating A Thermocouple" in this manual for set-up instructions.

GENERAL INFORMATION AND EXPLANATION OF PROBLEMS:

Sample Problem (A)

How many Btu's of solar heat impinge on the surface (2.5 ft^2) of a south-facing collector tilted 40° from the horizontal at 10:00 a.m. on January 21 (latitude, 40° N)?



From Table 2, the solar radiation is 237 Btu's/ft 2 . Therefore, the radiation striking 2.5 ft 2 is:

2.5 ft² x 237
$$\frac{Btu's}{ft^2}$$
 = 592.5 Btu's

Sample Problem (B)

Determine the total clear-sky solar radiation falling on a flat-plate collector tilted 50° to the horizontal on Jan. 21 in Boulder, Colorado at noon. See drawing. $10^{\circ} = \text{Angle of incidence, i}$ $30^{\circ} = \text{Solar altitude, } \alpha$

The amount of beam radiation impinging on a surface on the earth is a function of surface-to-sun geometry. The beam component of solar radiation (I_b) effectively received by a collector is given by

$$I_b = I_{bn} \cdot \cos i$$
 (1)

where I_{bn} is the beam radiation falling on a surface perpendicular to the sun's rays, and 'i' is the angle of incidence measured between the sun's rays and the normal to the collector surface. When the angle of incidence 'i' is equal to 0, $\cos(i) = 1$, and the beam radiation available to the collector (I_{b}) is the maximum value, I_{bn} .

The diffuse radiation component, (I_d) , received by the collector is determined by the equation

$$I_d = C \cdot I_{bn} \cdot Fa$$

where C is the dimensionless value given in the last column of Table 1 presented below; and Fa is the angle factor from the surface to the sky:

$$Fa = (1 + \cos(\theta)/2)$$

where θ is the collector tilt angle measured from the horizontal.

Table 1 Extraterrestrial Solar Radiation Intensity (Btuh/ft²) and Related Data for Twenty-First Day of Each Month, Base Year 1964

Btul	t/ft ²	Equation of Time, min.	Decli- nation deg	A Btuh/ft ²	B (Dimensionless	C . Ratios)
Jan Feb Mar Apr	442.7 439.1 432.5 425.3	-11.2 -13.9 - 7.5 + 1.1	-20.0 -10.8 0.0 +11.6	390 385 376 360	0.142 0.144 0.156 0.180	0.058 0.060 0.071 0.097
'May	418.9	+ 3.3	+20.0	350	0.196	0.121
June	415.5	, - 1.4	+23.45	345	0.205	0.134
July Aug	415.9	- 6.2 - 2.4	+20.6	344 351	0.207	0.136 0.122
Sep Oct Nov Dec	426.5 433.6 440.2 443.6	+ 7.5 +15.4 +13.8 + 1.6	0.0 -10.5 -19.8 -23.45	365 378 387 391	0.177 0.160 0.149 0.142	0.092 0.073 0.063 0.057

To convert Btuh /ft² to W/m², multiply by 3.1525; example: 442.7 Btuh/ft² x 3.1525=1395.6 W/m. (The Btu units in this chapter are thermochemical.) To convert deg to rad (SI unit), multiply deg by 0.01745.

In addition to the beam and diffuse radiation components, there may exist a further gain in radiated energy due to reflection from the immediate surroundings. In the Boulder, Colorado area about 70% to 90% of the total radiation collected is generally in the form of beam radiation. However, in northern climates where snow covers the ground for most of the winter, the diffuse and reflected radiation can account for half of the total energy collected.

$$I_{t} = I_{b} + I_{d} + I_{r}$$
 (2)

where I_{+} = total solar radiation

 $I_b = direct beam radiation$

 $I_d = diffuse radiation$

 $I_r = reflective radiation$

To calculate the beam radiation, refer to Table 2 "Normal" column.

Now refer to Table 2, "Normal" column. (No angle adjustment has been made). Note that Btuh = Btu's per hour. I_{bn} at normal at 12 noon = 294. Source of i is shown in figure below.

$$I_b = I_{bn} \cdot \cos i$$

= $\frac{(294 \text{ Btu})(\cos 10^0)}{\text{ft}^2}$
= $(294) \cdot (.984)$
 $I_b = 289.5 \text{ Btuh/ft}^2$

Table 2.*

Date	Solar	Time		Position	Btu/S	g. Ft.	Total	Insola	tion e Surfac	n Surf e Anal	aces e with	Hor
,	AM	PM	Alt.	Azm.	Normal	Horiz.	1	40	50	60	- 90	
Jan 21_	8 9 10 11	4 3 2 1	8.1 16.8 23.8 28.4 30.0 Surface	55.3 44.0 30.9 16.0 0.0	142 239 274 289 294	28 83 127 154, 164 948	65 155 218 257 270	74 171 237 277 291 1810	81 182 249 290 303 1906	85 187 254 293 306 1944	'84 171 223 253 263 263	

^{*} See Appendix B-5

Calculate diffuse radiation:

$$I_d = CI_{bn}$$
. Fa

Fa = $\frac{1 + .\cos 50}{2}$

Fa = .821

C = .058

 $I_d = (.058)(294)(.821)$

Idy= 13.99 Btuh

Calculate total solar radiation:

$$I_t = I_b + I_d$$

$$I_t = 289.5 + 14$$

$$I_t = 303.5 \text{ Btu}$$

Note that this figure is found under 50° surface angle at noon as total insolation

Sample Problem (C)



Equation (3) is used to determine total heat, H.

$$H = Btu of H_2O + Btu of air$$
 (3)

 $H = (mass)(specific heat)(\Delta temp) + (volume)(density)(specific heat)(\Delta temp)$ or $H = (lbs)(Btu/lb^OF)(OF) + (ft^3)(lb/ft^3)(Btu/lb^OF)(OF)$

Follow-Up Activities:

1. Various paint treatments may be used on the inside of the greenhouse to determine heat gain as a function of color.

- 2. Some students may wish to design a solar greenhouse on a larger scale and delve into other investigations; e.g., construction types, design, orientation, climatic effects, solar geometry.
- 3. The model greenhouse may be used to heat water. Pertinent data can be collected to illustrate the principle of a solar water heater.
- 4. A large scale solar greenhouse might be constructed from 3/8" plywood as an industrial education project.
- 5. The basic model design could be modified to make a solar oven,
- 6. The basic model design/could be modified to include solar distillation.
- 7. Determine R-factors and U-factors for insulation used in the model.

 Refer to Homeowner's Energy Guide (in bibliography); also ASHRAE,

 American Society of Heating, Refrigerating and Air Conditioning.

 1977 Fundamentals Handbook, Chapter 26, Table I.

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Nearing, Helen, <u>Our Sun Heated Greenhouse</u>, Garden Way Publishing, 1977.

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#14. SOLAR HEAT STORAGE AND HEAT OF PHASE CHANGE

PURPOSE: You will learn how to calculate the heat of fusion (heat of solidification) of a pure substance. With this information and class data, you will be able to choose which substance would be best suited for use in a solar heat storage unit.

INTRODUCTION: Imagine no available energy resources to heat your home!

Solar energy technology may very well be one of the best solutions to the future energy crunch. As you know, heat storage from a solar collector must be considered if one intends to stay warm at night or when the sun is obscured by clouds.

Heat storage has been effectively accomplished with large rock bins and water tanks. But one problem is the space they must occupy. Finding materials that release more heat per unit of storage volume becomes a very important consideration.

It has been found that as materials change phase they absorb or emit much more heat per gram than just one phase absorbs or emits as it is heated or codled. The heat absorbed by a melting substance at its melting point is called the heat of fusion; the heat released when a substance solidifies at its freezing point is called the heat of solidification. Can this "latent heat" be used to design a compact solar heat storage unit? You'll compare the heats of fusion of some pure substances and choose which might be best for your storage bin.

MATERIALS:

1 thermometer (0°C to 110°C)

1 small burner -

1 styrofoam cup, 1 pt

Test tubes (18 x 150 mm)

Test°tub∈ holder

Îce

Paradichlorobenzene $(C_6H_4Cl_2)$; melting point 53°C.

Ring stand, clamps for holding thermometer and test tube.

Plastic caps -

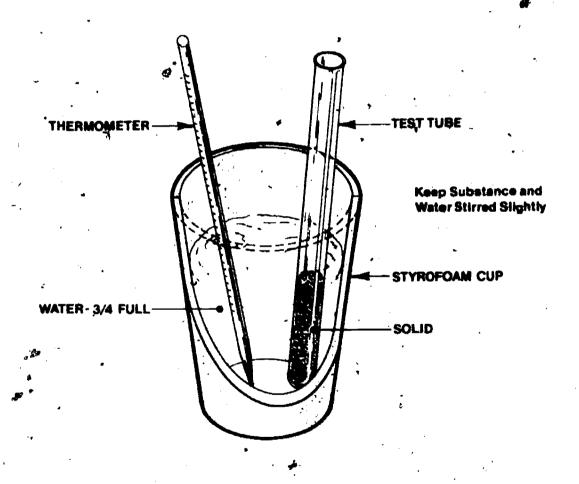


Fig. 1. Apparatus

PROCEDURE: Part I - Heat of Fusion

At the conclusion of this part of the experiment, Data Chart I should be completed.

Data Chart I

1. Mass of test tube	g
2. Mass of test tube + substance 15:	g
3. Mass of substance	· g



4.	Volume of water used	·	_ ^{mi}
<i>.</i> 5.	Initial temperature of water bath		_ °c .
6.	Final temperature of the water bath	·	o _C
	Λ Τ(⁰ C)-		o _C
8.	Heat in calories lost by the water to the substance	5	cal·
· 9.	Heat of fusion in calories per gram of		
	substance		cal/g
10.	Heat of fusion in kcal per mole	•	kcal/mole

- 1. Find the mass of a clean, dry test tube to the nearest .01 g. Fill it approximately half-full with one of the substances provided by your instructor, and mass it again (be sure to write down the name of your substance).
- 2. Heat enough water to fill your styrofoam cup calorimeter three-quarters full (know the volume exactly). The water temperature should be 25° C to 30° C above the melting point of the substance. Prepare calorimeter by placing one styrofoam cup in another cup. Cut holes for thermometer and test tube in cap.
- 3. Place the test tube and your thermometer into the calorimeter. See Fig. 1. Record the temperature of the water every 30 seconds. Note when the substance first appears to be melting and when it has <u>all</u> been melted.

 After all of the substance has melted and the temperature remains fairly constant, stop recording.

CALCULATIONS: As you have noted, the temperature of the water bath dropped. We must make the assumption that the heat energy associated with the temperature change (Δ T) was gained by the substance in the test tube and not lost to the surroundings; i.e., the heat gained by substance equals the heat lost by the water bath.

To calculate the heat lost by the water bath you must know the mass of water, ΔT (^{O}C), and the specific heat of water (1 calorie per gram per ^{O}C). The equation looks like this:

Heat lost by water = Heat of fusion = (mass g) (Λ T, O C) (specific heat, $\frac{1}{g}$ cal) (1) go Eq. (1) you can see that the heat will be expressed in calories.

- Calculate the number of calories required to melt the substance.
- 2. Calculate the number of calories required to melt one gram of the substance (divide the total number of calories by the number of grams of substance).
- 3. Calculate the number of kcal required to melt one mole of your substance. Hint:

cal/g x $\frac{1 \text{ kcal}}{1000 \text{ cal}}$ x g/mole = kcal/mole

where g/mole = the molecular weight

PROCEDURE: Part 2. Heat of Solidification

At the conclusion of this part of the experiment, Data Chart Warshould be completed.

Data Chart II.

1.	Mass of test tube	g
2.	Mass of test tube and substance	g
3.	Mass of substance	g
4.	Volume of water used	m1
5.	Initial temperature of water bath	O _C
6.	Final temperature of water bath	· OC
7.	$\Delta T (^{O}C)$	C

8.	Heat in calories gained by the water bath		
	from the substance .		_ cal
9.	Heat of solidification in calories per gram	,	
	of substance		_ cal/g
10.	Heat of solidification in kcal per mole		_kcal/mole

- 1. Mass a clean, dry test tube to the nearest .01 g. Then fill it one-half full with one of the substances provided by your instructor, and mass it again.
- 2. Obtain a known volume of water at a temperature which is approximately 20° to 30°C below the melting temperature of your substance and fill your styrofoam cup calorimeter.
- 3. Heat the solid until it is entirely melted. (Do not overheat). As soon as the crystal formation is evident, place the test tube in the calorimeter and record the temperature of the water bath every 30 seconds. When all of the substance has solidified and there is no further temperature change, stop recording.

<u>CALCULATIONS</u>: As the substance solidifies, it gives off heat to the water bath, causing its temperature to rise. Assuming no heat loss to the surroundings, the heat absorbed by water equals the heat released by the substance. Again the calculation uses the mass of substance, ΔT OC and the specific heat.

Heat gained by water = Heat of solidification = (mass, g)(ΔT , ^{O}C) (specific heat, $(\frac{1 \text{ cal}}{g \text{ C}^{O}})$

- 1. Calculate the number of calories released by the substance as it solidifies.
- 2. Calculate the number of calories released by one gram of your substance



as it solidified.

3. Calculate the number of kcal released by 1 mole of your substance. cal/g x $\frac{1 \text{ kcal}}{1000 \text{ cal}}$ x g/mole = kcal/mole

DISCUSSION QUESTIONS:

- 1. How do your two values (heat of fusion, heat of solidification) compare?
- 2. Is there any heat gained while the substance is only in one phase?
 (Hint: Think of specific heat capacity.)
- 3. What would be the effect on your results if the calorimeter were better insulated? How could you find out?
- 4. Find the specific heat of fusion of other substances which would be suitable for use in solar storage. Discuss the possibilities of using these substances.
- 5. Sketch a very simple diagram of the flow of air from solar collectors on a roof through a heat storage unit and then to a home's living space. Could one use this system to cool a house in the summer?

TEACHER'S GUIDE

OBJECTIVES: Students will understand: 1) that there is a definite heat effect associated with the melting and freezing processes; 2) the terms: heat, temperature, calorimetry, heat of fusion and heat of solidification; 3) various aspects of solar energy storage technology.

BACKGROUND: Energy storage can be accomplished in several ways. Passive thermal storage is the storing of the sun's heat in a structure (e.g., a wall of your house). Later, the wall will radiate heat into the house. Chemical storage utilizes a substance like hydrogen in a fuel cell. Fly wheels also store energy. Chemical batteries are a common means of electrical storage, but they are expensive.

energy. The latent heat of a eutectic is energy stored or released in a phase change. Most common substances used for storage have specific heats of one or less. A given eutectic with a high heat of fusion requires less mass for the same amount of heat storage capability.

A problem is finding materials that change phase in the temperature range of solar collector operation ($100-300^{\circ}F$). One substance that has been studied is Glauber's Salt ($Na_2SO_4 \cdot 10H_2O$). Its disadvantage is that it often deteriorates after several cycles.

TIME REQUIRED: 3-4 class periods.

SUGGESTIONS:

1. This activity is designed for an 11th grade chemistry class to develop the



topics of calorimetry, heat of fusion, and initiate interest in solar storage technology. It is hoped that post-lab discussion will include the need for alternate energy technologies.

2. To be conversant with engineers, one must use Btu's (British Thermal Units). One Btu is the amount of energy required to heat one pound of water one degree Fahrenheit. The appropriate conversion factor is:

1 Btu 252 calories

3. Other substances to use:

Substance	Melting point, OC	Molecular weight	Heat of fusion, cal/g	
1,4-Butanediol	15-19	90	ر المسلم المسلم المسلم المسلم. المسلم المسلم	
n-Octadecane	27-29	255		
Paraffin •	50-58	500	35.1	
Paradichlorobenzene	53	147	29.07	
Diphenylamine	53	163	≈ 30.4	
Naphthalene {	79-80	128	35.06	
Naphthol "	94	144	38.94	

4. Be certain that students know the difference between temperature and heat. Temperature is merely an arbitrary scale of the flow of heat from hot to cold, whereas heat is the amount of energy the system may contain measured in units like calories. Students should realize that a substance may release or absorb heat energy at its freezing point without a change in temperature.

SUGGESTED FOLLOW-UP ACTIVITY: Determine the dollar savings that a eutectic heat storage unit could save. Contact your local utility company about



prices of heating. Discuss criteria for selecting practical storage media.

ANSWERS TO QUESTIONS:

- 1. Nearly the same.
- 2. Yes

 $H = (\Delta T)$ (mass) (specific heatof eutectic) = calories

- 3. Insulating the calorimeter would reduce the heat lost to the surroundings. This would increase values obtained for the heat of fusion.
- 5. a. See Fig. 2.
 - b. Yes

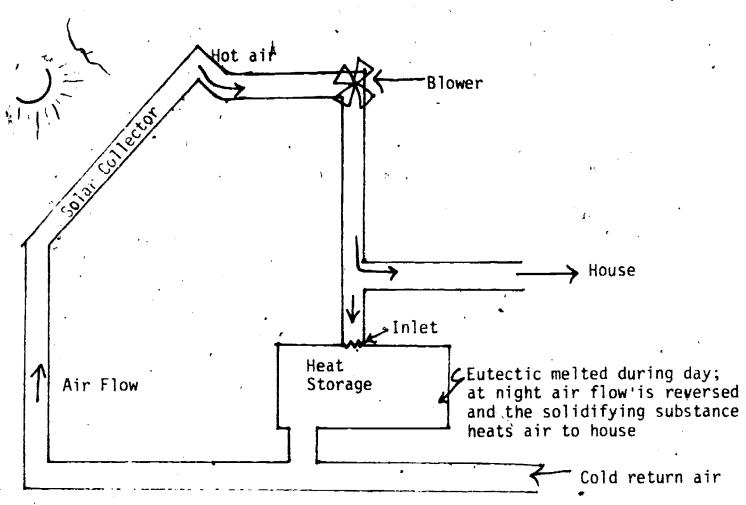


Fig. 2. 1

REFERENCES: *

Krieth, Frank and Jan F. Kreider, Solar Heating and Cooling, McGraw Hill, 1977.

Telkes, M., "Solar Heat Storage", American Society of Mechanical Engineering, 64-WA/SO1-9, 1964.

Telkes, "Storage of Solar Heating and Cooling", ASHRAE Journal, Sept. 1974, pp. 38-44.

Telkes, <u>Application of Solar Energy for Heating and Cooling</u>, ASHRAE, GRP 170.

#15. SOLAR ENERGY STORAGE IN GRAVEL

PURPOSE: You will measure the heat storage capacity and the efficiency of a gravel bed system.

INTRODUCTION: Solar energy heats our homes passively now through southfacing windows. We can augment our home heating with a solar collector,
storing excess heat for nights and cloudy weather. Solar heat can be
stored in water for later transfer to room air, or it can be stored in
gravel or broken brick.

In this experiment a hair dryer will be used to simulate an air-heating collector and a gravel storage bed will be used, but the principles can be applied to any collector/storage system. The size chosen for the gravel bed, the collector size, and the heat loss of your component, will help determine the highest bed temperature for a given amount of solar radiation. Remember, in transferring the heat from source to storage to house, use as few steps as possible, keeping the path of heat flow short, and insulate heavily to reduce heat losses.

MATERIALS:

Three thermometers

Plywood or sheeting material

Hair dryer, 1000-watts

Gravel (1.5 inches)

10 x 10 graph paper



Small electric fan

G1ue

Nails

Duct tape

2-inch duct

Screen

PROCEDURE:

- 1. Construct a container of the appropriate volume (1 ft³) to hold the rocks, using sheeting material or plywood. Seal all edges with 2" duct tape. The top should fit tightly to limit heat losses.
 - 2. Make an inlet at the top and an outlet on the opposite side near the bottom, so that the air flow contacts the maximum amount of rock.
 - 3. Determine the mass of rock you are using. Weighing can be avoided by remembering that the density of rock is about $126 \, lbs/ft^3$.

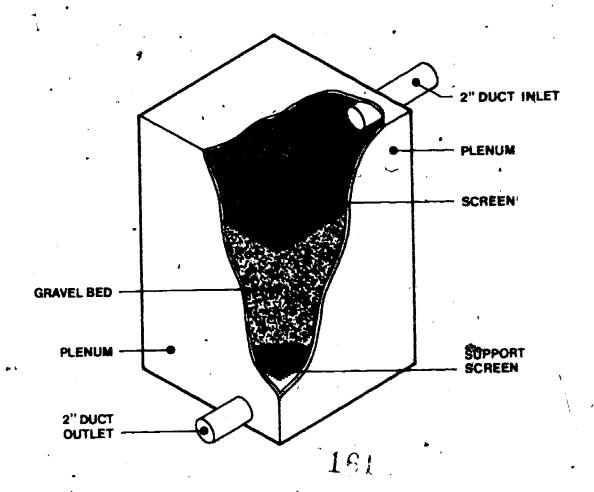


Figure 1.

- 4. Mount the hair dryer in the inlet.
- 5. Insert a thermometer in the inlet and a thermometer in the outlet ducts of your storage bin.
- 6. Insulate the storage bin to simulate a basement location.
- 7. Fill the storage component with gravel and seal top. Turn on hair dryer.
- 8. Measure and record temperatures of the air in the inlet and outlet ducts at 5 minute intervals for thirty minutes.
- 9. Shut off hair dryer.
- 10. If possible, temperature measurements should continue to determine how long it takes for the stored heat in the rock to be released.
- 11. Plot the temperature as a function of the time on 10×10 graph paper.
- 12. Compute the total amount of heat stored in the storage component using $\Delta H = m \times s \times \Delta T$

where m is the mass of rock, s is the specific heat, 0.205 cal/g· C^0 (0.205 Btu/1b· F^0), and Δ T is the difference between the initial temperature and the maximum temperature of the rock storage bin.

- 13. Determine the volume of your storage bin $(V = \ell \times w \times h)$, and the rock density (D = m/v).
- 14. Determine the air flow rate, R = volume of air/time

QUESTIONS:

- 1. How could we use this stored energy?' What was the cost?
- 2. What size solar system would you need to heat your home? Your classroom?
- 3. What other storage material could be used?

TEACHER'S GUIDE

OBJECTIVE: The student will be able to measure the heat storage capacity and efficiency of a gravel bed.

TIME REQUIRED: Five or six 45-minute class periods. This experiment may be done individually or in small groups.

MATERIALS: Three thermometers, plywood or sheeting material, insulating material, gravel (1/2 - 1"), 10 x 10 graph paper, glue, nails, duct tape, "2" metal duct; 1000-watt hand-held hair dryer.

SUGGESTED FOLLOW-UP ACTIVITIES:

- 1. Visit a solar home, measure and record Btu's produced, Btu's stored, and Btu's used.
- 2. Compare two or more solar storage units and their cost effectiveness.
- 3. Design and build an air-loop, rock storage, convective solar heater.

SUGGESTED CAREER EDUCATION ACTIVITIES: Given the opportunity to identify three occupations of interest, the student will list the names, addresses and phone numbers of local persons and/or groups with whom the student could discuss:

employment opportunities
training or skills required
hiring procedures

Given the opportunity to work in small groups, the student will identify the business opportunities in the solar industry.

REFERENCES:

The Solar Home Book, Bruce Anderson, M. Riordan, Cheshire Books, 1976.

Build-It Book of Solar Heating Projects, W.F. Foster, Tab Books, 1976.

Homeowner's Guide to Solar Heating and Cooling, W.M. Foster, Tab Books, 1976.

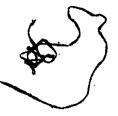
Sun Earth, R.L. Crowther, A.B. Wirchfeld Press, Inc., 1976.

APPENDIX A

Appendix A-1 Selected Constants

CONSTANTS :

Acceleration of gravity	9	980.7 m/séc²
		32.17 ft/sec ²
Atomic mass unit Electron rest mass	amu (M _e	1.660531 x 10 ⁻²⁷ kg 9.109558 x 10 ⁻³¹ kg
Proton rest mass	M _P	$1.672614 \times 10^{-27} \text{ kg}$
Neutron rest mass	→ Mn .	$1.674920 \times 10^{-27} \text{ kg}$
Avogadro's number	N _A	6.022×10^{23} /gram-mole
Boltzmann's constant	k .	1.38 x 10 ⁻²³ joule/ ⁰ K
Gas constant	R	1.987 cal/gram-mole ^O K
	•	1.987 Btu/lb mole ^O R
Planck's constant	h	6.62 x 10 ⁻³⁴ joule∙sec
Stefan-Boltzmann constant (black body)	σ,	$5.67 \times 10^{-8} \text{ W/m}^2\text{K}^4$
Velocity of light	C	2.9979 x 10 ⁸ m/sec 186,282 mi/sec
PREFIXES	· •	•
micro (μ) = one millionth	- hecta (h)	= one hundred times
milli (m) = one thousandth	kilo (k)	= one thousand times
centi (c) = one hundredth	mega (M)	= one million times
deci (d) = one tenth	giga (G)	= 10° times
deka (D) = ten times	tera (T)	= 10 ¹² times



Appendix A-2 Conversion of Units

Frue	To:	Multiply by	Fran	To	Multiply by
ere-ft	gal	3.26 x10 ⁵	cal/g	Btu/1b	1.80000
H	litere	1.233 x10 ⁶	-	Wh./kg	1.1633
	_	1233.5	cal/800	Btu/h	14.206
-	_	0.4047	#	•	4.184
.a.	ba			5/84c	4.184
agstros (%)	**	1, ×10-4		Stu/sec	3,9684=10-3
ten .	kg/s ²	1.0332×10*			3.0860
-	kg/cm ² /	1.0332	1 "	fg-1b/eec	4.2665x10 ⁴
**	g/cm ²	1,0332x10 ³		g-cu/sec	1.3272x10 ⁴
•	1b/10 ²	14.70	cal/cm ² sec	Brw/ft ² h	
61	tons/ft2	1.058	-	T/cs²	4.164
A 5	e t=	0.9869	` " .	#/= ²	4.184 x10
	dyne/cm ²	10 ⁵	-	J/m ² sec	4.184 x10 ⁴
	hg/m²	1,020 ×10 ⁴		kw/=2	41.84
	1b/in ²	14.5038		aj/≘ ² h	150.62
		1,0548±10 ³	cal/cm	L (langley)	1.0000
Stu				¥sec	4.184
*	kJ	1.0548	2.	kw/m²	1.1622=10-2
*	ca l	251.996	cal/c=2h		1,1622110
#	kWh,	2.928 ×10-4	" .	¥/cm² .	
**	ri-lu	778.3	cal/cm ² min	k#/= ²	6.9732=10-1
Btu/h	ft-lb/sec	0.2162		L/min	1,0000
	cal/sec	0.0700	cal/ca C sec/c	Biw/ft Ph/ft	0.067197
		0.2931	-	Btu/ft270h/ga	0,80636
•	W _E	-		J/cm ² C ⁰ sec/ca	4.184
Dtu/=1m	raf	0.01757		W/cm ² C°/cm	4.184
*	w _t	17.57	1	W/a ² Co/ca	4,184 ×10 ⁴
Btu/1b	J/kg	2.326 %10 ⁻³	<u> </u>		0.39370
**	cal/g	0,\$5555 .	(14)	10	
ate ² sis	w/= ²	18.914	1 "	11	0.03281
. ,	cal/cm ² uec	4.521 x10 ⁻³	c=2	, 10,3	0.15500
Btu/ft ² b	cal/em ² sec	7.5344×10 ⁻⁵		rt ²	1,0764=10-3
	J/cm2soc .	J. 1524×10 ⁻⁴	can de	in ³	6.1023x10 ⁻²
٠, ٠	1/12,	3 . 1524		ft ³	3,5315x10 ⁻⁵
,,	1 /1 2		cf=	liters/sec	0.4720
••	kJ/m²h	11,348	day	##C	86,500.0
•	,cal/cm ² b	0.27125	1 -	_	10-7
	cal/cm2min	4.5208m10 ⁻³	dyne	J/mec	10-6
	L/eia	4 5208+10-3	dyse-cm2	ber	10
Btu/ft ² f ⁰ h	cal/s ² C ⁰ sec	1.3573	erg '	Btu	9.4845×10 ⁻¹³
	v/∍²c°	5,6783	"	cm l	2,3901×10 ⁻⁸
Stu/ft ² F ⁰ h/ft	cal/cm ² C ⁰ h/c=	14.881	٦,	g-c=	1.0197m10 ⁻³
efalit & alli	J/m ² C ⁰ sec/m	1.73073 \$		kWh.	2.7778x10 ⁻¹
•		41.13548	111	υ#	36 . 480
	Cal/m2CORor/em			8	0.30460
"	W/=2C°/c=	17.294			0.0295
Btu/ft ² F ⁰ h/in	cml/cm ² C ⁰ h/cm	1 2401	ft(water)	ata lb/in ²	
*	kJ/m ² coa/cm	100.45	1	10/10	0,4335
**	cal 'm ² C ^o aec/cm	3.4447	i,	kg/= ²	304.8
••	cal/cm2Cosec/cm	4447×10 ⁻⁴	112	c 2	929.03
	/=2co/c	14.412		_2	0.092303
	*/= ² c ^o /=	0.14412	rt ³	ca _j	2.8317×10 ⁴
A				3	2,8317×10 ⁻²
cal .	g co	42664 . 9	.	. gal(U.S.)	7,49052
•	J	4.184			28 3106
	Btu	3.9683×10 ⁻³	 	litere	
e#	kWb.	1.1622×10-6	ft/sec	CM/NEC	30.48
**	kJ [*]	4.184 x10 ⁻³	"	km/h	1.0972#
	En,	1.1622×10-3	1	m1/6	0.68182

From	To	Multiply by	Fram	To	Multiply by
t-15	Btu	1.3841x10 ⁻³	k a	-1	0.62137
	ca l	0.32405	< n	- ft	3280.84
	3	1.35582	kca i	J	4,1840 <u>x</u> 10 ³
al(U.S.)	cm ³	3785.41	ka ²	8c700	247.105
	1b(mater)	8.34517		mi ²	0.38610
	kg(unter)	3.7852	120/0	ft/mec	0.9113
•	rt ³	0.13368		m1/h	0.62137 -
#	Acre-ft	3.0689x10 ⁻⁶	knote	cm/sec	51.444
**	liters	3.7852		fi/eec	1.6878
		2,2046×10 ⁻³		=1/b	1,1508
1-2	1b/1s-	1.4223x10 ⁻²	-	ha/h ',	1.652
/c="		9.6784x10 ⁻⁴	}	Dtu/min'	4
. 3	a tu	9.6784x10 10 ³	k¥.	- •	56 . 8253
/cs ³	kg/s			Bt u/h	. 3414.43
**	1b/1=3	3.6126x10 ⁻²		hp	1.3410
м	10/152	63.4280	. "	cal/min	1.43197±19
-01	Btu ,	9.3011x10 ⁻⁹	-	cs 1/apc	238 . 662
•	3	9,2066x10 ⁻⁵	"	J/sec	10 ³
-cs/80c	ca1/sec	2.3438x10 ⁻⁵	*	LJ/h	3600
P	Stu/min	42.4356	k Wh	Btu "	3410.08
•	24	0.74570		cal.	9.59336 x 10
69	2.1-sec	0.74570	-	kce 1	8,59326=10
	cel/min	1.06936=104	.	ti.	3600
	CE1/211	2.54000		16(HgO evap.)	3,55
n(vetor)	lb/ip ²	3.6126±10 ⁻²	1 2	Btw/ft ² h	317.81
	19/19 ke/s ²	25,3993	'k#/=2		
		2.458 ×10 ⁻³	 		0.10000
	e te cm²		. "	cal/cm ² sec	0.23901
	ft ²	6.4516 6.9444±10 ⁻³	,	kJ/s ² k	3000
3 .	3		L (langley)	cal/cm²	1.00000
in" .	cm ³	16.3871	" .	j/cm²	~ 4.184
-		1.6387×10 ⁻⁵		2	41.84
-	rt ³	5.7870×10 ⁻⁴	"	Btu/ft ²	3.6866
is(fig)	a t u	3,3421=10 ⁻²	L/min	cal/cm ² win	T.0000
•• '	ft(water)	1.1330	"	ea 1/cm ² soc	1,6667x10
1	Btu	9.48451×10 ⁻⁴	-	Btu/ft ² h	1.3272x10
"	<u>kWb</u>	2.7778x10 ⁻⁷	liter	is ³	61,0254
~	c=1	0.239006	"	gali	0.20418
e1	erg	107		1 113	3.53157×10
••	ft-16	0.737562		3	1000
**	77)	2,7778x10 ⁻⁴	15/18 ²	g/cm²	70,3070
I/cm ² .	cal/ca ²	0,239006		kg/m²	703,070
1/60c	*	1	ib/in ²	g/c=3	27 . 6799
	Stu/min	5,6907x10 ⁻²		kg/m ³	2.78799×10
**	cal/min	14.3404	lumen	W -	1.4706
••	hp .	1.34109x10 ⁻³		- -i	6.2137×10
ıJ	***	2,7778x10 ⁻⁴	1	-1 [1	3,2808
J/m ²	Stu/ft ²	8.8111x10 ⁻²	 		39.3701
KJ/= "	cal/n ²		. 2	in ft ²	
**	cal/m ²	239.006	B		10.7439
	cai/cm ⁻	0.02390	"	ທ ²	1550.00
2	kcs1/m ²	0,23901	1 "	ha.	- ₁₀ -4
ica 1/m²day	<u>#J/=²dar</u>	4184.0	 	scres	2,47105=10
••	kJ/m ² day	4.1840	•3	rt ³	38.3147
te _	16	2.20462	1 "	acre-ft	8.1071×10
ig/cm ²	s to	0.96784	•	in ³	8.10237×10
•	ft(ester)	32.8093		gal (U.S.)	264,172
	16/ft ²		1	-	

From	To	Multiply by	From	To	Multiply by
Dtu .	kJ MkJ kWh	1.0548×10 ⁶ 1.0548 292.875	thern	kuh MJ kcal	20.28 1.0548x10 ² 2.5200x10 ⁴ 3.41443
DLJ/m ² IS	kBtu/ft ² ydn	88.11 1760		Stu/h cel/sec	0,239045
	to km km ² acres	1609.34 1,60934 2,58999 840.000	¥/ca ²	hp J/sec cal/cm ² sec Btu/ft ² h kW/m ²	1,34192±10 1,00000 0,239045 31,721 10,0000
(newtow)	dyses 1b E	234,2989 - 10 ⁵ 0 .03108 14,0981 453,5924 0 .453592		de'l/ce ² min l/min hJ/m ² h cai ft-1b	14.3310 14.3310 3.80000×10 859.184 2655.22
thers	1256 - Ben FE	0,45392 10 ⁵ 0,10000		kg-m kJ/m²	347,098 3.6900x10

Appendix A-3

Conversion of Celsius to Fahrenheit

								_		
Temp oc	0	1	2	3	4	5	•	7	8	
-50	-50.0	-50.6	-61.6	-63.4	-65.2	-87.0	-68.8	-70.6	-72.4	-74.2
-40 .	-40.0	-41.8	-43.8	-45.4	-47.2	-49.0	-50.8	-52 6	-54.4 -36.4	-64 3 -38 3
-30	-22.0	-23'. 8	-25.5	-27 . 4	-29.2	-31.0	-32.0 -14.3	-34.6 -16.6	-10.4	-30.2
-30	- 4.0	- 5.8	- 7.8 +10.4	- 9.4	-11.2 + 6.8	-13.0 + 5.0	-14.5 + 3.2	+ 3.4	- 0.4	- 2.2 '
- 10	+14.0	+12.2					21.2	18.4	.17.8	+15.0
	33.0	30.2	28.4 35.6	26.6 37.4	39.2	23.0	43.8	44.6	46.4	48.2
+ 0 +10	32.0 50 0	51.8	53.6	35.4	57.2	39.0	60.8	43.6	64 4	66.3
20	64.0	69.8	71.8	73,4	75 2	77.0	78.8	80 , 6	82.4	\$4.2
39	86 .0	87.8	₩.6	91 4	93 2	95.0	95.6	94.6	100.4	102,2
40	104.0	105.4	107 8	109 4	111.2	113 0	114.8	136.6	118.4	120.2
50	133.0	123.8	125.6	127 4	129.2	131.0	132.8	134.6	136.4	136.2
60	140.0	141.6	143.6	149.4	147 2	140.0	180.8	183.6	154.4	174.2
70	150.0	150.6	161.6	163.4	165.2	147.0	166 6 166 8) MM . 6	190.4	163.2
90 90	376.0 194.0	177 8 195.a	179.8	191.4 199.4	301 '2	103.0	204.8	204.6	206 4	210.3
100	212.0	213 6	215.4	317.4	316.3	221.0	292.8	224.6	226.4	228.2
110	\$30.9	. 231.0	233 8	235.4	237.2	239 0	240.8	243 6	244.4	344.3
120	248.0	249 8	251.6	253.4	755.2	257 0	258.8	250,6	263 4	284.2
130	266.0	267.8	269.6	271.4	273.2	275.0	276.8	278.6	280.4	142.2
140	284.0	285.8	267.8	280.4	291.3	293.0	294.8	296.6	298.4	300 / S 316 / S
150	302.0	303.8	305.6	307 4	300.2	311.0	350.8	314.6	316.4	339.3
150	320.0 330.0	381.6	341 6	325.4	327.2	347.0	345.8	350.6	352.4	354.2
170 180	358.0	357.6	250.6	361.4	363.2	345.0	366.8	368.6	370.4	372.2
190	374.0	375 8	397 8	379.4	381.2	343 0	384 8	386 6	388.4	300.9
200	392 0	393.8	395 6	397.4	399 2_	491.0	402.8	404.6	406.4	408.2
210	410.0	411.8	413.8	415.4	417.2	419.0	420.8	422 6	424.4	435.3
220	428.0	429 #	431.6	433.4	435.2	437.0	436.8	440.6	442.4	444.2
230	446.0	447.8	449	451.4	453.2	435.0 473.0	454.8	458 6 476 6	476 4	480.3
. 240	464.0	465.6 463.6	467 6	469 4	471 2 4#9 2	491 0	492.8	494 #	496 4	498.3
250	500.0	30) 8	503.6	505.4	507.2	509.0	510.8	512.6	514.4	. 518.2-
270	518.0	519.8	521.6	523 4	525 7	527.0	526 8	530 6	532.4	534.2
280	536 0	537 #	5.89.4	541 4	543.2	า≰กย	546.1	244 #	250 4	552.2
340	554.0	555 =	207 6	550.4	561.2	\$6.1.0	584.8	544 6	564.4 584.4	570.2 568.2
300	572 0	573 A	575 8 59J 6	505.4	579 2	581.0 589.0	5H2 #	602.6	604.4	605.2
3 1 0 3 2 0	590 0 608.0	591 # 609 \$	611 6	813.4	615 3	817 0	818.8	430.6	827.4	624.3
330	525.0°	827 8	629 6	631.4	633.2	935.0	836.8	. 438 6	640.4	842.2
340	844.0	645.4	647.0	549 4	651.2	0.020	454.8	656.4	858.4	860.2
350	642.0	643.8	645.6	447.4	649.9	671.0	672.4	674.6	876.4	678.2
380	880.0	641 8	683 8	***	887.2	420.0	490 . 1	657.5	654 4	854 3
370	38. 0	890 8	701.6	703 4	705.3	707.0	70# 8 72 6 .8	710 d	712.4	714.2
380	716.0	717 8	719 8	721.4	-783.8 741.2	743.0	744.8	746.6	748.4	750.2
390 400	752.0	733.8	755 6	757.4	759.2	761 0	749.8	764 6	766 4	788 2
410	770.0	771.8	773 4	775.4	777.8	778.0	780.8	783.6	784.4	780.2
420	788 0	786 %	791.6	783 4	795.3	797.0	798.8	#0Q. 6	802.4	804.2
430	804.0	407.8	MO9 8	811.4	813.2	9120	816.8	610.6	820.4	632.2
4 40.	824.0	#25.g	427.6	829.4	431.2	к33.0	#\$4.8	838 . 6	836 4	840.2
450	842.0	843 8	H45.8	847.4	H49 2	M51 0	8.3	854.6	856 4	858.2
460	860.0	861.8	8 634	865.4	867.2	869.0	27 02	872.6	902 4	878 2 804.2
470	878 0	979.8	BAL 4	88344	903 2	887.0 905.0	904.8	908.6	910.4	e12.2
490 490	994 0 914.0	997 M	917 6	901 4 919 4	921 3	923 0	924.8	926.6	928.4	930.2
500	932.0	933	935 6	937 4	939 2	941.0	942 8	944.8	946 4	948.2
510	950.0	951.8	953.6	955 4	957.2	959.0	960.6	962,6	984 4	906.2
520	968.0	240 1	971.0	973 4	975.2	977.0	975.8	980.6	982.4	984.3
530	985.0	987.0	989.6	991.4	993 2	995.0	996.8	998.6	1000 4	1002.2

Conversion formulae:

$$^{\circ}$$
C = 5/9($^{\circ}$ F-32)

$$^{\circ}F = 9/5 \, ^{\circ}C + 32 \, 1 \, ^{\circ}$$

Appendix A-4

Energy Ratings of Fuels

FUEL	Btu .	Per Unit
Coal:		
Anthracite ,	12,700	pound
Bituminous	13,100	pound
Coke	12,400	pound
Hydrogen (dry)	325	ft³
~ / ·	62,050	pound
Natúral Gas (dry) .	1,035	ft³
	5 25,047	pound
Natural Gas Liquids (average)	21,325	pound_
	4,412,000	barrel
Butane (C4H10)	21,400	pound
•	4,506,000	barrel
Propane (C ₃ H ₈)	21,600	pound_
•	4,402,000	barrel
Petroleum:		\
Crude 0il	5,800,000	' barrel
*	138,100	. gallon
	18,100	pound
Gasoline '	5,253,000	barrel
	125,000	ga 1.1 on
•	22,200	pound
Kerdsene	5,670,000	barrel
	135,000	gallon
	19,700	pound

FUEL AND ENERGY EQUIVALENTS

1 bb1 crude oil	= 443 lb bituminous coal = 5,604 ft ³ natural gas = 1,700 kwh electricity
1 short ton bituminous coal	= 4.52 bbl crude oil
	= 25,300 ft ³ natural gas
• ,	= 7,679 kwh electricity
7000 ft ³ natural gas	= 79.01 1b bituminous coal
	= 74.95 gallons crude oil
·	= 303.34 kwh electricity
1000 kwh electricity	= 1 Mwh electricity
•	= 260.5 1b bituminous coal
•	= 3,397 ft ³ natural gas
	= 0.588 bbl crude oil

Appendix A-5
Specific Heat and Heat Storage

Specific Heat of Materials

Element		Specific heat. cal/g	Comprund		Specific heat, cal/g
Hydrogen	H ₂	3.41	Water	H ₂ O	1,00
He.lium	He	1.24	Lithium hydride	Lin	0.6
Lithium	Li	0.85	Lithium fluoride	L1F	0.37
Berillium	Be	0.436	Lithium chloride	LiCl	0.28
Sodaum	Na	0.293	Sodium fluoride	MAT	0.28
Nitrogen	N ₂	0.249	Sodium chloride	NaC1	0.82
Neon	Ne	0.246	Calcium'fluoride	CaF,	0.21
Boron .	В	0.245	Potassium fluoride	KF ~	0.20
Magnesium	Mg	0.243	Sodium hydroxide	Na Off	0.2
Oxygen	02	0,219	Potassium hydroxide	KOH	0.2
Aluminum	Al	0.215	Magnesium chloride	MgCl ₂	0.19
Pluorine	F ₂	0.197	Calcium chloride	CaCl ₂	0.16
Phosphorus	p	0.181	Potassium clhoride	KC1	0.16
Potassium	K,	0.180	Iron chloride	PeC1 ₃	0.15
Sulfur	s	0.175	Zinc chloride	ZnCl ₂	0.14
Silicon	S 1	0.168	Copper chloride	CuCl ₂	0.14

Heat Storage Capacity

	Density P	Specific heat c	Volume head capacity pc	Thermal* . conductivity k	Thermal diffusivity k/pc
Material	g/cm ³	cal/g C°	cal/cm ³ C°	cal cm/cm2 Cosec	cm²/sec
Water .	1.00 '	1:00	1.00	0.0014	0.0014
Iron shot	7.86	0.13	1.02	0.160	0.157
Oils	1.00	0.60	0.60	0.00036	0.0006
Rock '	2.50	0.20	. 0.50	0.0060	0.0120
MgCO ₃	3.0	0.20	0.60	0.0025	0.0042
MgCO3 6H2O	1.7	0.38	₹ 0.65	0.0030	0.0046

Appendix A-6

Heat of Fusion of Materials

Heat of Fusion of Materials

Material		Heat of fusion, cul/g	Melting temperature, °C
Beryllium chloride	BeC12	310	547
Sodium fluoride	Ka F	168	992
Lithium hydride	Lin	139	680
Nickel chloride	NiCl ₂	139	1030
Sodium metaborate	Na HO ₂	135	966
Beryllium fluoride	Ber ₂	128	. 800
Sodium chloride	NaC1	123	- 803
Potassium fluoride	KIP	117 ,	860
Calcium silicate	CaSiO ₃	115	1512
Magnesium chloride	MgCl _p	109	708
Lithium hydroxide	Lion	103	462
Magnesium fluoride	MgF ₂	95	1396
Lithium fluoride	Lip	93	896
Sodium silicate	Ma 8 10 ₃	. 84	1087
Iron chloride	FeC12	82	677
Potassium chloride	10C1	82	776
Lithium silicate	L1810 ₃	80	1201
Boric oxide	B ₂ O ₃	76	149
Lithium chloride	Lici	.76	613
Mangamose chloride	MnCl,	71	708
Chromium chloride	CrCl	66	814 *
Aluminum chloride	Alacia	6 3	190
Sodium browide	Na Br	61	742
Potassium bromide	KBr	59	<i>t</i> ' 730
Sodium hydroxide	Na OH	38	318
Lithium browide	LiBr	34	, 547
Potassium nitrate	XXXO3	28	337
Copper chloride	CuCl	25	430
		<u> </u>	•

APPENDIX B

177

Appendix B-1
Useful Solar Flux Quantities

Quantity ,	kW/m²	MJ/m²k	Btu/ft ²
Extraterrestrial solar flux (Solar Constant)	1.353	4.871	429.2
Desert sealevel, noon, direct (D)	0.97	3.49	308.
Desert sealevel, noon, direct + scattered (D+S)	1.05	3.78	334.
Standard mealevel, noon, (D)	0.93	3.35	295.
Standard sealevel, moon, (D+S)	1.03	3.71	327.
Urban, typical, noon, (D)	0.61	2.16	193.
Urban, typical, noon, (D+S)	0.81	2.92	257 .
24h average, Desert, fully tracking, (D+S)	0.40	1.43	149.
24h average, Desert, fixed & tilt, (D+S)	0.24	0.87	76.
Ratio, noon/24h average, Desert, fully Ratio, noon/24h average, Desert, fully Ratio, noon/24h average, Desert, fixed Ratio, noon/24h average, Desert, fixed Ratio, extraterrestrial/24h average, De Ratio, extraterrestrial/24h average, De	tracking, l tilt, (E l tilt, (E sert, FT,	(D+S) 2.57 d) 4.30 d+S) 4.20 (D+S) 3.40	·

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Appendix B-2

Normal Monthly Average Temperature of Selected Cities

(In Fahrenheit degrees. Airport data unless otherwise noted. Based on standard 30-year period, 1931 to 1960.)

	Location	Jan	Feb	Mar	Apr	May	lun) tal	Aug	Sep	Oct	Nov	Des	Ann
AL	Mobile	53.0	55.2	60.3	676	75.6	81.5	82.6	82,1	77,9	69.9	58.9	54.1	68.2
AK	Une all	25.1	26.8	30.4	38.0	45.6	52.3	55.3	54.1	489	41.6	34.3	28.4	
ΑZ	Phoenix	49.7	53.5	59.0	67.2	75.0	81.6	29.8	87.5	82.8	70.7	58.1	51.6	69.0
AR	Little Rock	40.6	44.4	\$1.8	62.4	70.5	~ 78.9	819	81.3	74.3	63.1	49.5		61.7
CA	Los Angeles	- 54.4	55.2	57.0	59.4	62.0	64.8	69 1	69 1	68.5	64.9	61.1		61.9
	Sacramento	45.2		53.4	58.4	64.0	70.5	75.4	74.1	71.6	63.5	52.9	46.4	60.4
	San Franciscot	50.7	53.0	54.7	55,7	57.4	59:1	58.8	59.4	62.0	61.4	\$7.4	\$ 52.5	\$6.8
co	Denver	28.5	31.5	36.4	45.4	56.2	66.5	1 2 .9	71.5	610	31.4	37.7	31.6	49.5
CT.		26.0	27 1	36.0	48.5	\$9.9	68.7	73.4	71.2	63.1	§3.0	41.3	28.9	49.8
DΕ	Wilmington	33.4	33 8	41.3	\$2. L	62.7	71.4	76 0	74.3	676	. 46.6	45.4	35 1	54.1
DC	Washington	36.9	37 8	g 44.8	\$5.7	65.8	74.≩	78.2	76.5	69 7	190	47.7	36.1	570
FL	Jacksonville	55.9	57.5	62.2	69.7	75.8	80.8	82.6	82,3	79.4	71.0	61.7	56.1	69.5
٠.	Mismi	66.9	67.9	70.5	74.2	776	6 0.8	£1.\$	87.3	81,3	77.8	72.4	68.1	75.1
GA	Atlanta	44 7	46.1	51.4	· 6Q.2	69.1	76.6	78.9	78.2	73.1	62.4	51.2	44.8	61.4
HI	Honolulu	72.5	72.4	72.8	74.2	75.9	77.9	78.2	79.4	79.2	78.2	75.9	73.6	75.9
ID	Boise	29.1	34,5.	41.7	50.4	58.2	65.8	, 75.2	72.1	62.7	51.6	38.6	32.2	51.0
iL.	Chicago	26.0	27.7	36.3	49.0	60.0	70.5	75.6	74.2	66.1	55.1	39.9	29.1	50.8
	Peoria	25.7	28.4	37.6	50.8	61.5		76.0	74.3	66.4	55.3	39.7	29.1	51.4
IN	Indianapolis	29.1	31.1	38.9	50.8	61.4	71.1	75.2	73.7	66 5	55.4	40.9	31.1	521
IA	Des Maines	19.9	. 23,4	53.8	48.7	. 60.6	71.0	76.3	74.1	65.4	54.2	37.1	25.3	49 2
KS	Wichita	32.0	36.3	44.5	56.7	66.0	76.5	80.9	80.8	71,3	59.9	44.4	35.8	57.1
KY	Louisville	35.0	35.8	43.3	- 54.8	64.4	73.4	776	76.2	69.5	57.9	44.7		55.7
L A	New Orleans	54.6	- 57 L	614	67 9	24.4	RO.1	81.6	81 9	78 i	70.4	60 C		68.6
MI	Pertland	21.8	22 R	11.4	42%	510	623	68 1	1-0 X	*# 7	48.6	18.1		45.0
MD	Baltimore	34.8	35.7	43 f	54.2	64.4	72.5	76.8	75.0	68.1	57.0	45.5		55.2
MA	Baston	29.9	30.3	37.7	47.9	58.8	67.8	73.7	71.7	65 3	55.0	44.9		51.4
MI	Detroit	26.9	27.2	34.8	47.6	59.0	69 7	74 4	728	65.1	53.8	40.4	29.9	50.1
	Sault Ste. Marie	15.8	15.7	23.8	38 0	49.6	59.0	64.6	64.0	55 8	46° 3	33.3		40 6
MN	Duluth	8.7	IÓ E	21.3	370	49.2	58.8	65 5	638	54.2	44 6	27.3		37.9
	Minne apolis								•					
	St Pand	124	15.7	27 4	44.3	57.3	66.E	72.3	70.0	60 4	48.9	31.2	18.1	43.7
MS	lackson	479	\$0.5	56.5	64 9	73 I	79 8	82.3	82.0	76.5	670	55.5	49 4	65.5
MO	Kansas City	31.7	45 B	433	55.7	65.6	75 9	81.5	79 #4	713	60.2	44.6	35 E	16 8
	SI Loun	119	34.7	42.6	154.9	. 64.2	74.1	78.1	76.8	69 5	58.4	44.1	34 8	45.3
MI	Great Laffe	22 1	238	10.7	4 1.6	51.0	199	69.4	66.8	5/4	475	34.3		44.7
NE	Omaha	22.3	26 5	16.9	51.7	630	741	78 4	76.2	66.9	55.7	189	28.2	51.5
NV	Reno	10 4	35.6	41.5	48.0	53.9	60 1	67.7	65.5	* 58 B	49.2	36.3		48 4
NH	Concord	21.2	22.7	31.7	415	55.5	64 5	69 6	67.4	59 3	48 7	17.6	25.0	45 6
NI	Atlanta City	14.8	34.7	41.1	51.0	61.3	70 0	75.1	73.7	67.2		46.7	36.6	54.1
NM	Albuquerque	15 0	19.9	45 8	55.7	65.1	74 9	18 5	76.2	70 Q	58.0	43.6	37.0	56.6
NY	Albany	22 7	23.7	13.0	46.2	57.9	67.3	721	700	61.6	50:8	39 1	26.5	476
	Bulfalo	24.5	24.1	31.5	43.5	\$4.8	64.8	69.8	68.4	614	50 8	19 1		46,7
	New York	32.2	33.4	40.5	51.4	624	71 4	76 8	75 1	68.5	58 3	470	-	54.5
NC.	Charlotte	42.7	44.2	50.0	60.3	64 0	771	79.2	78 7	129	62 5	50 4	427	60 8
	Raleigh	41.6	43.0	49.5	59-3	676	75.5	779	76, 4	71.2	60.5	50.0	41.9	
ND	Bismarck	9.9	135	26.2	415	559	64.5	71.7	69 1	58.7	46 7	28 9	17.8	
ОН	Lincinnati [†]	337	35 1	42 7	54.2	64.2	734	76 9	75.7	69 Q	57.9	44.6		55.2
	Cleveland .	28.4	28.5	35.1	47.0	\$8.0	67.8	719	70 4	64.2	53.4	41.3		49.7
	Colembus	29.9	31.1	38.9	50.8	61 5	70.6	74 8	73.2	659	54.2	41.2	-	12.0
ЭK	Oklahoma City	37.0	41.1	48 5	19 9	68.4	78.0	825	82.8	7 F B	62.9	48.4	40.3	
)R	Pur tiand	48.4	42.0	46 1	5 1 R	57.4	620	6.7.2	66.6	62.2	54.2	45 1	41.1	
A	Philadelphia	32.3	33.2	410	52.0	62.6	71.0	.156	716	66.7	55 7		33.9	1
	Pittsburgh	28 9	29 2	36.8	49 0	59.8	68 4	721	70 R	64.2	531	44.3 49.8	30 7	
31	Providence	79 2	29.7	17.0	47 2	575	66 2	72.1	70 S	612	512	430		
ic.	Columbia	46.9	48.4	54.4	63.6	72.2	79.7	81.6	80 s	75.3	64.7	53.7	32.0	
50	Sioux Falls	15.2	191	30 1	45.9	58.3	68.1	74.3	71 8	.618	50 3	32.6	21 1	64.0 ×
N	Memphis	41.5	44.1	51.1	61.4	70 3	78.5	813	80 5	439	63 1	50.1	42.5	
	Nashville	399	42.0	49.1	59 6	68.6	77.4	80.2	19.2	128			41.4	
X	Dallas	45.9	49.5	56.1	65 0	72.9	81.3	849	85.0		61 5	48.5		
•	El Paso	42 9	49 1	54.9	63 4	71.9	810	819	80.4	17.9 74.5	67.8	54.9	48 1	
	Houston	53.6	55 8	61.3	68 5	76 D	81.6	830	83.2	79.2	64 4	51 2	44 1	
T	Salt Lake City	27.2	32.5	40,4	49 9	189	67.4	76.9	14.5	64.4	71.4	60.E	55 7	
/1	Burlington	16.2	17.4	26 7	41.2	5 1 A	64 2	690	66 7		51.7	36 7	10.1	
/4	Norfolk	41.2	41 6	48 ()						58.4	47.6	35 3	21.5	
	Richmond	18 /	10 4		5 M 43	4.15	14 1.	/ N. N.	17 5	114.	620	51.4	42.5	
/A	Seattle Tacoma			411	58 i	6/0	15.1	/N I	16. 11	10 2	5# 7	48.5	19.1	
- ~	Spokape	38.3 24.5	40 S	454	49.2	54 4	59.8	64 9	641	544	52.4	419	40.8	
1 V	Charleston		10.0	181	47 3	56.2	617	70.5	6R በ	(40 g	49 1		10 1	
"	Milwaukee	16 6 20 6	375	44 4	55 j	8 48	72.0	74.9	718	6362	57 3	45.3	37.1	
		20.6	22.4	31 0	43.6	53.4	63 3	68 7	67.8	60.3	50 0	35.8	24.6	
, ,	San Juan	+ 74.4	27.3 74.4	32.4 75.3	47.0	17.4	610	70.0	6 77	58.6	47.5	34.2	29.5	45 9
YY	Cheyenne	25.4	27.3	32.4	42.6	529	630	70.0	67.7	58.6	47.5			

^{*}From U.S. Department of Commerce, NOAA, "Local Climatotogical Data" For more detailed data see U.S. Department of Commerce, NOAA, NCC, "Climatography of the United States," Publ. No. 84
†City office data.

Appendix B-3

Normal Total Heating Degree-Days

,	Location) ul	Aug	Sep	Oct	Nov	Dec) am	i tp	May	Apr	May	jun	Ann
			•		93	363	555	592	462	363	108	9	0	2,551
AL	Burningham	0	0	12	121	426	663	694	557	434	138	19	ō	3,070
	Mobile	`. `.	0	6	22	213	357	415	300	211	-ag12	. 0	0	1,560
	Montgomery	Ō	. 0	0 -	68	330	527	543	417	316	⁷ 90	0	0	2,291
AK	Anchorage	245	291	516	930	1,284	1,572	1,631	1,316	1,293	\$79	592	315	10,864
	Annette	242	201	327	567	7 3 4	299	949	#37	843	648	490	321	7,069
	Barrow	803	840	1,015	1,500	1,971	2,362	2,517	2,332	2,468	1,923	1,445	957 924	20,174 19,862
	Barter Is.	735	775 3 94	987 612	1,452	1,944	2,317 1,866	2,416 1,901	2,369 1,590	2,477 1,655	1,173	804		13,194
)		19 474	425	525	7/2	918	1,122	1,151	1,016	1,122	951	791	591	9,880
•	Cordeva	166	301	522	761	1,017	1,221	1,299	1,086	1,113	864	860	444	9,764
	Fairbanks "	171	312	642	1,203	1,831	,2,254	2,1591	1,901	1,739	1,045	1555		14,279
	Juneau '	101	335	453	725	921	1,135	1,247	1,070	1,073	810	108	38 1	9,075
.,	King Salmon	113	372	411	908	1,290	1,606	1,600	1,414	1,411	966 1,554	673 1,057	636	11,343
	Kotzehue McGrafh	381 208	446 338	723 633	1,249	1,728	2,127	2,192 2,244	1,817	1,758	1,122	648	258	14,281
	Nome	481	496	69.1	1.094	1,455	1,820	1,879	1:666	1,770	1,314	930	173	14,171
	St Paul	605	519	612	862	961	1,197	1,228	1,168	1,265	1,098	936	7 26	11,199
2)	Shemya	· 511,	475	501	784	8 76	1,042	1,045	958	1,011	885	837	696	9.687
	Yakutat	314	347	474	716	916	1,144	1,169	1,019	1,042	840	632 437	435	9,092 7,152
AZ	Flagstaff	46	68	201	558	867° 234	1,071	1,169	991	911 217	651. 75	7,7	0	1,765
	Phoenix Prescott	0	0	0 27	22 245	579	797	865	711	605	360	158	15	4,362
	Turson	0	Ö	, 0	25	231	406	471	344	242	75	6	0	1,500
	Winslow	ō	0	6	245	711	1,008	1,054	170	601	291	96	0	4,782
	Yuma	0	0	0ر "	0	148	119	16.1	22R	1 10	20		0	1,217
AR	1 i. Smith	, 0	0	1.2	127	450	704	781	196	456	144	22	0	1,292
	Little Rock	' 0	0	9	127	465	716	756	577 468	434 350	126 105	9	0	3,219 2,533
	Texarkana Diskontrala	0	0	0	7 8 3 7	345	561 502	626 546	364	267	105	19	ō	
CA	Bakerstield Bishop	~~ o	Ö	42	248	576	797	874	666	5 19	306	143	36	4,227
	Blue Canyon	14	50	1 20	347	579	766	865	781	791	582	397	195	
	Burbank	Q	0	6	43	177	301	166	272	239	138	81	18	-
	Eureka	270	257	258	129	414	499	546	470	505	438	372	285	
	Fresno	0	0	0 12	78. 40	319 156	558 288	586 175	406 297	319 267	150	56 90	15	
	Long Beach	0 28	22	42	78	150	291	172	102	288	219	158	81	2,061
	Los Angeles Mt. Shasta	25	34	123	406	696	902	983	784	738	525	347	154	5,722
	Dakland	53	50	45	127	• 309	481	527	400	153	255	180	90	-
	Pt. Arguello	202	126	162	205	291	400	474	147	403	139	298	24 1	
	Red Bluff ,	0	Ó	0	5.1	118	555	604	42R	141	168	47	0	-
	Sucramento	0	0	1 2 30	R1 202	163 4 8 0	577 691	614 178	442 661	160 520	216 426	102 264	57	
	Sandberg	0 6	0	15	37	123	- 251	311	249	202	123	24	16	
t	San Diego San Francisco	81	78	60	141	106	462	508	195	161	279	214	126	
	Santa Catalina	16	0	9	50	165	219	343	108	3 26	249	192	105	-
	Santa Maria	, 99	94		146	270	191	449	170	161	282	. 231	.665	
(0	Alamesa	84	99	2 19	6 19	1,065	1,420	1,476	1,162	1,070	696	446	168 84	
	(Byenade Shrinke	.9	/ 25 9	132	456	#25 #19	1,012	1,128	41 E	89 1 88 7	582 558	317 288	66	
•	Denver Grand Junction	6	0	10	313	786	. 1,113	1,209	407	729	387	146	21	5,641
	Purbio	0	ō	54	326	750	986	1,085	871	772	429	174	15	
C T	Bridgepont	0	0	66	307	615	986	1,079	41.6	853	510	204	27	
	Hक्रीord	0	6	99	372	711	1,119	1,209	1,061	899	495	177	24	-
~.	New Haven	0	12	g7 51	347 270	648 588	1,011 927	1,097 980	991 874	871 735	543 387	245 112	45	
DE FL	Welmington Appliachrooia	0	0	0	16	154	119	347	260	180	33	0	Ò	
,' •	Daytona Beach	Ö	ŏ	ŏ	0	75	211	248	190	140		0	0	
	f L Myeri	0	0	0	.0	24	109	146	101	62	0	0	0	
	Jacksonville	0	0	0	12	144	110	132	246	174	21	0	0	
	Key West	0	0	0	0	0	28	40	e 31	9	0	0	0	
	Lakeland Moreo Beach	0	0	0	0	' 57 0	164 40	195 56	146 36	99	0	0	0	
	Mrami Beach Orlando	0	0	0	0	12	198	220	165	105	6		0	
	Pensacola	0	ŏ	ő	19	195	353	400	211	183	36	ō	0	
	Tallahasser	0	0	0	28	198	160	3.25	286	202	16	0	0	1,485
	≤I ampa	11		1)		60		J 11.7		104			t	
	W. Palm Heach	0		0			,		64		٥ '	-	0	
GA	Athens	0		12	115	405		642	5 <u>7</u> 9	431	141	27 25	0	
	Atlanta Ægusta	0	0	18	127 % 78	414 333	552	619 549	529 445	437 350	168 90		0	
	Columbus	9	ŏ	0		333		552		338	16		0	
	Macon	0	o	Q	71	297		505	403	295	4 63		C	-

"Climatic Atlay of the United States," U.S. Government Printing Office, 1968

177

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	Location	jul 🗀	Aug 5	ip ()		10+	Der _	parti						
		-	- •			474	701	710	577	468	177	34		,326
GÁ	Rome ·	0	•0	_		246	437	437	353	254	45	0		.819
	Savannah	0	0	0	47	195	366	394	305	208	33	0 -	0 1	,529
	Thomaville	0	0		25			1,113	854	722	438	245		,809
IĐ	Sone	0	-		615 623 1		•			.085	651			3,475
	idaho falls 46W	16				,		•		107	657	384 1		1,760
•	idaha talis 42NW	16				716			815	494	436) 14	461 , 4	1,142
	i emission :	٥	-		4() f				1,048	901	335	119	141 -	1,011
	Pen stylke	0	Ō		**		191	\$10	640	539	195	47	O	3,821
Ħ	(APO ~	0	Ó		164	513			1,044	190	480	211	45 (6,155
	Chuago '	Q	0		326			.,	1,100	918	450	189	39 '	6,408
	Malme	0	9	99	335		1,113		1.025	849	426	183	33 (6,025
	Noru	0		≛ 7	126	-	1,221		1.137	961	516	2 34	60	6.830
	Rockie -	•	9	114	400		1.023	1,135	935	769	354	136	1.5	5,429
	Springheld	0	0	.72	29) 220	606	290	955	76!	620	737-	68	0	4,435
£N.	Evanoville	0	۵	66		_	1,135	1/178	1,028	890	471	189	39	6,205
	I L Wayne	0	9.	105	178		1,051	1.11.5	949	109	432	177	39	4,699
	rioganapoin	0	0	90	116		1,125	1,2/1	1.070	913	525	239	60	6,439
	South Bend	8	6	111 91			1,135	1,259	1,042	859	426	177	33	6,114
14	Burington	0	0		322	837	1,231	1,398	1,161	967	489	211	19	6,805
	Drs Maines	0	9 ~	99	363 450	906	1,287	1,420	1,204	1,026	546	260	78	7,}76
	Debuque	12	731	116	369	167	1,240	1,435	1.198	989	483	214	39	6,551
	Sylvia City	Ð	9	104		909	1,296	1,460	1,221	1,023	531	229	54	7,320
	Waterloo	12		1 38	428 276	305	1,023	1,163	935	781	37,2	149	16	5,479
8.5	Couraign.	0	0	57	-	568	939	1,051	840	719	354	1 24	9	4,956
	Dodge City	0	0	",	251		1,071	1.166	118	184	1 507	2 36	42	6,141
	(rougland	0-	6	ا ۱۱ م	481 270 229	672	980	1,122	893	722	330	124	1.2	5,182
•	Tupeka	ð	0	33	229	618	905	1,023	804	645	270	≇ 7	6	4,620
	Wichild	0	0		291	669	983	1,035	893	756	390	149	24	5,265
K.	Committee	0	0	75 54	239	609	902	946	818	645	325	105	0	4,683
	Lexington	0	0	54	248	609	890	930	618	682	315	105	9	4,660
	Louisville	0	0	94	56	273	431	471	361	260	69	0	0	1,921
U		0	-		31		369	409	294	fos	33	0	8	1,560
	Baton Rouge	0	0	ó	70	96	214	298	_218	171	22/	0	0	1,024
	Burr wood	0	0	0	19		- 341	381	274	195	39	, . 0	0	1,459
	Lake Charles	0	0	ŏ	19	192	322	363	258	192	39	0	0	1,385
	New Orleans	0	.0	Ö	47	297	477	- 552	426	304	61	्री	0	2,184
	Shreveport	0		336	682	1.044	1,535	1,690	1,470	1,308	858 -	468	183	9,767
્, ₩	E FC#Ibou	78	115	195	508	807	1,215	1,339	1,182	1.042	675	372	111	7,511
	Portland	12	53	48	264	585	905	936	\$20	679	327	-90	0	4,654
4	D Baltimore	0		4	307	624	955	995	876	741	384	127	12	5,087
	Frederick	0		108	36!	590	1,065	1,178	1,053	936	7579	267	59	6,368
` 🛤	A Stue Hall Chay	0		60	316	603	983	1.088	972	146	513	204	36	5,634
	Beston	0		93	332	573	7 894	992	941	276	621	384	1 29	5,891
	* Mantucket	12		219		831	1,231	1,339	1,196	1,063	660	326	105	7,578
19	Pienlield	25			534 450	774	1,172	1,271	1,123	994	612	304	7.	6,969
	Work exter			142	550	912	1,268	1,404	1,294	1,218	777	448	156	8,506
		64		273	160	738	1,088	1,181	1,058	936	522	220	42	6,232
	Detroit (City)	, (E)	539	924	1,293	1,445	1,296	1,203	777	456	159	8,481
	f it suspe	59		243,		843	11,212	1,130		1,066	619	319	98	1,311
	Flunt	16		159	465	804	1,147	_ `		1,011	579	279	75	6,894
•	Liand Rapids		28	135	\$34	\$13	1,163			1,011	579	273	69	6,909
	[TUTUE		22	138	431	936	1,268			1,187	771	. 468	177	8,191
	Macquette ;	51		240	527	762				995	594	310	78	6 6 9 6
	Mary Enu	1		120	400 580	951	1,367			1,277	810	477	201	9,048
	Marie Sie Marie	9		279		, 1,131 1,131 ,	1,581			1,355	840	490	198	10,000
1	th Deluth	7		**0						1,414	826	443	174	10,606
	international falls			363	701	1,236				1,166	621	288	81	8,382
	Manneapolis	2		189	505 474	1,014				1,150	630	301	91	8,295
	Rochester		5 34	386	549	1,065				1,221	666	326	105	8,879
	St Cloud '		8 47	225		315				310	8.7	Q	0	2,239
1	MS Jackson		0 0	0.		339	_				81	. 0	Q	2,289
	Méndian		0 0	- 0	81 53	279	-				. * 69			2,841
	Ysck shurig		0	0							324		12	5,046
	MO Columbia		0 9	54	251	651		•			294	4		
	Kamas *		0 0	39	220						348			5484
	St. Juseph		0 6		285									
	St. Louis		0 0	60	251			•						
	Springheld		0 0	45	224									
	MI Bellings		6 15	186	487								5 . 150	
	wo g essel		31 47	270	608									
	Great Falls		28 53	258	543					-				
	Havre		28 53	106	599		•							*
	Helena		31 59	294										
	Kalnpell			* 321	654						579			9. 1,723
	Males City		6 6											
	Mesarada		34 74	303	65	1 1:03	5 1,28	,, ,, 4		- J''				

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	Focation	jeri	Aug	ΆP	* Oct	Nov	Dec	Jan .	1 60 x	Mar	, Apr	May	Jun	Ann	
" NE	Grand Island	0	· _ 6	108	381	834	1,172	1,314	1,089	908	`462	211	15	6,510	
	Lincoln	0	6	75	301,		1,066	1,237	1,016	834	402	171	J ₅₀	5,864	
	Norfolk	9	0	111	397	873	1,234	1,414	1,179	983	498	233	/48	6,979	
- 64	North Platte	0	6	127	440	885	1,166	r,271	1,039	930	519	248	57	6,684	
	Omaha ,	0	12	. 10₹	357	828	1,175	1,355	1,126	\$39	7465	208	42	6,617	
	Şcothbluff.	0	0	138	459	876	1,128	1,231	800, 1	', 921	552	283		6,673	4
	Valentine	9	12	165	493	r 942	1,237	1,395	1,176	1:045	`5 79	288	84	7,425	
	- I Iko	9	14	225	561	924	1,197		1,016	911	621	409	192	7,413	
		28	. 43	234	592	939	1,184	•	1,075	977	.632	456	225.		
	Las Vegas Reno	. 0	. 0	10 204	. 78 490	387 801	61-7		487	335	111		0	2,709	
	Winnemucca	,0	87 34.	210	116	876	1,026	1,071	\ 823° 916	729 837	510 573	357	189	6,312	
NH	Concord	.6	59	177	505	822	1,240		1,184	1,032	636	298	. 75	7,383	
	MI, Wash Osby.	493	536	720	1,017	1,341	1,742		1 663	1,652	1,260	9 10		13.817	
N.	*Atlantic City	0	, 0	. 19	251	549	250		848	741	4 20	133	15.		
	Newark	1 0	Đ	10	248	575	921	, 981	876	129	351	115	0	4,859	
•	Trenton	· * 0	. 0	. 57	264	576	924	989	7887	753	399	1 2 1	12	4,980	,
NM		. 0	- 0	. 12	₽ 229	642	868	9 10	703	595	288	8 1.	0	4,148	
	Clayton	0	6	66	310	699	899	986	812	` 747	429	183	.21	5,158	
	Raton	9	28	726	431	825	1,048	1,116		- 834	543	301	₩3	6,278	
,	Roswell 4	,0	. 0	18	202	573	806	840	641	481	201	-	. 0	3,791	
NY	Silver City = = Albany	0	. 0	. 1 18	783 440	525 777	729		601	518	261	87	0	1,701	
14.1	Binghamton (AP)	. 22	65	201	471	810	1,194	1,311	3,156 1,154	992	. 564	239° 313	45	6,875 7,286	
	Binghamton (PO)			141,	406	777	1,107	1,190	1,081	949	543	229	45	10411	
	Buffalo	19	37	141	440	777	1,156	1,256	1,145	1,039	645	329	78	7,062	
•	· Central Park	• 0	0	30	233	5400	902	986	U 5	760	408	118	. 9	4,871	
,	'I F Kennedy Intl.	0	0	36	248	\$64	933	1,029	935	815	480	167	12	5,219	
	t aguardra	0	0	21	223	528	887	971	879	750	414	124	6	4,811	
•	Rochester	e9	1.31	126	415	.747	1,125	1,234	1,123	1,014	597	279	48	6,748	
	Schenectady	0	22	123	422	756	* 1,159	1,283	1,3.11	970	543	211	36	6,650	
	Syracuse	6	78	132	415	744	1,153	1,271	1,140	1,004	570	248	45	6,756	
NC	Asheville	σ	0	'48	245	\$55	775		683.	592	273	87	٠,0	4,042	
•	Cape Hatteras	0	0	0	78		4 521	580	518	440	177	25	0	2,612	
	Charlotte Greenshoro	0	0	, 33	124 192	438 513	691 778	691 784	582 672	481 552	156 234	47	0	3,191	
	Ratergh	0	Ö	31	164	450	716	725	616	487	180	34	0	3,805 3,341	
_	Wilmington /	Ö	. 0	Ö	, 74	291	521	546	462	357	96	0	0	2,347	
.*	Winston Welem	ō	. 0	21.	171	483	747	753	652	524		17	ō	1,494	
' ND	Britmarick	34	28	222	, 477	1,083	1,463	1,708		1,203	645	3 29	117	8,851	
	Devils Lake .	48	51	271	642	1,191	1,614	1,872	1,979	1,345	753	381	1 18	9,901	
	f argo	28 ,	1 37	219	574	1,107	1,569	1,789	1,520	1,262	690	332	90	9,226	
	Williston	31	4 5	261	601	1,122	1,515	1,758	1,473	1,262	681 -	- 357	141	9,243	
ОН	Akron	0	9	96	381	126	1,070	1,118	1,016	€71	459	207-	19	6,037	
	Cincinnati	0	ď	54	248	6 t 2	921	970	817	701	336	115	. 9	4,806	
	Cleveland . 1	9	25	105	184	718	1,088	1,159	1,047	918	552	3,60	66	6,351	
,	Columbus	0	6	84	147	714	1,019	1,088	949	1809	426	171	27	1,640	
•	Mansheld	9	. 22	JR 114	110	696 768	1,045 1,110	€1,097	955	#09	429	167	30	1,6 /7	
	Sanduska	0	Ä	66	313	684	1,032	1,164	1,042	9 24 868	543 495	245 198	50 36	6,401 5,796	
	Toledo	oʻ	160	117-	406	792	1,138		1,056	924	543	242	60	6,494	
	Youngslown '	6	19	120	412	#21	1,104	1, 69	1,047	921	540	248	60	6,417	
OK	Oklahoma City	0	0	15	164	498	766	1168	664	527	189	34	0	3,725	
	Tulsa	0	6	18	158	522	° 787	119 3	583	1539	21.3	47	- 0	1,260	
OR	Mstoria .	146	130	210	- 375	561	679	5.3	622	636	480	363	181	5,186	·
•	Burns	12	37 .		515	. 267	1,113	1,246	988	856	. 370	366	177	6,957	
	Eugene	34	34	129	366	\$85	719	E03	627.	589	426	279	135	4,726	
	Meacham	. 14	124	288	-580	918	1,091	1,209	1,005	983	726	\$27		7,874	
	Medford Pendleton	^ 0	. 0	78 '		678	871	918	697	. 642	432	242		5,008	
	Postland	25	28	113 114	350 335	711 597	, 884 , 735	1,017 .825	773 644	617	396	205	63	5,127	
	Roseburg	22	- 16	105	129	567	713		608	5 86 570	396 405	, 245 267	105 123	4,635 4,491	
	Selem	37	31	43 11	338	594	729	822	647	6111		273		4,754	
	Sexton Summit	81	81	771		666	874	958	109	818	509	465		6,254	
PA_	Allentown.	΄Ο,	0 '	90	353	693	1,045	1,116	1,002	849	471,		24	9,810	
	Erie * 5	<i>/</i> L	25	102	391	714	1,063	1,169	1,021	973	585	282	60	6,451	
	Harrisburg	, ō	8. þ	67	298	1648	992	1,045		766	396	124	12	5,251	
	Philadelphia .	. 0	0	60	291	21	.964	1,01%	290	744		7 115	12	56 01	
•	Pitrsburgh	0	9	105	, 375	726	1,063	1,119	1,002	874	480	195		5,987	
4	Reading	. 0	. 0	54	257	1597 763	939	1,001		735	372	, +05	0	4,945	
. '	Scranton Williamsport	0	19	132 111	434 375	762	1,104		.1,028	893	.498		- 33	6;254	
Ri S	Block ds.	. 0	16	78 .		717 594 .	1,073	1,122	1,002 955	1 856 877	46 8. 612	377	24	5,930	
	Providence	9	16	96	372	660	1,023	1,11	977 988	161	534	344 • 236	<i>5</i> 4.	\$,804 '5,954	
SC	Charlestop	-	٠.٥	. 6	59	287	471	487 5	389	291	54		, 0,	2,033	
	Columbia		, ō	0	84 .	345	\$77	570	470	357	₩.		0	2,484	
	-	•		, -			, •				₩.	. "	4	p, -=-	

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	Location	Jul	Aug	Sep	Oct	Nov	Lieu	jan	Feb	Mar	Apr.	May)un	Ann	
S C.	Florence	0	0	 Q	78	315	552	552	. 49	347	84		- 0	2,387	,
	Linterrytite o	0	`,0	0 -	117	367	636	448	535	434	120	12	0	2,884	
	Literville () Apartablise ()	- · ·	. 0	15	1 10	417	667	663	\$60	453	144	25	0	3,074	
30	Hurun	4	1.7	1=1	108	1,014	1,412	1,628	1,355	1,125	104	188	27	8,223	
	Rapid City	22	-12,	101	481	897	1,172	1,333	1,145	1,051	615	376	126	7,345	
	Sious Falls	19	25	164	462	972	1,361	1,544	1,285	1,082	573	270	78	7,839	
ŢN		. 0	Ō	51	4	571	828	#78	700	594	261-	44	0	4,143	
	f hattaninga	· U	U	1 16	141	444	F.78	12:	511	453	150	15	0	3,254	
	Knowith	0	U	en en	171	489	125	132	613	493	198	43	0	3,494	
	Memphis ,	0	0	i a	1 10	447	948	729	585	456	147	22	0	3,232	
	Nahville	0	0	30	158	49,5	732	778	544	5)2	189	40	0	3,578	
	Oak Ridge (CO)	0	0	. 39	192	531	772	7.28	665	552	228	56	0	3,817	
TX	Abilene	0	0	0	99	366	486	642	470	347	114	0	0	2,624	
•	Amarillo	, 0	0	18	205	570	797	877	* 664	346	252	56	9	3,985	
	Austin	, 0	0	. 0	31	225	388	468	325	223	51	Q	0	1,711	•
	Brownsville	0	0	. 0	0	64	149	205		74	0	0	0	600	
	Corpus Christi	0	9	` 0	0	1 20 321	220	6 291	174	109	0	0	0	314	
- 1	Dallas Li Paso		0	0	62 84	414	524 648	601	440 445	319 319	90 105	. 0	_	2,363	
	ft Worth	0	0	0	. 65	324	536	685 614	. 445	319	, 99	. 0	0	2,405	
•	Galveston	. 0	0	-0	· 0	138	270	350	258	189	30	0	0	1,235	
	- Houston	. 0	` "	. 0	. 6	183	307	384	234	192	36	0	Ö	1,396	
	Laredo	- 0	, 0	0	. 0	105	217	267	134	74	70	0	ő	797	
	4 Lubbock -	0	0	_18	174	513	744	800	613	484	201	31	Ö	3,578	
	Midland	ő	. 0	0.	87	381	592	651	468	322	90	ô	. 0	2,591	
	Port. Arthur	· 0,		0.	22	1 207	. 329	384	274	192	39		0	1,447	
	San Angelo	•0	0	Ü	61	3 314		567	4472	288	66	ñ	ō	2,255	
	San Antonio	ó	ō	ō	31	207	363	428	286	195	39	Ŷ	ō	1,549	
•	Victoria	Ö	0	. 0	6	150	270	344	230	152	21	. 0	ō	1,173	
- 1 X	WALO	• 0	ō	0	43	- 270	456	536	389	270	66	. 0	0	2,030	
	Wichija Fails	0	o	0	99	381	632	698	518	378	120	6	Ō	2,832	
UI	Millord	. 6	٥	99	443	867	1,141	1,252	. 948	822	519	279	87	6,497	
•	Salt Lake City.	· _0	0	81	419	849	1,082	1,172	910	763	459	- 233	84	6,052	
	Wendover	₹0	0	48	372	822	1,091	1,178	902	729	408	177	51	5,778	
VΙ	Burlington	28	65	201	539	198	1,349	1,513	1,333	1,187	714	353	90	8,269	
VA	Cape Hertry	0	0	0	- 112	360	545	694	633	* 536	246	53	0	3,279	
	Lynchburg	0	ο.	. 51	723	540	822	849	731	605	267	78	0	4,166	
	Nortolk	٠, ٥	0	0	1 36	408	698	7,58	655	-533	216	17	. 0	3,421	
•	Richmond 1 1	. 0	a 0	16	214	495	784	813	703	546	219	53	0	3,865	
	Ryanola	0	0	51	229	549	825	834	722	614	261	65	0	4,150	•
	Wash Nat'l, Ap '	٠ ـ ٥	0	31	24.7	519	8 34	£71	762	626	, 288	, 74	0	4,224	
WA	City in para	. 464	. 71	198	*422	636	m, 753	834		645	450	307	177	5,236	
	Seattle +	50	47	1 29	, 139	543	657	738	599	577	196	242	117	4,424	
	Seattle Boeing	34	40	147	384	624	763		655	408	411	242	94	4,838	
	Seattle Tacoma	\$6 9	62		, 391	611	750	828	678	657		295	159	5,145	
	Spub ane	_	24	162		L 879	1.082	.1,231	980	834	531	788	135	5,655	
	Stampfide Pass	274	144	\$*E \$	/01	1,(30)4	L 178	1 11 24 1	1,075	1,085	#55	654	481	9,281	
	Faluesh Is.	245	1/4	1614,	406	114	6.14	711	611	645	121	431	133	5,/19	
	Walla Walla	0	0	87	110	681	843	486	745	589	342	177	45	4,805	
	Yakima	0	12	144	450	828	1.039	1,161	261	713	. 435	220	69	5,941	
4 WV	Charleston	0	0	63	254	591	865	880	770	648	300	. 96	9	4,476	
	Elkins		43	135	400	129	992	1,002	576	797	444	198	48	5,675	
	Huntington	. 0	س.	63 60	257 284	585	856	880	764	616	294	99	12	4,446	
	Parkersburg	. 0	0				905	942	\$ 26	691	339	115	6	4,754	
WI	Green Bay	28	50 19	174	484	924	1,333	1,494	1,313	1,141	654	335	99	8,029	
	t a Crosse Madison	12 25	.40 -	153 174	437	924	1,139	1,504	9,211	1,070-	540	245	69	7,589	
	madison Milwaukee	43	47	174	473	930 876	1,330	1,473	1,274	1,113	618	310	102	7,463	
 .	(asper	4 ,	16	192	524	942	1,252	1,376	1,193 1,0 8 4	1,054	547	372	135	7,635	
	Chevenne .	19	31	210	543	924	1,101	1,290	1,056	1,029	657	381	129	7,418	
	Lander	6	19	204	555	1,020	1,299		1,036	1,011	672 654	, 381 381	102 153	7,278	
~ ~	Sheridan	25	31		. 539	948	1,200		1,154	1,054	642	. 366	150	7,870 7,683	
	2 Mari				,,,				1,124	1 14 24	of a n		, ,0	1,447	

One of the most practical of weather statistics is the "heating degree-day." First devised some 60 years ago, the degree-day system has been in quite general use by the heating industry for more than 40 years.

I the ting degree-days are the number of degrees the daily average temperature is below 65°F. Normally heating is not required in a building when the outdoor average daily temperature is 6°F. Heating degree-days are determined by subtracting the average daily temperatures below 65°F from the base, 65°F. A day with an average temperature of 50°F has 15 heating degree-days (65°50° 15) while one with an average temperature of 65°F or higher flas none.

Several characteristic make the degree-day figures especially useful. They are cumulative so that the degree-day sum for a period of days represents the total heating load for that period. The relationship between degree days and fuel consumption is finear, i.e., doubting the degree-days usually doubles the fuel consumption. Comparing normal semanal degree-days in different loading in Chicago, Ill., where the mean annual total heating degree-days are about 6,200 then to heat a similar building in New Orleans, i.e., where the mean annual total heating degree-days are about 6,200 then to heat a similar building in New Orleans, i.e., where the mean annual total heating degree-days are about the tamb whether the 100 degree-days occur in only 3 of 4 days or are spread over 7 or 8 days.

Appendix B-4 Mean Daily Solar Radiation (Langleys*)

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	Sale Lake City	163		256		354		479	8	570	7	621	7	6 20	6	551	7	446		316 281	2	202	2	168	2	
	Mt. Weather	172	2	274	2	338	2	414	2	508	2	525	3	510	3	430	3	375	. 3	281	3	122	3	77	3	
	North Head		-	167	2	257	3	432	2	,509	3	487	3	486	3	436	3	321	,	194	10	102	10	75	8.	
	Friday Harbor	8 7	8	157	7	274	8	418		514	9	578 -	. 10	586	10	507	11	351 458	-	274	4	136	4	100	4	
	Proser	117	4	222	4	351	. 4	521	5	616	4	680	. 4	707	. 4	604	•	410	7	245	į	146	*4	96	. 5	
	Pullman	121	4	205		304	2	462	2	558	4	653	- 4	699	5	562	-	299		170		93	ų	19	9	
	University of Washington	67	9	126		245	- 10,	,	9	445	10	461	10	496			10	324	10	_	10	104	9	64	10	
	Seattle Tacoma	75	9	139	9	265	4		9	101.	9	511	9	566		452	10	324 404	10		9	131	4	75	•	361
	Sookane	119		294		321		474	9	563	9	196	9	665			-	404 348	47		47	145	44	115	46	
	Madison	148	46	220	46	313	45	394	47	466	47	514	47	5.31	47	452	47	•	•	141	-	239	9		9	
	Lander	226		324	. 9	452	9	548	11	58	11	678	11	651	11	-	10	472			_	239	ī	186	4	403
	Laramie	216	3	3 295	3	424	3	106	3	554	~,	643	3	606	3	5 36	3	418	.)	744	,	***	•	***	-	
High	d Stations	,							_		_			***		597		440	. 4	451	9	600		572		597
	Canton Island	588		626		634	7	604	9	561	7	549		550				531		460	-	411			6	512
	San Juan, P.R.	. 404	5	5 481		580	4	622	4	519	5	536	•	639	•	-	7	535				394	8	-	i	526
	Swan Island	442		6 496	_	• • • •	6	•	6	6 25 642	ę,	544	•	588 679		-	,					482	_	421	,	
	2mSU Desire			7 518	. 7		7	627	7			456						7 . .	-	/ *= -		~ 				

Appendix B-5

Solar Position and Insolation Values

24 Degrees North Latitude

	Date	So		So posi	lar tion	έτι	JH/sq. ft	. total	insolati	on on s	urfaces	
	*	AM	PM	Alt	Azm				South fangle	acing su with he		
	•			-		Normal	Horiz.	14	24	34	44	90 '
		-	ا ۽ ا	4.8	65.6	71	10	17	21	25	28	31
1	jan 21	7 8.	5	16.9	58.3	239	83	110	126	137	145	127
١		9	3	27.9	48.8	288	151	188	207	,221	228	176
		10	2	37.2	36.1	308	204	246	268	282	287	207
1		ii	-	43.6	19.6	317	237	283	306	319	324	226
		┝──	2	46.0	0.0	320	249	296	319	332	336	232
		Su	rface	dally t	otals	2766	1622	1984	2174	2300		1766
Γ	Feb 21	7	. 5	9.3	74.6	158	35	44	49	53	56	46
	- ·	8	4	22.3	67.2	263	116	135	145	150	151	102
		9	3	34.4	57.6	298	187	213	225	230	228	141
1		10	2	45.1	44.2	314	241	273	286	291 328	287 -323	168
		11	11	53.0	25.0	1	276	310 323	324	341	335	191
` _	<u>. </u>	—	12	56.0	0.0	3036	1998	2276	├	2436	2424	1476
-		7	_	13.7	83.3	T	60	63	+	62	59	27
1	Mar 21	8	4	27.2	76.8		141	150		149	142	64
		9	1 .	40.2	67.9	1	212	226	229	225	214	95
1		10		52.3			266	285		283	270	120
١,		111	1	61.9	33.4	315	300	322	326	320	1	135
	_		12	66.0	. 0.0	317	312	334		333	317	140
		S	urfac	e daily	totals	3078	2270	2428	+	∤	2298	1022
	, spr 21	6	6				7			1	51	· 2
١	•	7					83	7	L			16
- }		8				1	160 227	15	3			46
		9				1	278		ı	1	1	61
٠ [10	•	4			310	1	- 1 · · ·			74
١		\ ''	12	77.	1	ì	321	1	i	1	280	79
+		5		e dally	<u> </u>	3036	2454	245	8° 2374	2228	2016	488
•	May 2	1 4	5 6									5 12
-		- 1	7 5	1				1 .				1
-			8 4				-			1		3
١		١,	9 3	3 48. 2 62.					1	1	1	1
	:	1		75.		4	1 .	3	5	4	240	1
-	i 		12	86.	- 1	0 288	322	2 31	7 30	+		+
		!	urfa	ce dall	totals	3032		+		-+	+	
	Jun 2	10		6 9.		6 9				2 1 5		
		1		5 22.				,		· ·	· .	
		1		4 35. 3 49.								
	ļ.	1.	9	3 49. 2 62.		ı			9 25		1	18
	•		,	1 76		- 1			00 28	1		7 19
		H	12	89)	.0 28	}	i	ত 29	1 .	9 230	5 22
	,	+		ice dall	y totals	299	4 257	4 24	22 223	0 199	2 1 70	204

Date	So tin			ition	- BTI	JH/sq. f	t. total	inśolat	ion on	surfac	es .
	AM	PM	Alt	A/m					facing : with f	surface noriz.	•
	,				Normal	Ησειζ.	14	24	34	44	90
Jul 21	6	6	8.2	109.0	81	23	16	11	10	9	6.
·	7	5	21.4	107.8	195	98	85	73	59	44	13
	8	4	34.8	99.2	239	169	157	143	125	104	16
	9	3	48.4	94.5	261	231	221	207	187	161	18
	-10°	2	62.1 75.7	89.0 79.2	272 ~ 278	278 307	270 302	256 ⁻ 287	235 265	206 235	21 32
	1	 2	86.6	0.0	280	317	312	298	275	245	36
			daily t	<u> </u>	2932	2526	2412	2250	2036	1766	246
A 21	\vdash				· ·	 	-		-		-
Aug 21	6	6 5	5.0 18.5	101.3 95.6	35 186	82	5 76	69	60	50	11
•	8	4	32.2	89.7	241	158	154	146	134	118	16
13	9	3	45.9	82.9	265	223	222	214	200	181	39
	10	2	59.3	73.Ò	278	273	275	268	252	230	58
	. 11		71.6	53.2	284	304	309	301	285	261	,71
	1	2	78.3	0.0	286	315	320	313	296	272	75
	Sur	face	daily t	otals	2864	2408	2402	2316	2168	1958	470
Sep 21	- 7	5	13.7	83.8	173	- 57	60	60	59	56	26
	8	4	27.2	76.8	248	136	144	146	143	136	62.
,	9	3 -	40.2	67.9	278	205	218	221	217	206	93
	10	2	52.3	54.8	292	258	275	278	273	261	116
	11	<u> </u>	61.9 66.0	33.4 0:0	2 99 •	291 302	311	315 327	309 321	295	• 131 • 136
·				otals .	2878	2194	2342	2366	2322	2212	992
-	<u> </u>		 ,	,					<u> </u>		
Oct 21	7 8	5	9.1 22.0	74.1 66.7	138 247	32	129	139	48 144	50 145	42 99
	9	3	34.1	57.1	284	180	206	217	223	221	138
	10	2	44.7	43.8	301	234	265	277	282	279	165
	11	1	52.5	24.7	309	268	301	315	319	314	182
•	1	2	55.5	0.0	311 :	279	314	328	332	327	188
•	Sur	face	daily t	otals	2868	1928	2198	2314	2364	2346	1442
Nov 21	7	5	4.9	65.8	67	10	16	20	24	27	29
	8	.4	17.0	58.4	232	82	108	123	135,	142	124
	9	3	28.0	48.9	282	150	186	205	217	224	172
,	10	2	37.3	36.3	303	203	244	265	278	283	204
1.	11	1	43.8 46.2	19.7	312	236 247	280 293	302	316	320	222
	·		daily t	0.0	315 2706	1610	1962	2146	328 2268	332 2324	228 1730
Dec 21	.7	5	3.2	62.6	30	3	7	9	11	12	1/30
200 21	8	4	14.9	55.3	225	71	99	116	129	139	130
	<u>`</u> 9	3	25.5	46.0	281	137	176	198	214	223	184
	10	2	34.3	33.7	304	189	234	,258	275	283	217
	11	1	40.4	18.2	314	221	270	295	312	320	236
	. 1	2	42.6	0.0	317	232	282	308	325	332	243
	Sur	face	daily t	otals	2624	1474	1852	2058	2204	2286	1808

1 BTUH/SQ. FT. = 3.152 W/m²
NOTE: 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1.0 clearness factor.
2) See Fig. 4, p. 394 in [3] for typical regional clearness factors.
3)- Ground reflection not included on normal or horizontal surfaces.



Date	Sol			lar ition	BIL	11/94 11				** *	, ,
	AM	PM	Alt	A/m			7.	ا طابدر م ایوم	with t	411/ - -	
		,	\vdash		Normal	Menis	72	37	4%	37	W/ .
			, i				6	. 4	1 1	,	1
Jan 21	7	5	1.4	65.2 36.5	203	54	93	1///,	114	175	115
	9	3	12.5 22.5	46.0	269	118	175	193	2114,	212	181
	10	1 2	30.6	33.1	295	167	235	25%	21/1	2/4	111
	11	1	36.1	17.5	306	198	273	29%	\$1 1 %	317	
	1	2	38.0	. 0.0	310	209	285	308	$=\frac{321}{2}$	174	27.5
	Sur	face	daily i		2458	1288	1839	21/08	2118	2164	1//9
Feb 21	7	5	7.1	3:3.5	121	22	34	37	48	*42	54
100 11	8	4	19.0	- C-C	247	95	127	136	140	220	1/4
	9	3	29.9	53.4	288	161	206	217 278	222	275	
4	0	2	39.1	39.4	306 315	212	266 304	317	321	315	
	<u> 1</u>	Ţ 1	45.6 48.0	0.0	317	255	316	3:10	334	124	222
<u>·</u>	+	2			2872	1724	2188	2300	2345	2322	1444
	+	T	daily	\$1.9	185	54	60	60	59	34,	
Mar 21	7 8	5	12.7 25.1	3	260	129	146	147	144	150	/1
,	9	3	36.8	62.1	290	194	222	224	2210		111
	10	2	47.3	47.5	304	245	280	283 321	278	24.5 344)	150
1	11	1	55.0	26.8	311	277	317		315	1	1
·	╄	2	58.0	0.0	313	287	329	333	327	317	1//
	58	riáce	daily i		3012*	2084	2378	2403	2358	2244	•
Apr 21	6	6	6.1	99.9	66	14	75	11	4	1 50	1 5 14
, .	8	5	18.8	92.2 84.0	206 255	158	134	١	154	176	54
	9	3	43.9	74.2	278	220	225	217	20%	186	1,8
	10	2	55.7	60.3	290	267		272	254	214	7:
	11	1	85.4	37.5	295	297	313	. 306	290	71,5	117
	1	2	69.6	0.0	297	307	325	318	301	2/4	118
	Sui	face	daily t	otals	3076	2390	2444	2356	2706	1994	764
May 21	6	6	10.4	107.2	119	361		13	13	1.12	. 7
٠	8	5	22.8 35.4	100.1 92.9	211 250	107 175	88 159	75 145	60 127	105	13
•	9	3	48.1	84.7	269	233	223	209	188	163	1 <u>5</u> 33
	10	2	60.6	73.3	280	277	273	259	237	208	
	11	1	72.0	51.9	285	305	305	290	268	237	72
		2	78.0	0.0	286	315	315	301	278	247	77
	•	į.	l	otals	3112	2582	2454	2284	2064	1788	469
Jun 21	6		12.2	110.2	131	45	26	16	15	1.4	9
•	7 8	5 4	24.3 36.9	103.4 96.8	210 245	115	159	76 143	59 122	41 99	14 16
	9	3	36.9 49.6	89.4	264 >	8	221	204	181	153	19
	10	2	62.2	79.7	274	279	268	251	. 227	197	نه
	11	1	74.2	60.9	279	306	299	282	257	224	56
	,		1		200	216	200	292	267	224	60
, ·	1	2	81.5	0.0	280	315	309	,272	207	234	60

185

ERIC Total Provided by ERIC

Date	So			lar tion	ВТ	JH/sq. ft.	total	insolati	on on :	surfaces	
	AM	РМ	Alt	Azm				South fa	acing si		
					Normal	Horiz.	22	329	42	52	90
Jul 21	- 6	6	10.7	107.7	113ر	37	22	14	13	12	8
	7	5	23.1	109.6	203	107	87 158	75	60	44	14
	8	4	35.7	93.6	241	174	220	143	125 185	104	16 31
1	10	3 2	48.4 60.9	85.5 74.3	261 271	231 274	269	254	232	204	54
ļ	n	î	72.4	53.3	277	302	300	285	262	232	69
		2	78.6	0.0	279	31 1°	310	296	273	242	• 74
	Su	rface	daily t	otals	3012	2558	2422	2250	2030	1754	458
Aug 21	6	6	6.5	100.5	59	14	9	7	6	6	4
Į -	7	5	19.1	92.8	190	85	77	69	60	50	12
1	8	4	31.8	84.7	240	156	152	144	132 197	116 178	33 65
	9	3	44.3	75.0	263 276	216 262	220 272	212 264	249	226	91
•	10	1	56.1 66.0	38.4	282	292	305	298	281	257	107
	<u> </u>	2	70.3	0.0	284	302	317	309	292	268	113.
	Su	rface	daily t	otals	2902	2352	2388	2296	2144	1934	736
Sep 21	7	5	12.7	81.9	163	51	56	56	55	- 52	30.
	8	4	25,1	73.0	3	124	140	141	138	131	75
1	9	3	36.8	62.1	272	188	213	215	211	201	114
	10	2		47.5	287	237	270	273	268	255	145
•	111	1 1 1 2	55.0	26.8	294 296	268 278	306 318	309 321	303 315	289 300	164 171
-			daily 1	-	2808	2014	2288	2308	2264	2154	1226
Oct 21	┿—	5	6.8	73.1	99-	19	29	32	34	36	32
Oct 11	7,7	1	18.7	64.0	1 -	90	120		133	134.	104
	9	3	29.5	53.0	1	155	198	208	213	212	153
	10	2	38.7	39.1	293	204	257	269	273	270	168
1	11	1	45.1	21.1	302	236	1294	307	311	306	209
<u> </u>		12	47.5	0.0	304	247	306	320	324	318	217
9	Su	rrface	daily	totals ·	2696	1654	2100	2208	2252	2232	1588
Nov 21	7		1.5			0	0	1	1	1	(1
	8		12.7			55	91		113	119	111
	9		22.6			118	173		202 265	208	176 217
	10		30.8			166	233		303		241
	111	12	36.2 38.2	1	[207	282	ì	316	i	249
	-+		daily	-	2406	, 1280	1816	 	2084	 	1742
Dec 2	+	\top	10.3	_		41	77	+	101	108	107
Jaz	' ` 9		19.8			102	161	r	195	204	183
	10	•	27.6		3	150	221	244	259		226
	11		32,7		3	180	258		1	1	251
		12	34.6			190	271	-ł	311	318	259
1	S		dally	totals	2348	1136	1 704	1888	2016	2086	1794

1 BTUH/SQ. FT. = 3.152 W/m²
NOTE: 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1.0 clearness factor.

2) See Fig. 4, p. 394 in [3] for typical regional clearness factors...

3) Ground reflection and inclined on normal or horizontal surfaces.

150

40 Degrees North Latitude

·	Date	Sol			olar ition	ВТ	JH/sq. ft	, total	insolatio	on on s	urfaces	
		AM	PM	Alt	Azm	·		!	South fa	cing su		•
						Normal	Horiz.	30	40	50	60	90
	1 21	8	4	8.1	55.3	142	28	·65`	74	81	85	84
	Jan 21	9	3	16.8	44.0	239	83	155	171	182	187	171
		10	2	23.8	30.9	274	127	218	237	249	254	223
	,	11	<u> </u>	28.4	16.0	289	154	257	277	290	293 306	263
L			2	30.0	0.0	294	164	270	291	303	+	
L		Su	rface	daily		2182	948	1660	1810	1906		1726
Γ	Fcb 21	7	5	4.8	72.7	69	→ 10 73	19	21 122	23 126	24 127	22 107
-		8	4 3	75.4 25.0	62.2 50.2	224	132	195	205	209	208	167
		10	2	32.8	1	295	178	256	267	271	267	210
-		11	1	38.1	18.9	305	206	293	306	310	304	236
			12	40.0	0.0	308	216	306	319	323	317	245
	,	J. Su	rface	dally	totals	2640	1414	2060	2162	2202	2176	1730
1	Mar 21	7	5	11.4	80.2		46	55	55	54	` 51	35
	•	- 8		22.5			. 114	140		138	131	89 138
	•	9	3	32.8	1		173	215		213	°25	176
		10	1 2	41.6	1		247	310	,	307	3	200
1		-	1: 12	50.0	ı		257	322	1	320	305	208
ŀ					totals	2916	1852	2308	2330	2284	2174	1484
H	Apr 21	+-	1	7.4	T -	1	20	11	8	- 7	7	4
-	Api 21	7	1 5	18.9	,		87			61	50	12
		8	4	30.			152			133		53 9 3
-		9		1			207 250			1)	126
		10		1 -		•	277		1	285		147
		- <u>-</u>	12	61.	i'	- 1	287	320	313	296	271	154
t		S	urfac	e daily	totals	3092	2274	241	2 2320	-2168	1956	1022
-	May 2	+-	7		9 114.	7 1	0		0 0		0	0
	riet Z		6 6	12.	7 105.	6 144						13
			7 5									25
			3 4				1				,	60
		1		1	1		267	7 27	0 255	233	205	
		l		66		1 283	193	1	l l	1	1.	!
.			12	70	.0 0.	0 284	301	-+	- 	+	+	+
		5	urfa	e dall	y totals	3160	2552	2 244	- 	+	+	+
	Jun 2				2 .117.			,			2 2 7 16	
		- 1		5 14				,	0 1	1 -		
		- 1		5 26 4 37	.0 99 .4 90			1			1 97	16
					8 80	.2 263	3 23	a· 21	9 :20	2 179	9 151	
			0	2 59	.8 65	.8 272						
		1		1 69						, -		
		_	12	73	- 1 + -	0 279		_+-	" 			+
	1		Surfa	ce dall	y totals	318	0 264	8 243	34 222	4 17/	7 10/1	, 010

Date	Sol tim	•		lar tion	вті	JH/sq. ft	, total i	nsolatio	on on s	urfaces	•
	AM	РМ	Alt	Azm			s	outh fa			
1					Normal	Horiz.	30	40	50	60 、	90
Jul 21	5	7	2.3	115.2	2	0	0	0	0	. 0	0
,	6	6	13.1	106.1	1.48	±50	26	17	15	14	9
	7	5	24.3	97.2	208	114	89	°75	60	44	14
1	8	4	35.8	87.8	241	174	157	142	124	102	24
1	9	3	47.2	76.7	259	225	218	203	182	157 -	58
ļ	10	2	57.9	61.7	269	265	266	251	229	200	86
Į.	11	1_	66.7	37.9	275 c	290	296	281	258	228	104
	. 1	2	70.6	0.0	2/6	298	307	292	269	238	111
	Su	face	daily	otals	3062	2534	2409	2230	2006	1728	702
Aug 21	6	6	7.9	99.5	. 81	21	12	9	8	7	5
	7	5	19.3	90.9		-87	76	69	60	49	12
	8	4	30.7	79.9	237	150	150	141	129	113	50
	9	3	41.8	67.9	260	205	216	207	193	173	89 120
•	10	2	51.7	52.1	272	246	267	259 292	244 276	252	140
	11	11	\$9.3	29.7	278	273	300	•			
	. 1	12	62.3	0,0	280	282	311		287	262	147
<u> </u>	Su	rface	daily	totals	2916	2244	2354	2258	2104	1894	978
Sep 21	7	5	11.4			43	51	'51	49	47	32 84
	8	4	22.5			109	133	134	131 203	124	132
	9	3	32.8			167	206	208 265	260	247	1.68
	10	2	41.6			211	262 298	301	295	281	192
	11	12	47.7 50.0	1	1	249	310	313	307	292	200
	 -		daily		2708	1788	2210	2228	2182	2074	1416
. 174	+	T .	4.5	т —		1 7	14	15	17	17	16
Oct 21	8	4	15.0	1		68	10€	113	. 117	118	100
	9	1 '	24.4	1		126	185	1	200	198	160
	1.0	١.	32.4		I.	170	245		261	257	203
•	111		37.6		5	199	283	295	299	294	229
	\vdash	12	39.5	1		208	295	308	312	306	238
	+-			totals	2454	1348	1962	2060	2098	2074	1654
Nov 21	8	4	8.	2 55,4	36	28	63	72	-78		81
1404 21	, 9		17.0		232				178		167
1	10	1				126	215	233			219
] 11		28.				254	273	285	288	. 248
		12	30.	i i		163	267	287	298	301	258
	5	urfac	g daily	totals	2128	942	1636	1778	1870	1908	1686
Dec 21	1 8	3 4	5.	5 53.	0 89	14	39				.56
	و ا		,						,		163
-	10	,	, -	,		107	1-200	221			22
}	1	1		,			23	9 262	276	5 283	25
		12	26.	i		14	3 25	3 27	290	296	26
	+		العصوة		1978	3 783	2 148	0 163	J 74	1706	164

1 BTUH/SO. FT. = 3.152 W/m²
NOTE 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1.0 clarness factor.

2) So Fig. 4, p. 394 in [3] for typical regional clearness factors.

3) Ground reflection not included on normal or horizontal surfaces.

48 Degrees North Latitude

	Date	So	iar ne		olar Ition	BTI	JH/sq. fi	. เอ็เล	insolati	on on s	urfaces	
		AM	PM	Alt	Azm	·		•	South f	acing so with he	urface orlz	
Į.		-	-			Normal	Horiz.	38	48	58	68	90
					4 54.6	37	4	17	19	21	22	22
1	jan 21	8	3	3.5 11.0	42.6	185	46	120	132	140	145	139
		10	2	16.9	29.4		83	190	206	216	220	206
		11	1	20.7	15.1	261	107	231	249	260	263	243
		—	2	22.0	0.0	l .	115	245	264	275	278	255
+		Su	rface	daily -	totals	1710	596	1360	1478	1550	1578	1478
\vdash	Feb 21	7	5	2.4	F/	12	1	3	4	4	4	4
1	F 60 21	8	4	11.6			49	95		105	106	96
1.		9	3	19.7	1	1	100	178		191	190	167
1		10	2	26.2			• 139	240		255	251	217
1.		13		30.5		290	165	278	١.	294	288	247
1		_	12	32.0	0.0	293	173	291	304	307	301	°258
T		Si	ırface	dally	totals	2330	1080	1880	1972	2024	1978	1720
T	Mar 21	7		10.6	T	1 153	37	49			45	35
1	(4160 7 -	8		19.5			96	131			122	96
ı		9		28.2			147	205			193	152
-	ζ	10		35.4	37.1		187	263			248	195 223
- 1		11	-1	40.	3 19.	8 295	212	300	303		283	Fr I
-			12	42.0	0.0	0 298	220		+-	 	294	\232
	*	5	urfac	e daily	totals	2780	1578		+		+	1632
1	Apr 21	1 6	6	8.	6 97.					1		5
- 1	, tp. a.		1 5				. 85	F				21 69
-			1-4	28,	5 74.			,		1 -	113	
١			9 -3							245	223	
4		91	0 2					,		1		
- 1		1	1 1	51.	5 24.	1	,	Į.	i	1		.~
			12	53.	6 0.	0 288		- - -		+	+	+
		- 5	urfac	e dail	y totals		_		<u> </u>	+	- 	
- [May 2	1	5 7	5	2 114			2		رو جاء	6 3 5 13	10
			6 6		1 .				17 1			13
,			7 5					3	19 7 16 14	- 1		ı
				34					7 20		- 1	
				3 44								
				2 53			2	,	26 28	* 1		1 .
		91		_	.5 28	.6 279 .0 28	1	1 .	06 29	1	1	1
		+	12 Surfa		y totals				_		-+	982
		+		~~~		- 1	- 	1	9		8	7 5
	· Jun S	21	- 1		.9 116 .2 106	1	· 1				8 1	3
	^	-				.8 22					9 3	
						1.6 24			57 14	- (5 35
•						.6 .26				8 17	5 14	
	1	- 1.	9			1.8 26	1 -			14 22	18	
	1.		11			1.2		30 2	91 2	73 24	8 21	6 126
	-	• 🕂	12		1	0.0 27	~		A . 1	33 25	8 22	5 133
		_			ly total	5 331	2 262	26 24	20 22	04 195	0 164	4 .874
					,						,	

Date		lar ne		olar ition	ВТІ	JH/sq. ft	. wel	insolat	ion on	surface	·\$
	ΑM	PM	Ait	Azm				South 1	acing s with h		
. • •			•		Normai	Horiz.	38 _	48	. 58	68	90
Jul 21	5	7	5.7	114.7	43	10	5	. 5	4	4	3
-	6	6	15.2	104.1	156	62	28	18	16	15	11
	7	.5	25.1	93.5	211	118	89	75	59	42	114.
	8	4	35.1	82.1	240	171	154	140	121	99	43
	9 10	3 2	44.8 53.5	68.8 51.9	256 2 66	215 250	~214 261	. 199 :246	178 224	153 195	83 116
	11	1	60.1	29.0	271	272	291	276	253	-223	137
		2	62.6	0.0	272	279	301	286	263	232	144
	Su	rface	daily-t	otals	3158	2474	2386	2200	1974	1694	956
Aug 21	6	6	9.1	98,3	99	28 -	14	10	. 9	8	6
	7	5	19.1	87.2	190	85	75	67	58	47	20
	8	4	29.0	75.4	232	141	145	137	125	109	65
<u> </u>	9 10	3 2	38.4 46.4	61.8 45.1	254 266	189 . 225	210 260	201 252	187 237	168 214	110 146
	11	1	52.2	24.3	272	248	293	285	268	244	169
	\vdash	2	54.3	0.0	274	256	304	296	279	255	, 177
•	Su	rface	daily t	otals	2898	2086	2300	2200	2046	1836	à 208
Sep 21	7.	5	10.0	78.7	131	35	44	44	43	40	31
	8	4	19.5		215	92	124	124	121	115	90
	9	3	28.2	53.4	251	142	196	197	193	183	143
	10-	2	35.4 40.3	37.8 19.8	269 278	181	251 287	254 ³ 289	248 284	236 269	185 212
	<u> </u>	2	42.0	0.0	280	213	299	302	296	281	221
۰ نو	Su	rface	Baily 1	orals	2568	1522	2102	2118	2070	1966	1546,
Oct 21	7	5	2.0	71.9	4	0	1	1	1	1	1
	8	4	11.2	60.2	165	44	86	91	95		87
	9	3	19.3		233	94	167,	176	180	178	157
	10	2	25.7	33.1	262	-133	226	239	242	239	207
	11	1	30.0	17.1	274	157	266	277	281	276	237
	 	2	31.5	0.0	278	166	279	291	294	288	247
i) 		rface	dally	totals	2154	1022	1774	1860	1890	1866	1626
Nov 21	8	4	3.6	54.7	36	5	17		137	22 141	135
	10	3 2	11.2	42.7 29.5	179 233	83	117	202	212	215	201
	11	î	20.9	15.1	255	1407	227	245	255	258	
,		12	22.2	0.0	261	115	241	259	270	272	250
	Su	rface	daily	totals	1668	596	1336	1448	1518	1544	442
Dec 21	9	3	8.0			27	87		105	110	109
, 🐔	10	2	13.6			63	164	180	192	197	190
	11	11	17.3	1	1	86	207	1	239 254	244	231
	<u> </u>	12	18.6	0.0	250	94	222	241	454	260	244

1 BTUH/SQ. FT..= 3.152 W/m²
NOTE: 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1:0 clearness factor.

2) See Fig. 4, p. 394 in [3] for typical regional clearness factors.

3) Ground reflection not included on normal or horizontal surfaces.

56 Degrees North Latitude

Date	Sol		_	lar tion	ВТ	JH/sq. fi	. total	insolati	on on s	urfaces	
-	АМ	РМ	Albe	Azm		- 5	9	South fa	cing se		
.					Normal	Horiz.	46	56	66	76	90
Jan 21	9.	3	5.0	41.8	- 78	11	50	55	59	6Ó	60
,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	10	2	9.9	28.5	170	39	135	146	154	156	153
	11	1	12.9	14.5	207	58	• 183	197	206	208	201
	1	2	14.0	0,0	217	65	198	214	222	225	217
	Sui	face	daily t	otals	1126	282	934	1010	1058	1074	044
Feb 21	8	4	7.6	59.4	129	. 25	65	69	72	72	69
	9	3	14.2	45.9	214	65	. 151	159	162	161	15 1 208
•	10	2	19.4	31.5	250	98 119	215 254	225 265	228 268	224 263	243
	11	1	22.8	ł	266	1	268	279	282	276	255
	10		24.0	0.0	27.0	126		1716			1598
	Su	1	daily 1	т —	1986	740	1640	<u> </u>			
Mar 24	7.		8.3	77.5		28 75	119	40 120	39 117 ن	37 111	32 97
	8 9	3	16.2 23.3		1 .	118	, 192	193	189	180	154
	10	2	29.0	34.9		151	249	251	246	234	205
	11	lī.	32.7			172	285	288	282	268	236
•		12	34.0	0.0	284	179	297	300	294	280	246
	Su	rface	daily	totals	2586	4268	2066	2084	2040	1938	1700
Apr 21	5	7	1.4			0	0	,	0	0 7	0 6
1-	6	6	9.6		_		14	1	57	46	29
•	7 8	5	18.0 26.1		•	129	143		123	108	82
•	9	3	33.6	1	1	169	208	200	186	167	133 °
	10	2		39.7	272	201	259		236		174
,	11	1	44.1		•	220	1	1 1	268		,200
æ		12	45.6	0.0		227	+		279	+	209
	Sı	ırfacı		totals	3024	1892	+		2038	+	1458
May 21	4			125.		0				1 '	0
,	5		16.	5 113.4 5 101¶	4 93 175	25		- 1	1	1 :	11
1	6		24.		. 1	-1	1 .		58	41	16
	8	4	33.	1 76.	3 244	163					63
	9										109 146
1	10		1		1			•		1	170
	11	12	$-\begin{vmatrix} 32. \\ 54. \end{vmatrix}$	1	į.		}	1	1	- 1	178
-	+-			totals	3340		+	- 0	1962	1682	1218
Jun.2	+				- 	+	+	2 2		2 2	1
Jun.2	' <u>3</u>			1		7	1	4 13	3 11	10	8
		5 6	19.	3 103.	6 185	8					12
		7 5									
		8 4									
	1	9 3							3 214	4 184	133
	1								7 . 24	1 /	1
1 4	\vdash	12	57.	1	.0 271	26	4 29	4 27	5 25	1 219	164
ł	1		1	T.1 4	: L				_	$\overline{}$	

							•	۵				
`	Date	Şo tir	lar ne		lar ition	ВТ	JH/sq. ft	. total	insolati	lon on	surface	,
	·	AM	PM	Ait	Azm				South f angle	acing s with h		
	•			*	K	Normal	Horjz.	46	56	66	76	90
Ì	Jul 21	4	8.	1.7	125.8	0	Q	0	0	٥	0	0
ļ	ļ	5	7	9.0	113.2	91	2.7	11	. 10	9	8	6
ı	1	6	6	17.0	101.9		72	30	18	16	14	12
١	1	7	5	25.3	89.7	212*	119	88	(74	58	41	15
-	į	8	3	33.6 41.4	76.7 62.0	237 252	163 201	·151	136	117	96	61 106
ı	,	10	2	48.2	44.6	261	230	254	239	217	189	142
		11	i	52.9	23.7	265	248	283	288	245	216	165
	'	1	2	54.6	0.0	267	254	293	278	255	225	173
		Su	rface	daily t		3240	2372	2342	2152	1926	1646	1186
1	Aug 21	5.	7	2.0	109.2	1	.0	, ô	0	.0	. 0	, 0
[.	2	6	.6	10.2	97.8	112	34	16	- 31	0/ 10		7•
]		7	5	18.5	84.5	187	82	73	65	56	.45	28
		8		26.7	71.3	225	128	140	131	119	104	78
**		9	3	34.3		246	168	202 251	193 242	179 227	160 206	126 166
		11	2	40.5	40.0	258 264	199	282	274	258	235	191
,		1	<u> </u>	46.3	.0.9	266	225	293	285	269	245	200
	-			daily I		2850	1884	2218	2118	1966	1760	1392
	Sep 21	7	5	8.3	77.5	107	25	* 36	36	, 34	32	28
٠,		8	4	16.2	64.4	194	72	1/11	111	108	102	. 89
1		9	3	23.3	50.3	233	114	181	182	178	168	147
•	·	110	∤•2	29.0	34.9	253	146	236	•237	232	221	199
	,	11	12	32.7 34.0	17.9	263 265	166	271 283	273	267 279	254 265	223 233
	-	-		daily		2368	1220	1950	1962	1918	1820	1594,
	Oct 21	 	T 4	7.1	° 59.1	104	1 . 20	53	57	59	59	57
		. 9	3	13.8	•45.7	193	60	1 38	145	148	147	.138
		10	2	19.0	31.3		92	201	210	213	210	195
	•	11		22.3	16.0		112	240	250	253	248	230
		├ ─-	12	23.5			119	253	263	266	+	241
		├ —	, 	daily		1804	688	1516	1586	461,2		1480
	Nov 21			5.2			12	132	54 £143	57 149	152	58 148
		10	-1	10.0	1		58	179	193	201	203	196
		<u></u>	12	14.2	i	1	65	194	-209	217	1 '	213
	•	-		dQb	<u> </u>	1094	284	914	986	1032	1046	
	Dec 21	1	3	1.9		5	2 0	3	4	4	. 4	4
		10		6.6			19	· 86		101	104	103
		11	11	9.5			,37	141	154	163	167	164
			12	10.6	0.0	180	43	159	173	182	186	182
		Su	urfaçe	daily	totals	748	156	620	678	716	734	722

1 BTUH/SQ. ET. = 3.152 W/m²
NOTE: 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1.0 clearness factor.

2) See Fig. 4, p. 394 in [3] for typical regional clearness factors.

3) Ground reflection not included on normal or horizontal surfaces.

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64 Degrees North Latitude

Date		Sal		Se pos	ition	BTU	JH/M. f	ı, totál					
<u> </u>		AM	PM	Alt	Αzm				South angl	facing e with	sur	z.	
9	+	\dashv				Normal	Horiz.	54	64	74] [14	90
jan 2	11	10	2	2.8	28.1 14.1	'22 81	2 12	17 72				20 81	20 81
	-	<u> 11.</u>	1 2	5.2 6.0	0.0	100	16	91	ì	1	2	103	108
•	\dashv			daily		306	45	268	290	30	2	306	304
Feb 2	,,†	8	4	3.4	58.7	′ 35	4	17			94.	19	19
F C D 2	"	ور	3	8.6	44.8	147	e31	103					107
	ļ	1.0	2	12.6			55	170				178 219	21
•		11	1	15.1 16.0	1			225		•		232	226
	_		2 eface	dally	ــــــــــــــــــــــــــــــــــــــ	1432	400	1230	+		02 1	282	1252
		-	***	_			18	3(0 2	9 :	29	27	25
Mar.	21	7 8	5	6.5 20.7			54	10	1 10	2 9	99	94	89
		و. ا	3	18.1	48.1	227	87	17			69	160	153 203
		10	2	22.3			112	22 26			24 59	246	235
,		11	$\frac{1}{12}$	25.1 26.0			134	- 1	1	`	71	258	246
-	<u> </u>			٠	totals	2296	/932	185	6 187		30	1736	1656
Арг	21	5	7			5 , 27	, 5		2	2	2	1	1
יייר	•	6		10.	4 95.				5	9	5.0	7	37
1		7		17.					' 6	28 1	16	-102	91
ţ		8									76	158	145
ŀ		10		· 1				24	16 2		224	203	188
	_	11	1	-1			184	1		- 1	255	-233	3216
	•		12	97.	6 0		- 				266	249	225 1594
		S	urfac	e dall	y totals			- - ,			936	1736	3
May	/ 21	1 4	1 8		8 125			1]	₹ .	4	10	9.	8
		-1		- 1	.6 112 .9 99	1 13	5.4 7		29 '	16	74	12	
				5 17 5 24			B 11	7	86	72	5.6-	39	28
					.9 71	.5 23	9 15			33	115	94	120
			9	3 36	.8 56			1 -		90	170 213	145 186	J 28
			- 1	2 41		1.9 26				64	242	213	
		+	1 <u> </u> 12	1 44).1. 26).0 26	1	1	,,,		251	222	1
+		+,			y total					24 1	898	1624	1436
 	n 2	-+-			1.2 139	9.4 2	11	4	2	2.	2.		
'			4	8 9	0.0 12	5.4 9	- 1	27	10	9	8 13		
.			5		1.7 11			50 ·	16 34	1.9	17	1	1
1	•				1.0 10 7.5 8	0.8 19 7.5 22		32	91	74	-55		2
[•		8					56	150	133	112	88	
			9			7.8 25	5 1	95	_	187	164		
			10	2 4	4.9 4	0.4 25				230 258	20 6 233		
]		+	12			7.	v	1	_ i	267	242	1	1
	•		3 4	, j 🕶	I	I	/ -		1.				

	Solar Solar										
Date	time position				ВТ	UH/sq.	ft. tota	u insol	ation o	n surfa	ces
•	AM	PM	Alt	Azm] -	Souti	h facing	surfac horiz.	.e
_ ~					Normal	Horiz.	54	64	74	84	90
Jul 21		8	6.4	125.3	53	~ 13	. 6		5 9		4
	5	•7 6	12.1 18.4	112.4	128	44	**	13	3 11	10	
~ ·,	1 7	5	25.0	99.4 86.0	1 79 21,1	81 118	30		, ,		
}	. 84	4	31.4	71.8	231	152	146				
1	9	3	37.3	56.3	245	182	201		–		
1 \	10	2	42.2	39.2	253	204	245				
\	11	1	45.4	20.2	257	218	273	258	236		
	- 1		46.6	0.0	259	223	282	267	245	216	195
	* 	face	dally t	otals	3372	2248	2280	2090	1864	1588	1400
Aug 21	5	7	4.6	108.8	29	6	3	3	2	2	2
l .	6	6	11.0	95.5	123	39	76		1 10		
İ	7 8.	5	17.6	81.9	181	77.	69		52		35
l ⁱ	9	4 3	23.9 29.6	67.8	214	113	132	123			
ĺ	10	2	34.2	52.6 736.2	234 246	144 ² 168		182		150	
ĺ	111	- 1	37.2	18.5	252	183	237 268	229 260	215	194	179
•	12		38.3	0.0	254	.188	278	270	255	232	205 215
	Surf	ace d	ally to	otals	2808	1646	2108	1008	1860	1662	1522
Sep 21	7	5.	6.5	76.5	.77	16	25	25	24	23	1
	8	4	12.7	72.6	163	-51,	92	92	90	85	21 ' 81
	9		18.1	48.1	206	83	159	159	156	147	141
	10		22.3	32.7	229	108	212	213	209	198	189
			25.1	16.6	240	124	246	248	243	230	220
	12	_ +	26.0		244	129	258	260	254	241	230
			ally to		2074	892	1726	1736	1696	1608	1532
Oct 21	8	4 3	3.0	58.5	17	2	9	9	10	· 10	10
' 1	10		8.1 2.1	44.6 30.2	122 176	26 50	86	91	93	92	-90
Ì	11		4:6	15.2	201	65	152 193	159 201	161 203	159 200	155
	12		15.5	0.0	208	°71	207	215	217	213	195 208/
	Surfa	ice da	ily to	tals ·	1238	358	1088	1136	1152	1134	1106
Nov 21			3.0	28.1	23	3	18	20	_ 21 [°]	21	.21
·	11	<u> </u>	5.4	14.2	79	. 12	70	.76	78	80	79
	12	_4	6.2	0.0	97	17	89	. 96	100	101	100
-		* 	ily to		302	46	266	286	1298	302	300
Dec 21		1	1.8	13.7	4	0	3	- 4	4	14	4 .
	12		2.6	0.0	16	2	14	15	16	17	† 7
<u> </u>	-Surfa		#y to:		24	2	20	22	24	24	24

1 BTUH/SQ. FT. = 3.152 W/m³
NOTE: 1) Based on data in Table 1, p. 387 in ref. [3]; 0% ground reflectance; 1.0 clearness factor.

2) See Fig. 4, p. 394 in [3] for typical regional clearness factors.

3). Ground reflection not included on normal or horizontal surfaces.

APPENDIX C

Table 1 THERMAL PROPERTY DATA

Material,	Thermal Conductivity, k. (Btu-ft/hr ft ^{2 O} F)
. Air	.015
Water	.35
Brick	.2230
Concrete	5 .5075
Glass	.4250
Granite	1.08 - 2.33
Limestone	.3375
Sandstone	.87 - 1.33
Wood	.14
Gypsum Wallboard	.0925
Corkboard	.022025 -
Fiberglass batt	.025030
Glass wool blanket	(022023
Insulating boards	.027031
Polystyrene	.022
Polyurethane	.011
Rock wool, loose	.024030
A1 umi num	55' - 110
Copper	15 - 65
Nickel	8 - 12
Silver, cast	242
Steel (∿1% carbon)	21 - 25
F Zinc, cast	60 - 65

Table 2
COEFFICIENTS OF TRANSMISSION OF WINDOWS

3	Window	(U-values)
a	Single glass .25"	1.10
29	Thermopane .5" airspace	.49
	Storm windows 1 to 4" gap	.50

Table 3 R-VALUES

Material	R-Value
Plasterboard, 1" thick	0.45
Plywood, 1/4" thick	0.31
Paneling, softwood 1" thick	1.25
Paneling, hardwood 1" thick	0.95
Cork wall covering 1/8" thick	0.44
Acoustic tile	2.75
Terrazzo, 1" thick	0.08
Concrete, 1" thick	0.08
Lino]eum flooring	0.05
Vinyl flooring	0.05
Rubber floor tiles	0.05
Carpet with fibrous pad	2.00
Air, 3/4" to 4"	0.96
Insulating board, 15/32"	1.11
Styrene foam, 1"	4.5
· · · · · · · · · · · · · · · · · · ·	

Table 3 (continued)

Material	•	R-Value	,
Mineral wool, 1"		3.3	
Glass, single pane		0.88	•
Glass, two pane	- 3	J.54	1
Glass, thermopane		2.04	
Glass, three pane	•	2.12	•
Glass, storms with 1" to 4" gap	•	2.00	
For comparison 1" fiber glass	••	3.30	
/ US Department of	Energy 197	77	

Table 4
INSULATION (R-values)

Material	· Batts or	Blankets		se Fill (Po	
Thickness (Inches)	Glass Fiber	Rock Wool	Glass Fiber	Rock Wool	Çellulose Fiber
	3.38	3.66	2.20	2.75	3.66
2	,6.76	7.32	4.40	5.50	7.32
. 3	10.14	10.98	6.60	8.25	10.98
4	13.52	14.64	8.80	11.00	14.64
5	16.90	18.30	11.00	13.75	18.30
6	20.28	21.96	13.20	16.50	21.96
7	23.66	25.62	15.40	19.25	25.62
8	27.04	29.28	17.60	22.00	29.28
9 _	30.42	32.94	19.80:-	24.75	32.94
10 *	33.80	36.60	\$2.00	27.50	36.60
, 11	37.18	40:26	24.20	30.25	40.26
12	40.56	43.92 1	26.40	33.00	43.92

Table 5
RECOMMENDED R-Values

	FHA Minimum for Gas Heat	National Mineral Wool Assoc. for Oil Heating	TVA C Electric Heating	Studies based on minimum life cost	Owen Corning
Ceilings	R -19	R-19 or 22	R-19	R-30	R -38
Walls	R-11	R-11	R-11	R-20	R-19
Floors over unheated			1		
space		R-11	R-11	1 R-20	R-22

Table 6 RETROFITTING

Typical Old House.	Typical Old House			Retrofitted		
Roof	;	. , 3		12" fiberglass	R=43	
Wall		4	***	3½ blown	R=18	
Basement		10		3½ fiberglass or 2" styrofoam	R=20	
. Ground Floor		5		6" fiberglass	R=25	
Door	•	?		Add storm insu- lated door	R-3 R-10	
Window	•	1 .		Add storm window shutters	- R-10	
	······································	Maine Audubon	Society	<i>'</i>		

Heat Loss for Typical Walls

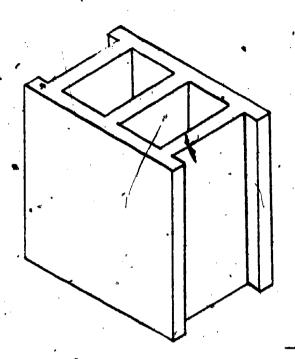


Figure 1

CONSTRUCTION	RESISTANCE
1. Outside surface (15 m.p.h. wimd)	0.17
2. Concrete block, sand and gravel aggregate, 8 in.	- 1.04
3. Inside surface (still air)	0.68
ا من المنظم ا المنظم المنظم المنظ	. 1.89
	•
q = 37.04 Btu/hr ft ²	

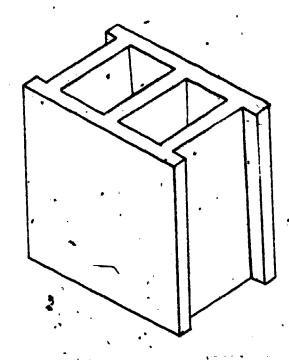


Figure 2

CONSTRUCTION

- Outside surface (15 m.p.h. wind)
- 2. Concrete block, lightweight aggregate, 8 in.
- Inside surface (still air)

q = 23.18 Btu/hr ft²

RESISTANCE

- 0.17
- 2.17
- 0.68 3.02

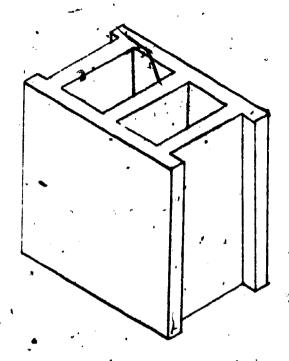


Figure 3

CON	ISTRUCTION	£ .	RESIS	STANC	<u>E</u>
Į.	Outside surface (15 m.p.h. wind)		0	. 17	
2.	Concrete block, sand and gravel aggregate, i with mineral wool, 8 in.	nsulated	1	.92	
3.	Inside surface (still-air)		2	.68 .77	
-	q = 25.27 Btu/hr ft ²	•			,

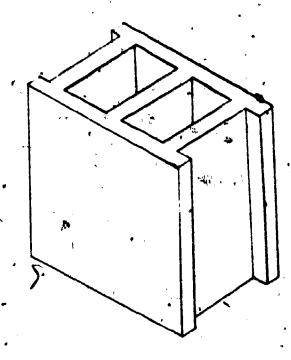


Figure 4

	- .
S CONSTRUCTION .	RESISTANCE
1. Outside surface (15 m.p.h. wind)	0.17
2. Concrete block, lightweight aggregate, insulated with mineral wool, 8 in.	5.00
3. Inside surface (still air)	0.68 5.85
q=`11.97 Btu/hr ft ²	, •

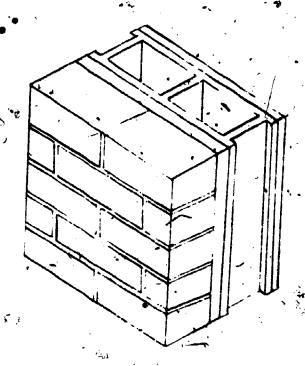


Figure 5

CO	NSTRUCTION	•		RESISTANC
í.	Face brick, 3.625 in.	•		. 50
2.	Rigid polystrene extruded, 1 in.	•		. 36
3.	Concrete block, lightweight aggregate, 4 in.			1.49
4 .	Plaster, .375 in.			.03
5.	Outside surface, (15 m.p.h. wind)	•	•	0.17
6.	Inside surface (still air)	,		0.68 3.23

q = 21.64 Btu/hr ft²

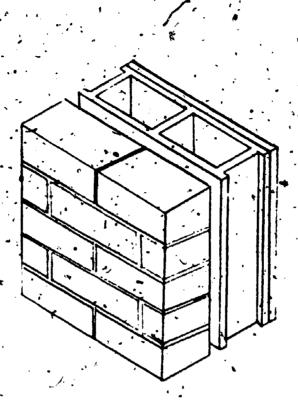


Figure 6

CONSTRUCTION		RESISTANC
1. Outside surface (15 m.p.h. wind)		0.17
2. Face brick, 3.625 ip.	•	0.50
3. Non-reflective air space, 1.00 in.		1.12.
4. Concrete block, lightweight aggregate, 4.00 in.		1.49
5. Plaster, sand aggregate, .375 in.	*	0.08
6. Inside surface (still air)		0.68
q = 17.35 Btu/hr ft ²	٠,	

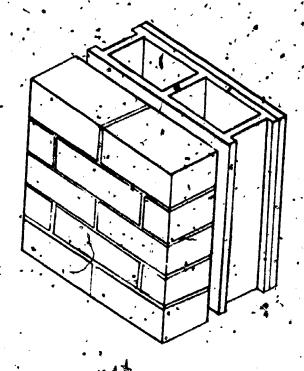


Figure 7

CONSTRUCTION	RESISTANCE
1. Face brick, 3.625 in.	0.50
2. Outside surface (15 m, p.h. wind)	0.17
3. Air space, 1 in.	-1.12
4. Concrete block, lightweight aggregate, insulated with mineral wool, 4 in.	,2.50
5. Plaster, .375 in.	0.03
6. Inside surface (still air)	<u>0.68</u> 5.00

q= 13.99 Btu/hr ft²

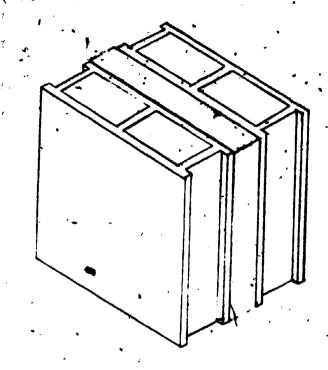


Figure 8

CON	STRUCTION	*	RESISTANCE
1.	Outside surface (15 m.p.h. wind)	٠	0.17
2.	Concrete block, lightweight aggregate, 4 in.		1.49
3.	Polyurethane foam, & in.	•	15.15
4.	Concrete block, lightweight aggregate, 4 in.	•	1.49
5.	Inside surface (still air)		0.68 18.98

q = 3.69 Btu/hr ft²

/201

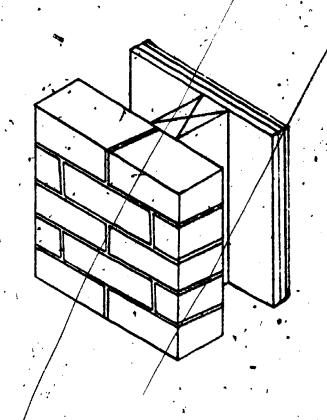


Figure 9

CON	STRUCTION	RESISTANCE
1.	Outside surface (15 m.p.h. wind)	0.17
2.	Face brick, 3.625 in.	0.50
3.	2" x 4" studs @ 16" o.c.	4.38
4.	Air space, 3.50 in.	1.14
5.	Impregnated sheathing, .75 in.	2.04
Ġ.	Plywood, .75/in.	.93
7.	Gypsum board, 50 in.	.45
8.	Inside surface (still air)	0.68
		10.29
		_

10

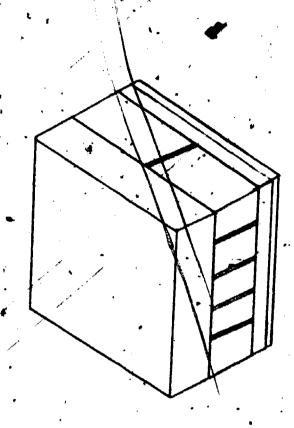


Figure 10

CONSTRUCTION

- 1. Marble, 3.00 in.
- 2. Face brick, 3.625 in.
- 3. Pine, 1.00 in.
- 4. Plaster, .75 in.
- 5. Outside surface, (15 m.p.h. wind)
- 6. Inside surface (still air).

q = 15.10 Btu/hr ft²

RESISTANCE

- .33
 - .50
- 1.39
- 1.56
- 0.17
- $\frac{0.68}{4.63}$

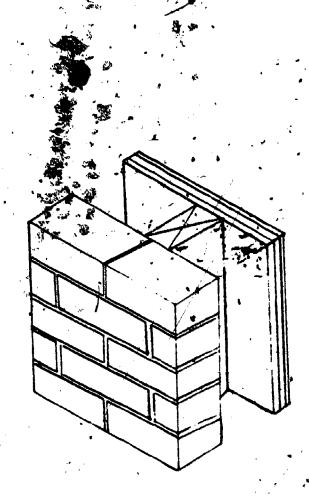


Figure 11

CON	STRUCTION		*	RESISTANCE
1.	Outside surface (15 m.p.h. win	ıd)	•	0.17
2.	Face brick, 3.625 in.	· ' &		0.50
3.	2" x 4" studs 0 16" o.c.			4.38
4.	3.5 in. polyurethane insulation	n´		26.52
5.	Impregnated sheathing, .75 in.		4.	2.04
6.	Plywood, .75 in.			.93 ~1
7.	Gypsum board, .50 in.	1		. 45
8.	Inside surface (still air)			0.68
		•	•	35.67

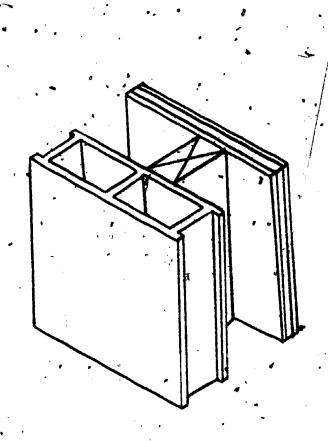


Figure 12

CON	STRUCTION	RES1STANCE
1.	Concrete block, lightweight aggregate, 4.00 in.	1.47
2.	Outside syrface, 15 m.p.h. wind	0.17
₩.	2" x 4" studs @ 16 in. o.c.	, 4.38
4.	Impregnated wood sheathing, .75 in.	2.04
_5.	Plywood, .75 in.	.93
6.	Gypsum board, .50 in:	.45
7.	Inside surface (still air)	0.68
	q = 6.92 Btu/hr ft ²	•

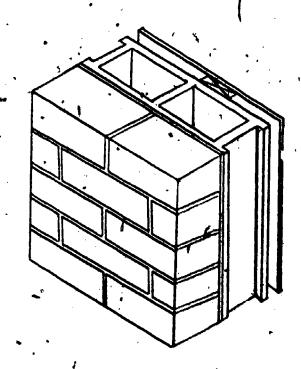


Figure .13

CONSTRUCTION	RESISTANCE
1. Outside surface, 15 m.p.h. wind	0.17
2. Face brick, 3.625 in.	0.50
3. Cement mortar, 0.5 in.	0.10
4. Concrete block, cinder aggrégate, 4.00 in.	1.47
5. Reflective air space, 0.75 in.	2.77
6. Nominal 1" x 3" vertical furring	0.94 ,
7. Gypsum wallboard, .5 in.	0.45
8. Inside surface (still air)	0.68
	7.08

q = 9.88 Btu/hr ft²

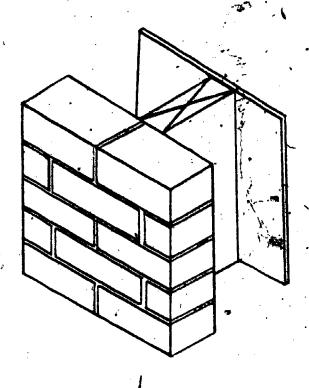


Figure 14

CON	ISTRUCTION	RESISTANCE
1.	Outside strfaçe, 15 m.p.h. wind	0:17
	Face Brick, 3.625 in.	0.50
3.	2" x 6" studs	6.88
4.	Air space, 5.50 in.	1.16
5.	Gypsum board, 0.5 in.	0.45
6.	Inside surface /(still air)	<u>0.68</u> 9.84

q = 7.11 Btu/hr ft²

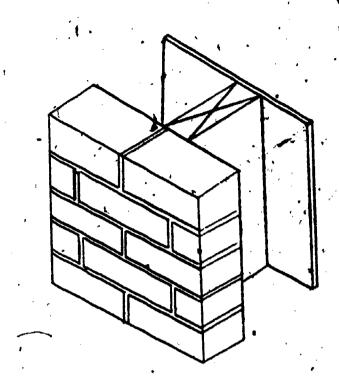


Figure 15

CON	STRUCTION	RESISTANCE
1.	Outside surface, 15 m.p.h. wind	0.17
2.	Hace brick, 3.625 in.	0.50
ą .	2" x 6" studs ,	6.88
4.	Polyurethane insulation, 5:50 in.	41.67
5.	Gypsum board, 0.5 in.	0.45
6.	Inside surface (still air)	0,68
	At framing: R = 8.683 Between framing: R =	43.47
	Average R factor at the stude is = 37.67	
د	$q = 1.77 Btu/hr ft^2$	

214

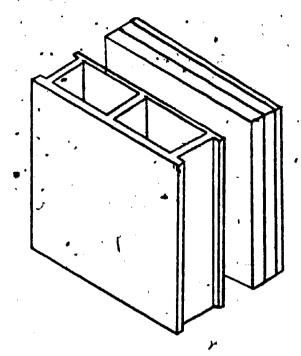


Figure 16

CON	ISTRUCTION	RESISTANCE .
ી.	Outside surface, 15 m.p.h. wind	0.17
2.	Concrete block, lightweight aggregate, insulated with mineral wool, 4 in.	2.50
3.	Air space, 2 in.	1.12
, 4 .	Polyurethane, lin.	7.58
5.	Wood siding, 1 in.	4 0.67
6.	Gypsum board, .50 in.	0.45
7.	Inside surface (still air)	0.68
	$q = 5.439 \text{ Btu/hr ft}^2$	

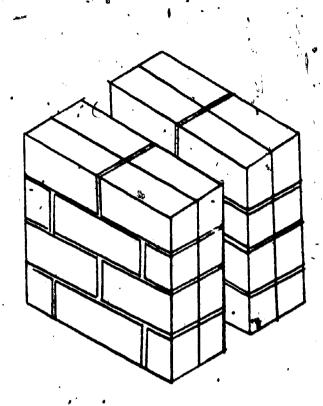


Figure 17

τ_0	NS	TRI.	רח	T	Ω I	V
- -	11.	יתי			v	1

- 1. 2 wythes of bricks, 4.50 in.
- 2. Air space, 2°in.
- 3. 2 wythes of bricks, 4.50 in.
- 4. Outside surface, 15 m.p.h. wind
- Inside surface (still air)

q = 21.54 Btu/hr ft²

RESISTANCE

- 0.62
- 1.15
 - 0.62
 - 0.17
 - 0.68 3.24

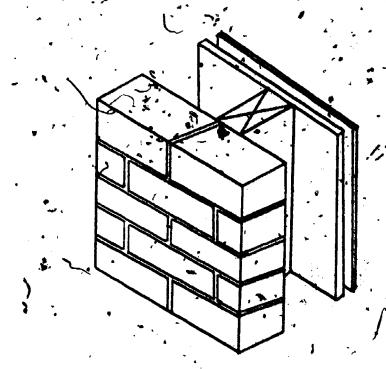


Figure 18.

CONSTRUCTION

- 1. Outside surface, 15 m.p.h. wind
- 2. Face brick, 3.625 in.
- 3. Wood studs, 2" x 4"
- ·4. Air, 3.50 in.
- 5. Plywood, .75 in.
- 6. Reflective air space, :75 in.
- 7. Gypsum board, 5 in.
- 8. Inside surface (still air)

RESISTANCE

- 0.17
- 0.50
- 4.38
- 1.16
- 0.93
- 2.77,
- 0.45
- 0.68
- 11.21

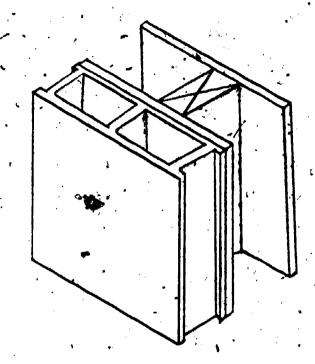


Figure 19

CONS	STRUCTION	· · · · ·	RESISTANCE
1.	Outside surface, 15 m.p.h. wind	ı t	0.17
, 2.	Concrete block, lightweight aggregate, with mineral wool, 4 in.	insulated	2.50
3.	Plywood, .75 in.	•	0.93
4.	Wood studs, 2" x 4"	****	4.38
5.	Polyurethane, 3.50 in.	10	26.52
6.	Plywood, .75 in.		0.93
7 `	Inside surface (still air)		0.68
- ,)	36.11

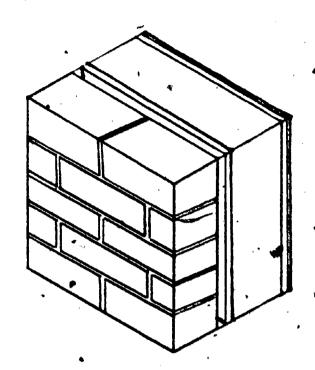


Figure 20

COV	CONSTRUCTION		RESISTANCE
1.	Outside surface, 15 m.p.h. wind	•	0.17
2.	Face brick, 3.625 in.		0.50
3.	Reflective air space, .75 in.	• "	2.77 [.]
4.	Polyurethane, 4.50 in.		- 34.09
5.	Plywood, .75 in.	•	0.93
6.	Plywood, .75 in.	•	0.93
7.	Gypsum wallboard, .375 in.		0.32
8.	∄nside surface (still air)	•	0.68
		•	40.39

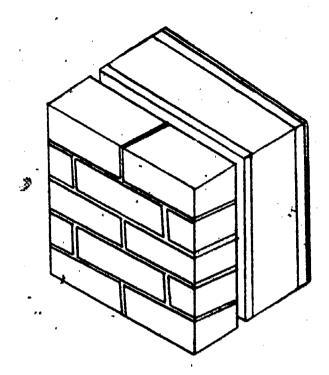


Figure 21

CON	ISTRUCTION	RESISTANC
۱.	Outside surface, 15 m.p.h. wind	0.17
2.	Face brick, 3.625 in.	0.50
3.	Reflective air space, .75 in.	2.77
4.	Polystyrene, 4.50 in.	17.04
5.	Plywood, .75 in:	0.93
6.	.Plywood, .75 in.	0.93
7.	Gypsum, board, .375 in.	0,32
8.	Inside surface (still air)	0.68
		23.34
	q = 2.99 Btu/hr ft ²	

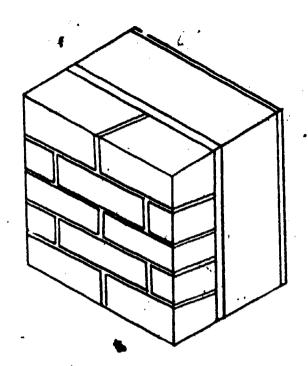


Figure 22

CONSTRUCTION		RESISTANCE	
1.	Outside surface, 15 m.p.h. wind		0.17
2.	Face brick, 3.625 in.		0.50
3.	Wood sheathing, .75 in.,	,	2.04
4.	Polystyrene, 4.50 in.	• '	17.04
5.	Gypsum wallboard, .50 in.		0.45
6.	Inside surface (still air)	•	<u>0.68</u> 20.88



q = 3.35 Btu/hr ft²

APPENDIX D

ESTIMATED ANNUAL ENERGY CONSUMPTION OF APPLIANCES (KILOWATT-HOURS)

Major

Air Conditioner (room) - 860

Air Conditioner (central) - 1240

Clothes Dryer - 993

Dishwasher - 363

Freezer (16 cu ft.) - 1190

Freezer (frostless, 16.5 cu ft.) - 1820

Range - 700

Range (self-cleaning) - 730

Refrigerator (12 cu. ft.) - 728

Refrigerator/Freezer (12.5 cu. ft.) - 1217

Refrigerator/Freezer (12.5 cu. ft.) - 1500

Refrigerator/Freezer (frostless, 17.5 cu. ft.) - 2250

Washing Machine (automatic) - 103

Washing Machine (wringer type) - 76

Water Heater - 4811

Kitchen

Blender - 15
Broiler - 100
Carving Knife - 8
Coffee Maker - 140
Deep Fryer - 83
Egg Cooker - 14
Frying Pan - 186
Hot Plate - 90
Mixer - 13
Microwave Oven - 190
Roaster - 205
Sandwich Grill - 33
Toaster - 39

Kitchen (continued)

Waffle Iron - 22 Waste Disposer - 30

Heating & Cooling

Air Cleaner - 216
Electric Blanket - 147
Dehumidifier - 377
Fan (attic) - 291
Fan (furnace) - 43
Fan (rollaway) - 138
Fan (window) - 170
Heater (portable) - 176

Heating & Cooling (continued)

Heating Pad - 10

Humidifier - 163

Iron (hand) - 144

Health & Beauty

Germicidal Lamp - 141

Hair Dryer - 14

Heat Lamp (infrared) - 13

Shaver - 1.8

Sun Lamp - 16

Toothbrush - 0.5

Vibrator - 2

Entertainment

Radio - 86

Radio/Record Player - 109

TV:

Black & White:

Tube type - 350

Solid state - 120

Color:

Tube type - 660

Solid state - 440'

Housewares

, Clock - 17 •

Floor Polisher - 15-

Serving Machine - 11

Vacuum Cleaner - 46

Reference: "Tips for Energy Savers", Federal Energy Administration,

August 1977

APPENDIX E

GLOSSARY

- ABSORPTIVITY. The ratio of the incident radiant energy absorbed by a surface to the total radiant energy falling on the surface.
- ACCELERATION. The time rate of change of velocity in either speed or direction.
- ALBEDO. The ratio of the light reflected by a surface to the light falling on it.
- ALTERNATING CURRENT. An electric current whose direction of flow is changed at periodic intervals (many times per second).
- AMPERE. A unit of measure for an electric current; rate of flow of electric current.
- ATOMIC ENERGY. Energy derived from the mass converted into energy in nuclear transformations.
- BARREL. Although seldom put in actual "barrels," crude oil is measured in a unit called the barrel, equal to 42 U.S. gallons. One barrel of crude oil has the same energy as 350 pounds of coal.
- BASE LOAD. The minimum amount of power demanded of a utility (electric or gas) over a given period of time.
- BLACKOUT. A situation in which all power is cut off from electrical generating facilities; or can be caused by storm damage, equipment failure, or overloaded utility equipment.
- BREEDER REACTOR. A nuclear chain reactor in which a greater number of fissionable atoms are produced than the number of parent atoms consumed.
- BRITISH TERMAL UNIT (BTU). The quantity of heat required to raise the temperature of one pound of water one degree Fahrenheit at or near its point of maximum density (39.10F).
- BROWNOUT. The deliberate lowering of voltage (and thus the power supplied to all users) by electric utility companies; employed when demand for power exceeds generating capacity.
- CALORIE. The amount of heat necessary to raise one kg of water by one degree Celsius. Uncapitalized, the calorie is the amount of heat required to raise one gram of H₂O one OC.
- CAPACITY. The maximum load for which a generator, transmission circuit, power plant, or system is rated.
- CELSIUS SCALE: The temperature scale using the ice point as 0 and the steam point as 100, with 100 equal divisions or degrees between.



- CHEMICAL ENERGY. The kind of energy that may be released when a chemical reaction takes place.
- CIRCUIT. The complete path traversed by an electric current.
- COAL. Solid, combustible, organic hydrocarbon formed by the decomposition of vegetable material under pressure and heat below the earth's surface.
- COAL GASIFICATION. The conversion of coal to a gas suitable for use as a fuel.
- COLLECTOR EFFICIENCY. The ratio of the energy collected by a solar collector to the radiant energy incident on the collector.
- COMBUSTION. Burning; technically, a rapid oxidation accompanied by the release of energy in the form of heat and light.
- CONDUCTION (ELECTRICAL). The process of carrying an electrical current.
- CONSERVATION OF MATTER AND ENERGY (LAW OF). Matter and energy are interchangeable; the total amount of energy and matter in the universe remains constant.
- CONSERVATION OF MECHANICAL ENERGY (LAW OF). The sum of the potential and kinetic energy of an ideal energy system remains constant.
- CONVECTION. The transfer of energy by moving masses of matter.
- CONVENTIONAL HYDROELECTRIC PLANT. A hydroelectric power plant that utilizes streamflow only once as the water passes downstream, as opposed to a pumped-storage plant which recirculates all or a portion of the streamflow.
- COULOMB. The amount of charge delivered by an electrical current of one ampere flowing for one second.
- CRUDE OIL. Liquid fuel formed from the fossils of animals and plants at the bottom of ancient seas; petroleum as it comes from the ground.
- CURRENT (ELECTRIC). The rate of movement of electricity.
- CURTAILMENT. Cutting back the use of energy resources in an emergency as opposed to conserving or wisely using energy resources.
- DENSITY. The mass per unit volume.
- DIRECT CURRENT. An electric current that flows in only one direction through a circuit.
- DIRECT ENERGY CONVERSION. The process of changing any other form of energy into electricity without machinery; for example, a battery which changes chemical energy into electricity.

- EER. Energy efficiency ratio; Btu's divided by wattage.
- ELECTRICAL ENERGY. The energy associated with electric charges and their movement; measured in kilowatt-hours.
- ENERGY. The capability of doing work. Potential energy is energy due to position of one body with respect to another or relative parts of the same body. Kinetic energy is due to motion.
- ENTROPY. A measure of the unavailability of energy to do useful work; every spontaneous process in nature is characterized by an increase in the total entropy of the bodies concerned in the process.
- ENVIRONMENT. The sum of all external conditions and influences affecting the life, development, and ultimately the survival of an organism.
- EUTECTIC SALT. A material used to store heat by melting. Heat is later released for use as the material solidifies.
- EVAPORATION. The change from liquid to gas in which molecules escape from the surface of the liquid.
- FEEDSTOCK: Energy resources used as raw materials in the production of products rather than as fuels for burning.
- FIRST LAW OF THERMODYNAMICS (also called the LAW OF CONSERVATION OF ENERGY).

 Energy can neither be created nor destroyed.
- FISSION. A nuclear reaction in which the atom is split into two approximately equal masses. There is also the emission of extremely great quantities of energy since the sum of the masses of the two new atoms is less than the mass of the parent atom.
- FLYWHEEL. Energy storage based on the principle that a spinning wheel stores mechanical energy.
- FORCE. That property which changes the state of rest or motion in matter measured by the rate of change in momentum; the force (F) required to produce an acceleration (a) in a mass (m) is given by F = ma.
- FOSSIL FUELS. Coal, oil, natural gas, shale, peat; originating from geologic deposits of ancient plant and animal life under millions of years of heat and pressure.
- FRICTIONAL FORCE. Force required to move one surface across another.
- FUEL. A substance used to produce heat energy.
- FUEL ÆELL. A device in which fuel and oxygen are combined to produce electricity.

- FUSION. A nuclear reaction involving the combination of smaller atomic nuclei or particles into larger ones with the release of energy from mass transformation; also called a thermonuclear reaction because of the extremely high temperature required to sustain it.
- GAS. A state of matter in which the molecules are practically unrestricted by cohesive forces; a gas has neither definite shape nor volume.
- GAOLINE. A mixture of hydrocarbons obtained from petroleum.
- GENERATOR. A device that converts heat or mechanical energy into electrical energy.
- GEOTHERMAL ENERGY. The heat energy available in the earth's subsurface, believed to have been produced by natural radioactivity; the temperature increases by about 1°F for each 100 feet of depth.
- GRAM. A unit of mass in the cgs system.
- GROSS NATIONAL PRODUCT (GNP): A measure of economic activity; the total market value of all goods and services produced in a country (depreciation and other allowances for capital consumption are not deducted).
- HEAT. Energy possessed by a substance in the form of internal kinetic energy; transferred by conduction, convection, or radiation.
- HEAT CAPACITY. That quantity of heat required to increase the temperature of a system or substance one degree Celsius.
- HEAT ENERGY. Energy that causes an increase in the temperature of an object; it may change the object from solid to liquid or from liquid to gas.
- HEAT OF FUSION. .The heat absorbed by a body when undergoing a phase change from a solid to a liquid with no change in temperature.
- HEAT OF SOLIDIFICATION. Heat released by a substance when undergoing a phase change from a liquid to a solid with no change in temperature (numerically equal to the heat of fusion).
- HIGH-SULPHUR CONTENT. Generally, coal or oil that contains more than one percent of sulphur by weight.
- HORSEPOWER. Unit of power in the English system.
- HYDROELECTRIC PLANT. An electric power plant in which the turbine-generators are driven by falling water.
- INCIDENT ANGLE. The angle between the direction of the sun and the perpendicular (normal) to the surface on which sunlight is falling.
- INSOLATION. Sunlight or solar radiation, including wavelengths of ultraviolet (<0.4 microns), visible (0.4 microns to 0.7 microns), and infrared radiation (> 0.7 microns). Total insolation includes both direct and diffuse insolation.



- INSULATION: A substance that can slow down the flow of heat or sound from one material to another.
- JOULE. 'A metric unit measure of work; the energy produced by a force of one newton operating through a distance of one meter (joule = newton-meter).

KILOCALORIE. 1000 calories.

KILOWATT (kw). 1000 watts

- KILOWATT-HOUR (kwh). The amount of work or energy delivered during the steady consumption of one kilowatt of power for a period of one hour.
- LATENT HEAT. A physical property of material indicating the amount of heat in calories or BTU's absorbed when a material changes from a solid to a liquid (fusion) or from a liquid to a gas (vaporization). Energy is given up when a gas condenses or when a liquid solidifies.
- LIGHT. Visible radiant energy.
- LIQUEFIED NATURAL GAS (LNG). Natural gas that has been changed into a liquid by cooling to about $-160^{\circ}\mathrm{C}$ at which point it occupies about 1/600 of its gaseous volume at normal atmospheric pressure, thus reducing the cost of shipping and storage.
- LIQUID. A state of matter in which the molecules are relatively free to change their positions with respect to each other but restricted by cohesive forces so as to maintain a relatively fixed volume.
- LOW-SULPHUR CONTENT. Generally, used to describe coal or oil which is one percent or less sulphur by weight.
- MASS. A measure of the weight of matter in an object. The weight of an object depends on its mass. The United States standard measure is the avoirdupois pound as defined by 1/2.20462 kilograms.
- MEGAWATT (MW). A unit of power equal to 1,000 kilowatts or one million watts; usually used to describe the capacity of power plants.
- METHANE. Colorless, nonpoisonous, and flammable gaseous hydrocarbon (CH₄); emitted by marshes and by dumps undergoing decomposition; the principal constituent of natural gas.
- MOLECULE. The smallest unit or quantity of matter which can exist by itself, yet retain all the properties of the original substance.
- MOMENTUM. Quantity of motion measured by the product of mass and velocity.
- MOTION. Continuous change of location or position of a body.



- NATURAL GAS. Mixtures of hydrocarbon gases and vapors occurring naturally in certain geologic formations, usually associated with oil.
- NEUTRON. A constituent particle of all nuclei of mass number greater than 1.
- NEWTON. A unit of force necessary to accelerate one kilogram one m/sec².
- NONRENEWABLE RESOURCES. Near-term depletable energy resources, such as the fossil fuels (coal, gas, and oil).
- NUCLEAR POWER PLANT. One in which heat for creating steam is provided by fission rather than combustion of fossil-fuel.
- NUCLEAR POWER. Electric power produced from a power plant by converting the energy obtained from nuclear reaction.
- NUCLEAR REACTION. A change in an atomic nucleus, such as fission, fusion, neutron capture, or radioactive decay, as distinct from a chemical reaction, which is limited to changes in electron structure surrounding the nucleus.
- OFF-PEAK POWER. Energy supplied during periods of relatively low system demands.
- OMM. The basic unit of electrical resistance of a conductor.
- OIL SHALE. A sedimentary rock containing solid organic matter (kerogen) that yields amounts of oil.
- OPEC. The Organization of Petroleum Exporting Countries.
- PEAKING DEMAND. The maximum peak load that can be supplied by a generating unit, station, or system in a stated period of time.
- PHASE CHANGE. The process involved when a material changes from a solid to a liquid or a liquid to a gas, each requiring an absorption of energy with no temperature change; or when the material changes from a gas to a liquid to a solid, each requiring a loss of energy with no temperature change.
- PHOTOVOLTAIC CELLS. (Solar cells): Semiconducting devices that convert sunlight directly into electric power.
- PLUTONIUM. A fissionable element, artifically produced by neutron bombardment of ${\rm U}^{2\,3\,8}$.
- POWER. The time-rate at which work is done.
- PRESSURE. Force per unit area.
- PUMPED STORAGE PLANT. A hydroelectric power plant which generates electric energy for peak load use by utilizing water pumped into an elevated storage reservoir during off-peak periods.

- PYRANOMETER. An instrument for measuring sunlight intensity. It usually measures total (direct plus diffuse) insolation over a broad wave-length range.
- PYRHELIOMETER. An instrument that measures the intensity of the direct beam radiation (direct insolation) from the sun. The diffuse component is not measured.
- RADIATION. The emission and propagation of energy through space or through material medium in the form of waves.
- RENEWABLE RESOURCES: Nondepletable resources.
- RESERVES. The amount of a natural resource which can be recovered by present-day techniques and under present economic conditions.
- RESERVOIR. A pond, lake, tank or basin, natural or man-made, used for the storage, regulation, and control of water.
- RESOURCES. The estimated total quantity of natural resources, including prospective undiscovered mineral reserves.
- SECOND LAW OF THERMODYNAMICS. One of the two laws which govern the conversion of energy; sometimes called the "heat tax," it can be stated in several equivalent forms, all of which describe the inevitable passage of energy from a useful to a less useful form in any energy conversion.
- SOLAR CELL. An electric cell_which converts radiant energy from the sun into electrical energy.
- SOLAR ENERGY. Radiant energy from the sun.
- SOLID. A state of matter in which the relative motion of the molecules is restricted and they tend to retain a definite fixed position relative to each other with a definite shape and volume.
 - SOLID WASTE. Unwanted or discarded material with insufficient liquid content to be free flowing.
 - SOUND ENERGY. A kind of energy carried by molecules that vibrate so that longitudinal waves are formed.
 - SPECIFIC HEAT. A physical property of materials that indicates the amount of heat required to raise the temperature of one pound of material one degree Fahrenheit, measured in BTU/1b OF; or one gram of material one degree Celsius measured in calorie/gOC.
 - FSTATIC ELECTRICITY: Electricity at rest.

- STEAM-ELECTRIC PLANT. A plant in which the turbines connected to the generators are driven by steam.
- STOCKPILE. A storage pile or reserve supply of an essential raw material
- STORAGE CELL. An electrochemical cell in which the reacting materials are renewed by the use of a reverse current from an external source.
- STRIP-MINING. A process in which rock and topsoil strata covering ore or fuel deposits are scraped away by mechanical shovels.
- TEMPERATURE. Average translational kinetic energy of the molecules of a substance due to heat agitation.
- THERMAL POLLUTION. Degradation of water quality by the introduction of a heated effluent; primarily a result of the discharge of cooling waters from industrial processes, particularly from electrical power generation.
- THERMODYNAMICS. The science and study of the relationship between heat and other forms of energy.
- TRANSFORMER. A machine which can increase or decrease the voltage of an alternating current of electricity.
- TRANSMISSION LINES. Wires or cables by which high voltage electric power is moved from point to point.
- VOLT. Unit of potential difference required to make a current of one amp flow through a resistance of ohm; V = joules/coulombs.
- WATT. A unit of power equal to the transfer of one joule of energy per second; /watt = /volt x /ampere.
- WEIGHT. The measure of gravitational force acting on a mass; W = Fgrav = Mg.
- WORK. A force acting through a distance.

